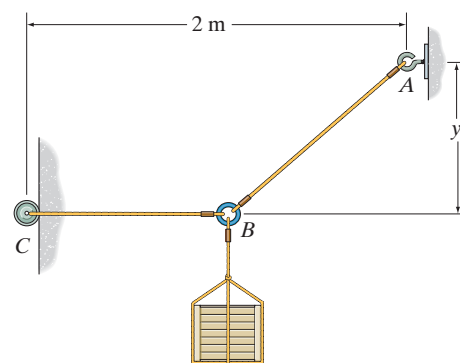


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**•3–1.** Determine the force in each cord for equilibrium of the 200-kg crate. Cord  $BC$  remains horizontal due to the roller at  $C$ , and  $AB$  has a length of 1.5 m. Set  $y = 0.75$  m.



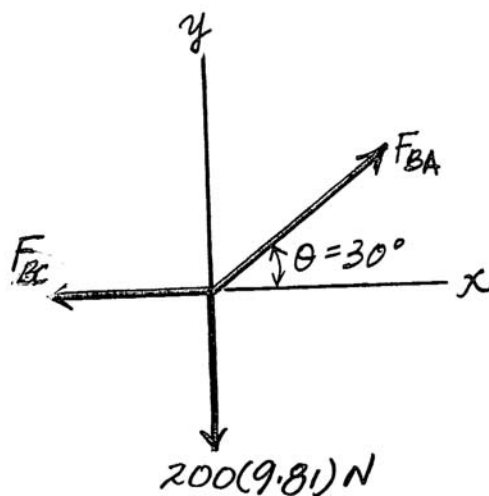
**Geometry:** From the geometry of the figure,

$$\theta = \sin^{-1}\left(\frac{0.75}{1.5}\right) = 30^\circ$$

**Equations of Equilibrium:** Applying the equations of equilibrium to the free - body diagram in Fig. (a),

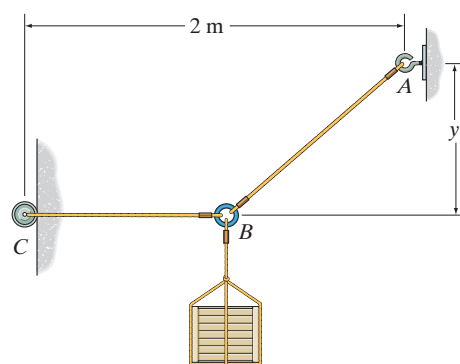
$$+\uparrow \Sigma F_y = 0; \quad F_{BA} \sin 30^\circ - 200(9.81) = 0 \quad F_{BA} = 3924 \text{ N} = 3.92 \text{ kN} \quad \text{Ans.}$$

$$+\rightarrow \Sigma F_x = 0; \quad 3924 \cos 30^\circ - F_{BC} = 0 \quad F_{BC} = 3398.28 \text{ N} = 3.40 \text{ kN} \quad \text{Ans.}$$



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**3–2.** If the 1.5-m-long cord  $AB$  can withstand a maximum force of 3500 N, determine the force in cord  $BC$  and the distance  $y$  so that the 200-kg crate can be supported.



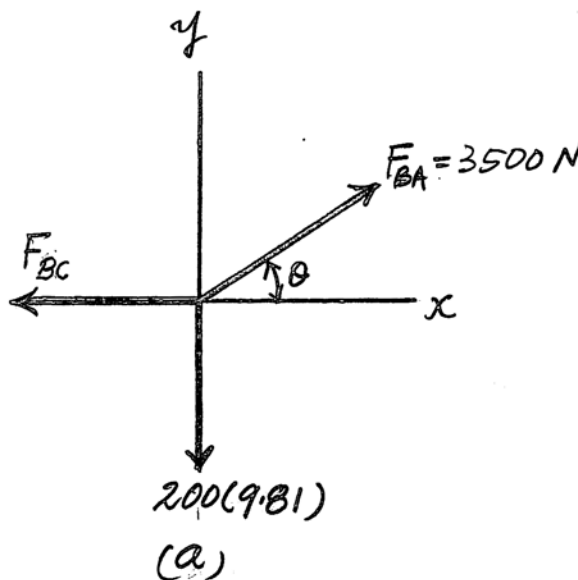
**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram in Fig. (a),

$$+\uparrow \Sigma F_y = 0; \quad 3500 \sin \theta - 200(9.81) = 0 \quad \theta = 34.10^\circ$$

$$+\rightarrow \Sigma F_x = 0; \quad 3500 \cos 34.10^\circ - F_{BC} = 0 \quad F_{BC} = 2898.37 \text{ N} = 2.90 \text{ kN}$$

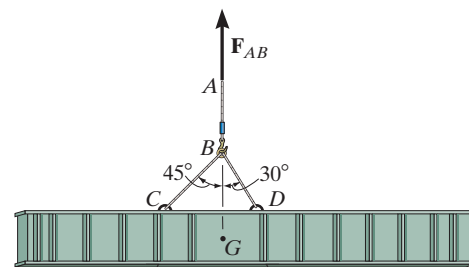
**Ans.**

$$y = 1.5 \sin 34.10^\circ = 0.841 \text{ m} = 841 \text{ mm} \quad \text{Ans.}$$



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**3-3.** If the mass of the girder is 3 Mg and its center of mass is located at point  $G$ , determine the tension developed in cables  $AB$ ,  $BC$ , and  $BD$  for equilibrium.



**Equations of Equilibrium:** The girder is suspended from cable  $AB$ . In order to meet the conditions of equilibrium the tensile force developed in cable  $AB$  must be equal to the weight of the girder. Thus,

$$F_{AB} = 3000(9.81) = 29\,430\text{ N} = 29.43\text{ kN} = 29.4\text{ kN} \quad \text{Ans.}$$

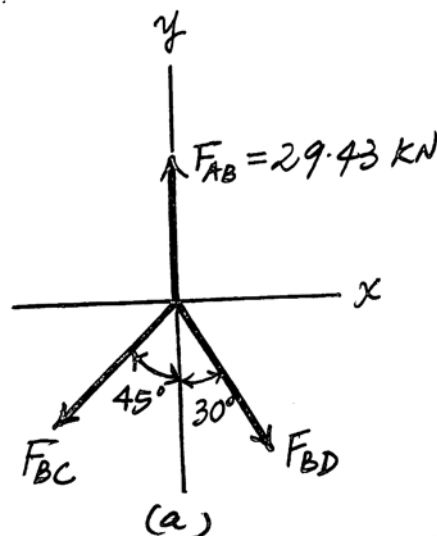
Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram in Fig. (a),

$$+\rightarrow \Sigma F_x = 0; \quad F_{BD} \sin 30^\circ - F_{BC} \sin 45^\circ = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad 29.43 - F_{BD} \cos 30^\circ - F_{BC} \cos 45^\circ = 0 \quad (2)$$

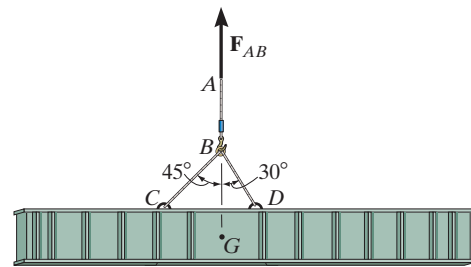
Solving Eqs. (1) and (2), yields

$$F_{BC} = 15.2\text{ kN} \quad F_{BD} = 21.5\text{ kN} \quad \text{Ans.}$$



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\*3–4. If cables  $BD$  and  $BC$  can withstand a maximum tensile force of 20 kN, determine the maximum mass of the girder that can be suspended from cable  $AB$  so that neither cable will fail. The center of mass of the girder is located at point  $G$ .



**Equations of Equilibrium:** The girder is suspended from cable  $AB$ . In order to meet the conditions of equilibrium the tensile force developed in cable  $AB$  must be equal to the weight of the girder. Thus,

$$F_{AB} = m(9.81) = 9.81m$$

**Ans.**

Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram in Fig. (a),

$$\begin{aligned} \rightarrow \Sigma F_x = 0; \quad F_{BD} \sin 30^\circ - F_{BC} \sin 45^\circ &= 0 \\ F_{BD} &= 1.4142 F_{BC} \end{aligned} \quad (1)$$

$$+ \uparrow \Sigma F_y = 0; \quad 9.81m - F_{BD} \cos 30^\circ - F_{BC} \cos 45^\circ = 0 \quad (2)$$

Since  $F_{BD} > F_{BC}$ , cable  $BD$  will break before cable  $BC$ . Substituting  $F_{BD} = 20\,000$  N into Eq. (1),

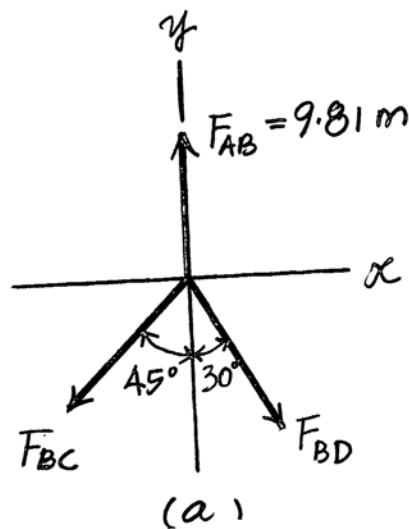
$$F_{BC} = 14\,142.14 \text{ N}$$

Substituting this result into Eq. (2), yields

$$9.81m - 20\,000 \cos 30^\circ - 14\,142.14 \cos 45^\circ = 0$$

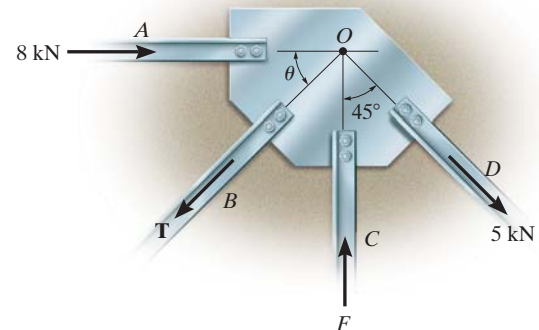
$$m = 2\,785 \text{ kg} = 2.78 \text{ Mg}$$

**Ans.**



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•3–5. The members of a truss are connected to the gusset plate. If the forces are concurrent at point  $O$ , determine the magnitudes of  $F$  and  $T$  for equilibrium. Take  $\theta = 30^\circ$ .

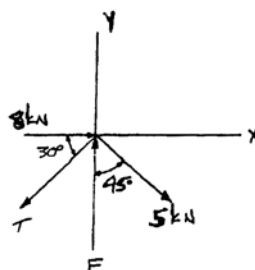


$$\rightarrow \Sigma F_x = 0; \quad -T \cos 30^\circ + 8 + 5 \sin 45^\circ = 0$$

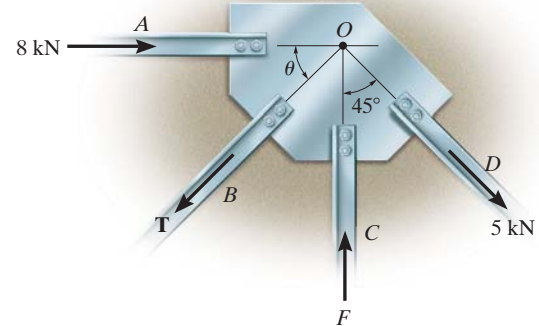
$$T = 13.32 = 13.3 \text{ kN} \quad \text{Ans}$$

$$+\uparrow \Sigma F_y = 0; \quad F - 13.32 \sin 30^\circ - 5 \cos 45^\circ = 0$$

$$F = 10.2 \text{ kN} \quad \text{Ans}$$



3–6. The gusset plate is subjected to the forces of four members. Determine the force in member  $B$  and its proper orientation  $\theta$  for equilibrium. The forces are concurrent at point  $O$ . Take  $F = 12 \text{ kN}$ .



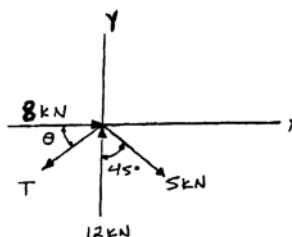
$$\rightarrow \Sigma F_x = 0; \quad 8 - T \cos \theta + 5 \sin 45^\circ = 0$$

$$+\uparrow \Sigma F_y = 0; \quad 12 - T \sin \theta - 5 \cos 45^\circ = 0$$

Solving,

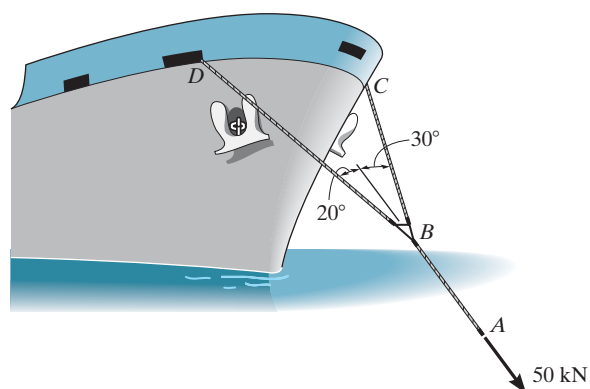
$$T = 14.3 \text{ kN} \quad \text{Ans}$$

$$\theta = 36.3^\circ \quad \text{Ans}$$



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**3–7.** The towing pendant  $AB$  is subjected to the force of 50 kN exerted by a tugboat. Determine the force in each of the bridles,  $BC$  and  $BD$ , if the ship is moving forward with constant velocity.



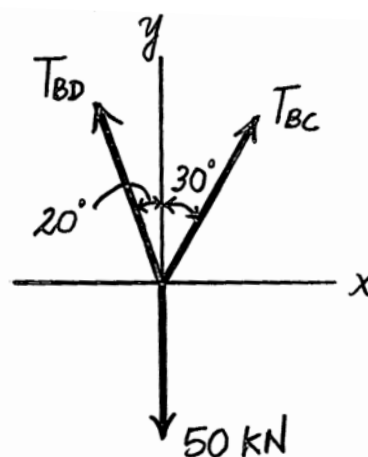
$$\rightarrow \Sigma F_x = 0; \quad T_{BC} \sin 30^\circ - T_{BD} \sin 20^\circ = 0$$

$$+\uparrow \Sigma F_y = 0; \quad T_{BC} \cos 30^\circ + T_{BD} \cos 20^\circ - 50 = 0$$

Solving,

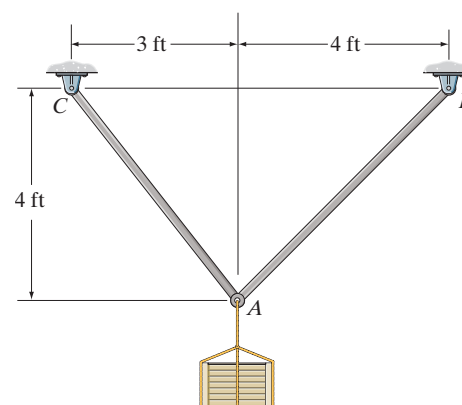
$$T_{BC} = 22.3 \text{ kN} \quad \text{Ans}$$

$$T_{BD} = 32.6 \text{ kN} \quad \text{Ans}$$



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**\*3–8.** Members  $AC$  and  $AB$  support the 300-lb crate. Determine the tensile force developed in each member.



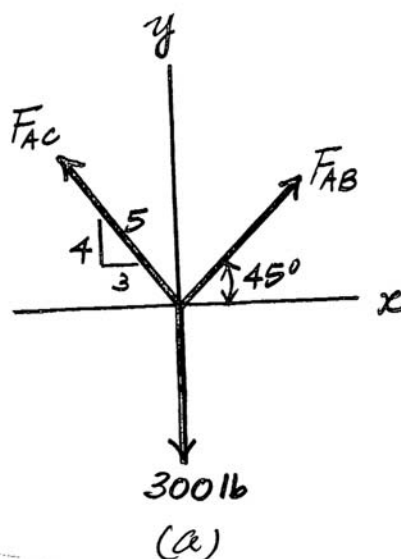
**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free - body diagram in Fig. (a),

$$+\rightarrow \Sigma F_x = 0; \quad F_{AB} \cos 45^\circ - F_{AC} \left( \frac{3}{5} \right) = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{AB} \sin 45^\circ + F_{AC} \left( \frac{4}{5} \right) - 300 = 0 \quad (2)$$

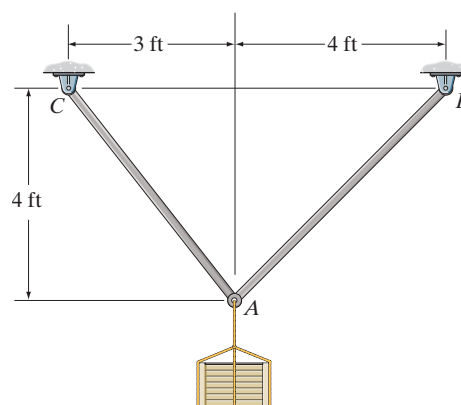
Solving Eqs. (1) and (2), yields

$$F_{AC} = 214 \text{ lb} \quad F_{AB} = 182 \text{ lb} \quad \text{Ans.}$$



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•3–9. If members  $AC$  and  $AB$  can support a maximum tension of 300 lb and 250 lb, respectively, determine the largest weight of the crate that can be safely supported.



**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free - body diagram in Fig. (a),

$$\rightarrow \Sigma F_x = 0; \quad F_{AB} \cos 45^\circ - F_{AC} \left( \frac{3}{5} \right) = 0 \quad (1)$$

$$+ \uparrow \Sigma F_y = 0; \quad F_{AB} \sin 45^\circ + F_{AC} \left( \frac{4}{5} \right) - W = 0 \quad (2)$$

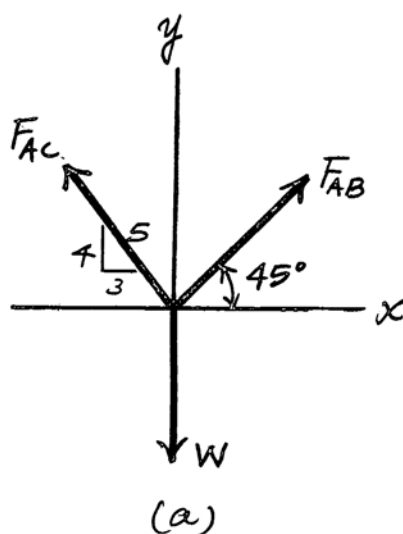
Assuming that rod  $AB$  will break first,  $F_{AB} = 250$  lb. Substituting this value into Eqs. (1) and (2),

$$F_{AC} = 294.63 \text{ lb}$$

$$W = 412 \text{ lb}$$

**Ans.**

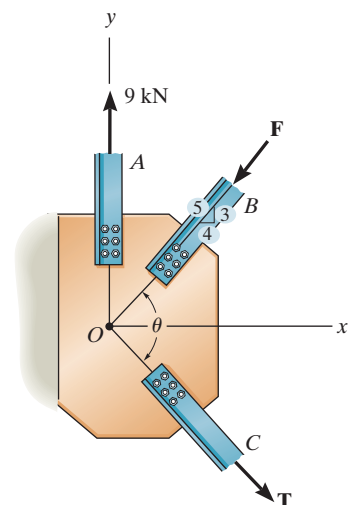
Since  $F_{AC} = 294.63 \text{ lb} < 300 \text{ lb}$ , rod  $AC$  will not break as assumed.





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**3–10.** The members of a truss are connected to the gusset plate. If the forces are concurrent at point  $O$ , determine the magnitudes of  $F$  and  $T$  for equilibrium. Take  $\theta = 90^\circ$ .



$$\phi = 90^\circ - \tan^{-1}\left(\frac{3}{4}\right) = 53.13^\circ$$

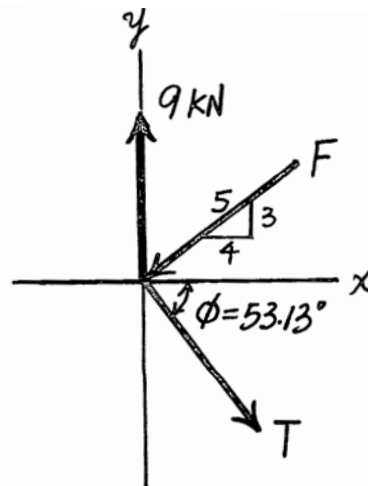
$$\rightarrow \Sigma F_x = 0; T \cos 53.13^\circ - F\left(\frac{4}{5}\right) = 0$$

$$+\uparrow \Sigma F_y = 0; 9 - T \sin 53.13^\circ - F\left(\frac{3}{5}\right) = 0$$

Solving,

$$T = 7.20 \text{ kN} \quad \text{Ans}$$

$$F = 5.40 \text{ kN} \quad \text{Ans}$$



**3–11.** The gusset plate is subjected to the forces of three members. Determine the tension force in member  $C$  and its angle  $\theta$  for equilibrium. The forces are concurrent at point  $O$ . Take  $F = 8 \text{ kN}$ .

$$\rightarrow \Sigma F_x = 0; T \cos \phi - 8\left(\frac{4}{5}\right) = 0 \quad (1)$$

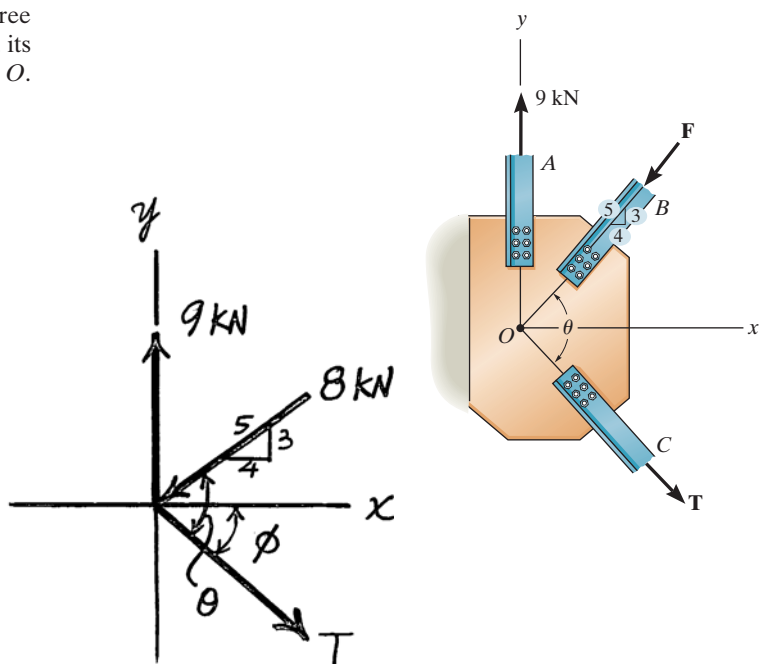
$$+\uparrow \Sigma F_y = 0; 9 - 8\left(\frac{3}{5}\right) - T \sin \phi = 0 \quad (2)$$

Rearrange then divide Eq. (1) into Eq. (2):

$$\tan \phi = 0.656, \quad \phi = 33.27^\circ$$

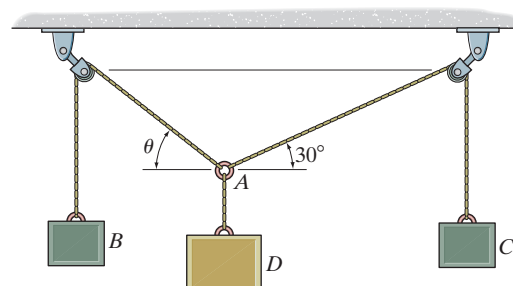
$$T = 7.66 \text{ kN} \quad \text{Ans}$$

$$\theta = \phi + \tan^{-1}\left(\frac{3}{4}\right) = 70.1^\circ \quad \text{Ans}$$



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**\*3–12.** If block *B* weighs 200 lb and block *C* weighs 100 lb, determine the required weight of block *D* and the angle  $\theta$  for equilibrium.



**Equations of Equilibrium:** Applying the equations of equilibrium along the *x* and *y* axes to the free-body diagram shown in Fig. (a),

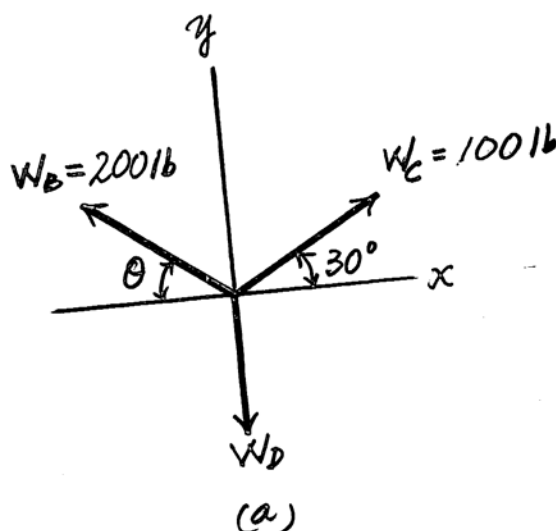
$$\begin{aligned} \rightarrow \Sigma F_x = 0; \quad & 100 \cos 30^\circ - 200 \cos \theta = 0 \\ & \theta = 64.34^\circ = 64.3^\circ \end{aligned}$$

**Ans.**

Using this result and writing the equation of equilibrium along the *y* axis, yields

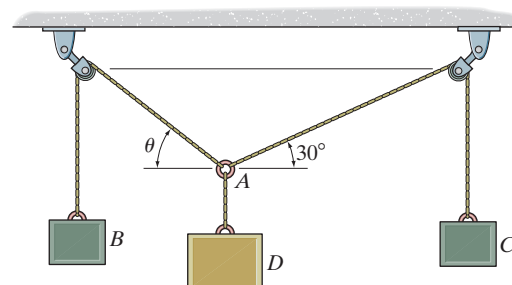
$$\begin{aligned} + \uparrow \Sigma F_y = 0; \quad & 100 \sin 30^\circ + 200 \sin 64.34^\circ - W_D = 0 \\ & W_D = 230 \text{ lb} \end{aligned}$$

**Ans.**



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•3–13. If block  $D$  weighs 300 lb and block  $B$  weighs 275 lb, determine the required weight of block  $C$  and the angle  $\theta$  for equilibrium.



**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram shown in Fig. (a),

$$+\rightarrow \Sigma F_x = 0; \quad W_C \cos 30^\circ - 275 \cos \theta = 0 \quad (1)$$

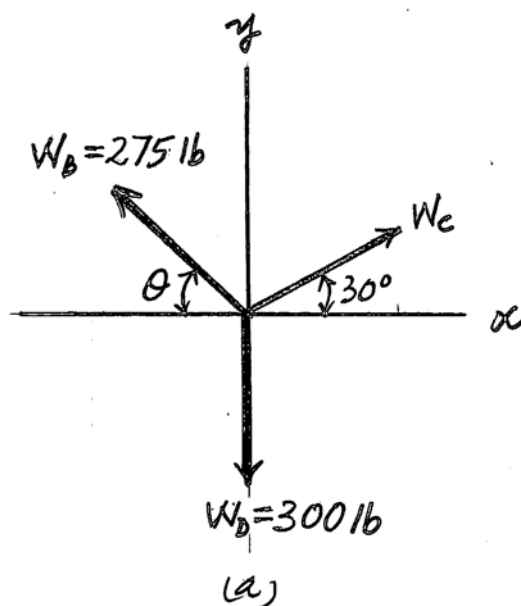
$$+\uparrow \Sigma F_y = 0; \quad W_C \sin 30^\circ + 275 \sin \theta - 300 = 0 \quad (2)$$

Solving Eqs. (1) and (2), yields

$$\theta = 40.9^\circ$$

$$W_C = 240 \text{ lb}$$

Ans.



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**3–14.** Determine the stretch in springs  $AC$  and  $AB$  for equilibrium of the 2-kg block. The springs are shown in the equilibrium position.

$$F_{AD} = 2(9.81) = x_{AD}(40)$$

$$x_{AD} = 0.4905 \text{ m}$$

$$\rightarrow \Sigma F_x = 0; \quad F_{AB}\left(\frac{4}{5}\right) - F_{AC}\left(\frac{1}{\sqrt{2}}\right) = 0$$

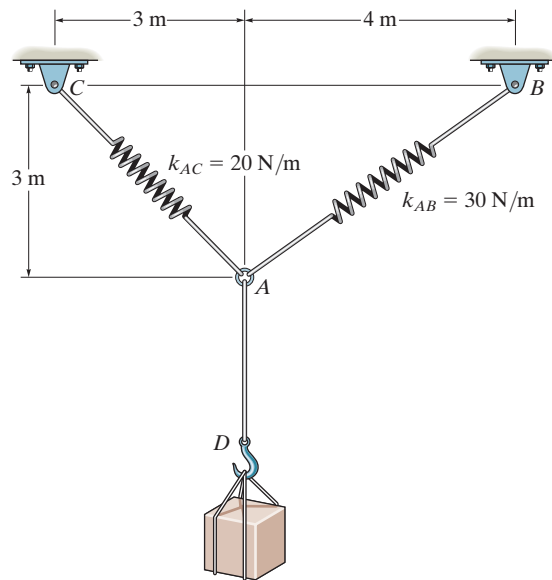
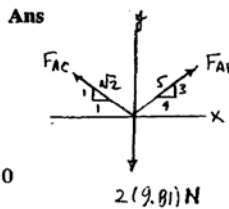
$$+ \uparrow \Sigma F_y = 0; \quad F_{AC}\left(\frac{1}{\sqrt{2}}\right) + F_{AB}\left(\frac{3}{5}\right) - 2(9.81) = 0$$

$$F_{AC} = 15.86 \text{ N}$$

$$x_{AC} = \frac{15.86}{20} = 0.793 \text{ m} \quad \text{Ans}$$

$$F_{AB} = 14.01 \text{ N}$$

$$x_{AB} = \frac{14.01}{30} = 0.467 \text{ m} \quad \text{Ans}$$



**3–15.** The unstretched length of spring  $AB$  is 3 m. If the block is held in the equilibrium position shown, determine the mass of the block at  $D$ .

$$F = kx = 30(5 - 3) = 60 \text{ N}$$

$$\rightarrow \Sigma F_x = 0; \quad T \cos 45^\circ - 60\left(\frac{4}{5}\right) = 0$$

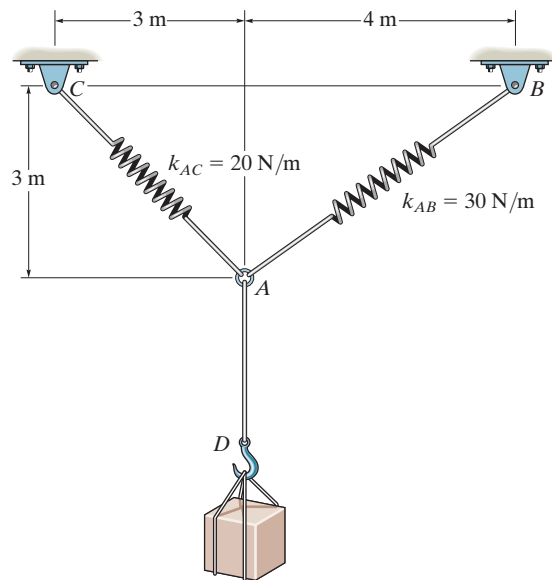
$$T = 67.88 \text{ N}$$

$$+ \uparrow \Sigma F_y = 0; \quad -W + 67.88 \sin 45^\circ + 60\left(\frac{3}{5}\right) = 0$$

$$W = 84 \text{ N}$$

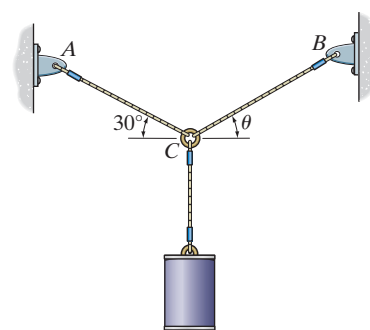
$$m = \frac{84}{9.81} = 8.56 \text{ kg}$$

**Ans.**



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**\*3-16.** Determine the tension developed in wires  $CA$  and  $CB$  required for equilibrium of the 10-kg cylinder. Take  $\theta = 40^\circ$ .



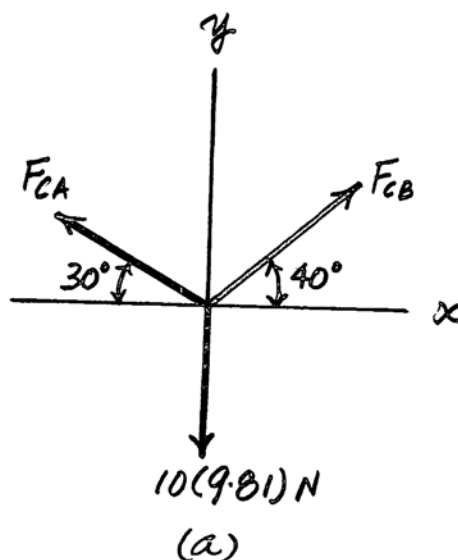
**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram shown in Fig. (a),

$$+\rightarrow \Sigma F_x = 0; \quad F_{CB} \cos 40^\circ - F_{CA} \cos 30^\circ = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{CB} \sin 40^\circ + F_{CA} \sin 30^\circ - 10(9.81) = 0 \quad (2)$$

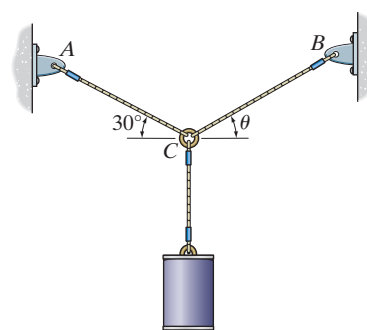
Solving Eqs. (1) and (2), yields

$$F_{CA} = 80.0 \text{ N} \quad F_{CB} = 90.4 \text{ N} \quad \text{Ans.}$$



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•3–17. If cable  $CB$  is subjected to a tension that is twice that of cable  $CA$ , determine the angle  $\theta$  for equilibrium of the 10-kg cylinder. Also, what are the tensions in wires  $CA$  and  $CB$ ?



**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes,

$$+\rightarrow \Sigma F_x = 0; \quad F_{CB} \cos \theta - F_{CA} \cos 30^\circ = 0 \quad (1)$$

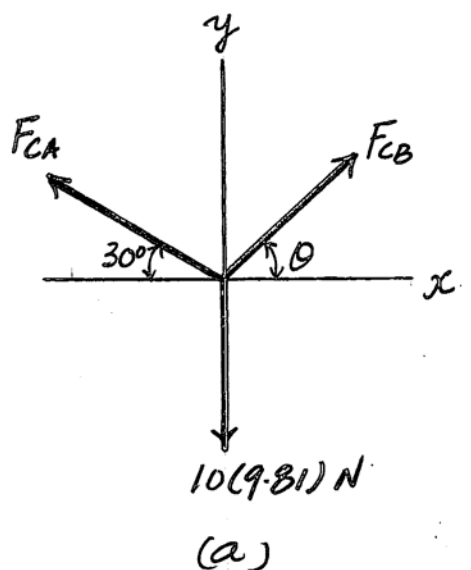
$$+\uparrow \Sigma F_y = 0; \quad F_{CB} \sin \theta + F_{CA} \sin 30^\circ - 10(9.81) = 0 \quad (2)$$

However, it is required that

$$F_{CB} = 2F_{CA} \quad (3)$$

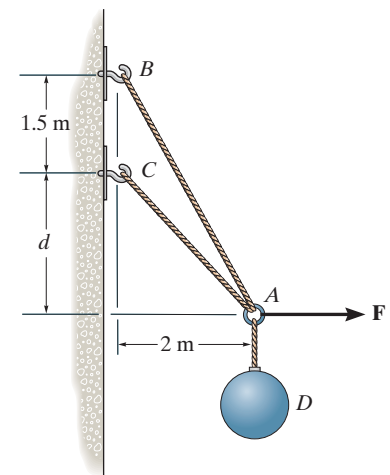
Solving Eqs. (1) and (2), yields

$$\theta = 64.3^\circ \quad F_{CB} = 85.2 \text{ N} \quad F_{CA} = 42.6 \text{ N} \quad \text{Ans.}$$



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**3–18.** Determine the forces in cables  $AC$  and  $AB$  needed to hold the 20-kg ball  $D$  in equilibrium. Take  $F = 300\text{ N}$  and  $d = 1\text{ m}$ .



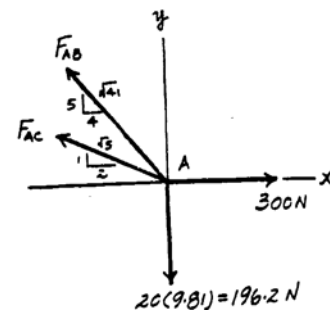
**Equations of Equilibrium :**

$$\begin{aligned} \rightarrow \Sigma F_x = 0; \quad 300 - F_{AB} \left( \frac{4}{\sqrt{41}} \right) - F_{AC} \left( \frac{2}{\sqrt{5}} \right) &= 0 \\ 0.6247 F_{AB} + 0.8944 F_{AC} &= 300 \end{aligned} \quad [1]$$

$$\begin{aligned} + \uparrow \Sigma F_y = 0; \quad F_{AB} \left( \frac{5}{\sqrt{41}} \right) + F_{AC} \left( \frac{1}{\sqrt{5}} \right) - 196.2 &= 0 \\ 0.7809 F_{AB} + 0.4472 F_{AC} &= 196.2 \end{aligned} \quad [2]$$

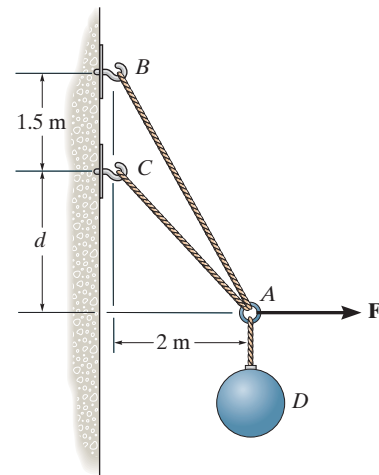
Solving Eqs. [1] and [2] yields

$$F_{AB} = 98.6\text{ N} \quad F_{AC} = 267\text{ N} \quad \text{Ans}$$



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**3–19.** The ball  $D$  has a mass of 20 kg. If a force of  $F = 100$  N is applied horizontally to the ring at  $A$ , determine the dimension  $d$  so that the force in cable  $AC$  is zero.



**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad 100 - F_{AB} \cos \theta = 0 \quad F_{AB} \cos \theta = 100 \quad [1]$$

$$+ \uparrow \Sigma F_y = 0; \quad F_{AB} \sin \theta - 196.2 = 0 \quad F_{AB} \sin \theta = 196.2 \quad [2]$$

Solving Eqs. [1] and [2] yields

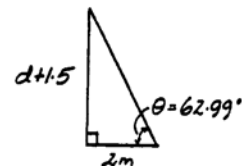
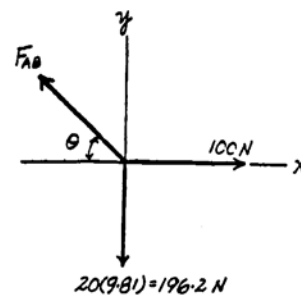
$$\theta = 62.99^\circ \quad F_{AB} = 220.21 \text{ N}$$

From the geometry,

$$d + 1.5 = 2 \tan 62.99^\circ$$

$$d = 2.42 \text{ m}$$

**Ans**





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**\*3–20.** Determine the tension developed in each wire used to support the 50-kg chandelier.

**Equations of Equilibrium:** First, we will apply the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $D$  shown in Fig. (a).

$$+\rightarrow \Sigma F_x = 0; \quad F_{CD} \cos 30^\circ - F_{BD} \cos 45^\circ = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{CD} \sin 30^\circ + F_{BD} \sin 45^\circ - 50(9.81) = 0 \quad (2)$$

Solving Eqs. (1) and (2), yields

$$F_{CD} = 359 \text{ N}$$

$$F_{BD} = 439.77 \text{ N} = 440 \text{ N}$$

**Ans.**

Using the result  $F_{BD} = 439.77 \text{ N}$  and applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $B$  shown in Fig. (b),

$$+\uparrow \Sigma F_y = 0; \quad F_{AB} \sin 30^\circ - 439.77 \sin 45^\circ = 0$$

$$F_{AB} = 621.93 \text{ N} = 622 \text{ N}$$

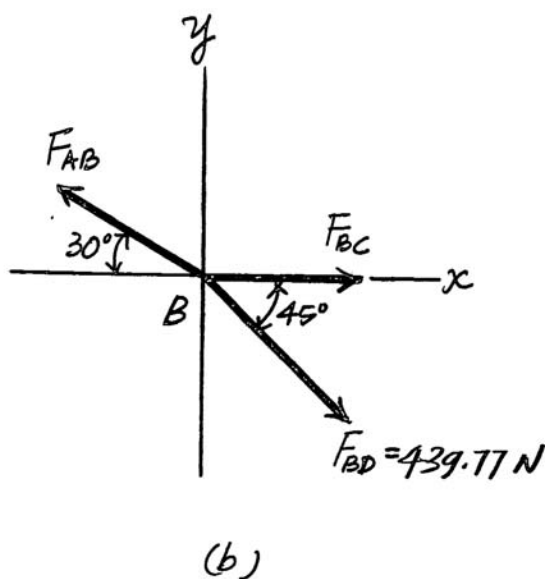
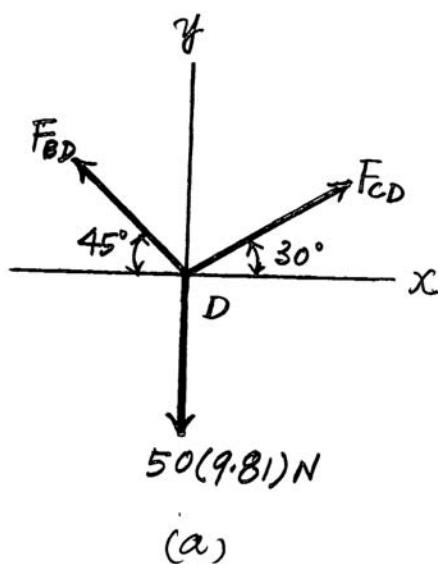
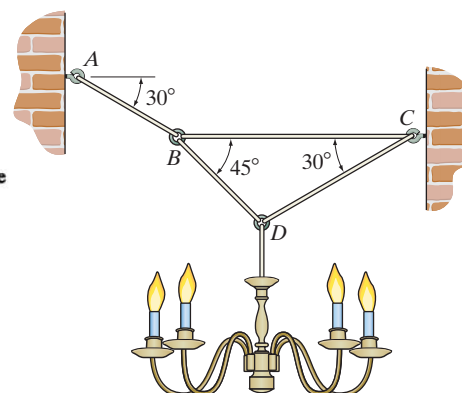
**Ans.**

$$+\rightarrow \Sigma F_x = 0;$$

$$F_{BC} + 439.77 \cos 45^\circ - 621.93 \cos 30^\circ = 0$$

$$F_{BC} = 228 \text{ N}$$

**Ans.**



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•3–21. If the tension developed in each of the four wires is not allowed to exceed 600 N, determine the maximum mass of the chandelier that can be supported.

**Equations of Equilibrium:** First, we will apply the equation of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $D$  shown in Fig. (a).

$$+\rightarrow \Sigma F_x = 0; \quad F_{CD} \cos 30^\circ - F_{BD} \cos 45^\circ = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{CD} \sin 30^\circ + F_{BD} \sin 45^\circ - m(9.81) = 0 \quad (2)$$

Solving Eqs. (1) and (2), yields

$$F_{CD} = 7.1814m \quad F_{BD} = 8.7954m$$

Using the result  $F_{BD} = 8.7954m$  and applying the equation of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $B$  shown in Fig. (b),

$$+\uparrow \Sigma F_y = 0; \quad F_{AB} \sin 30^\circ - 8.7954m \sin 45^\circ = 0$$

$$F_{AB} = 12.4386m$$

$$+\rightarrow \Sigma F_x = 0; \quad F_{BC} + 8.7954m \cos 45^\circ - 12.4386m \cos 30^\circ = 0$$

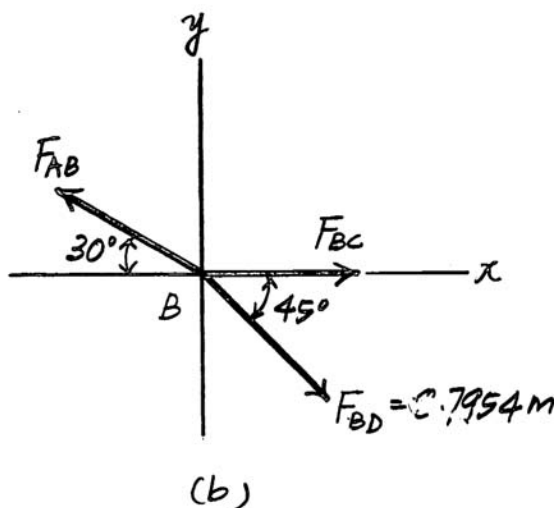
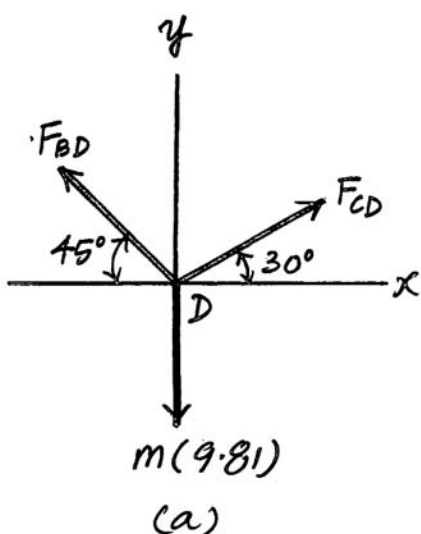
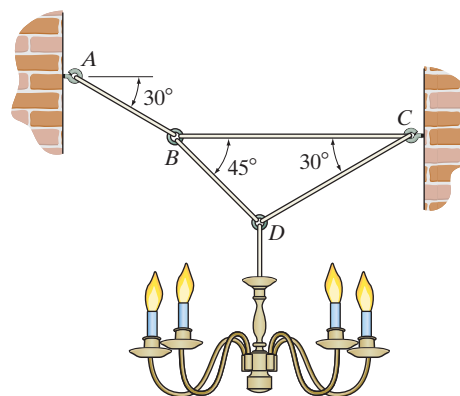
$$F_{BC} = 4.5528m$$

From this result, notice that cable  $AB$  is subjected to the greatest tensile force. Thus, it will achieve the maximum allowable tensile force first.

$$F_{AB} = 600 = 12.4386m$$

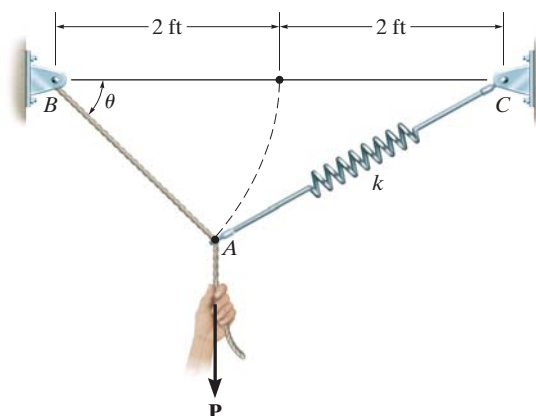
$$m = 48.2 \text{ kg}$$

**Ans.**



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■3-22. A vertical force  $P = 10$  lb is applied to the ends of the 2-ft cord  $AB$  and spring  $AC$ . If the spring has an unstretched length of 2 ft, determine the angle  $\theta$  for equilibrium. Take  $k = 15$  lb/ft.



$$\rightarrow \Sigma F_x = 0; \quad F_s \cos \phi - T \cos \theta = 0 \quad (1)$$

$$+ \uparrow \Sigma F_y = 0; \quad T \sin \theta + F_s \sin \phi - 10 = 0 \quad (2)$$

$$s = \sqrt{(4)^2 + (2)^2 - 2(4)(2)\cos\theta} = 2\sqrt{5-4\cos\theta} - 2$$

$$F_s = ks = 2k(\sqrt{5-4\cos\theta} - 1)$$

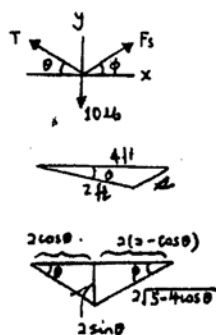
$$\text{From Eq. (1):} \quad T = F_s \left( \frac{\cos \phi}{\cos \theta} \right)$$

$$T = 2k(\sqrt{5-4\cos\theta} - 1) \left( \frac{2 - \cos\theta}{\sqrt{5-4\cos\theta}} \right) \left( \frac{1}{\cos\theta} \right)$$

From Eq. (2):

$$\frac{2k(\sqrt{5-4\cos\theta} - 1)(2 - \cos\theta)}{\sqrt{5-4\cos\theta}} \tan \theta + \frac{2k(\sqrt{5-4\cos\theta} - 1)2\sin\theta}{2\sqrt{5-4\cos\theta}} = 10$$

$$\frac{(\sqrt{5-4\cos\theta} - 1)}{\sqrt{5-4\cos\theta}} (2\tan\theta - \sin\theta + \sin\theta) = \frac{10}{2k}$$



$$\frac{\tan \theta (\sqrt{5-4\cos\theta} - 1)}{\sqrt{5-4\cos\theta}} = \frac{10}{4k}$$

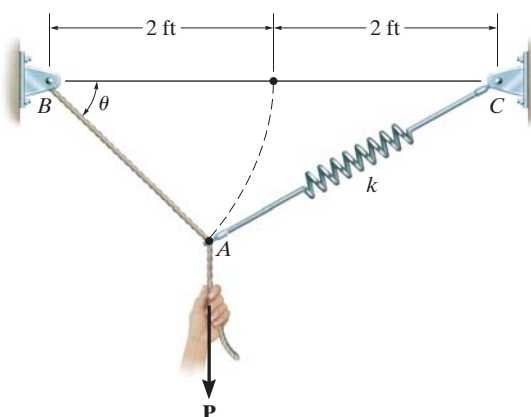
Set  $k = 15$  lb/ft

Solving for  $\theta$  by trial and error,

$$\theta = 35.0^\circ \quad \text{Ans}$$

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**3–23.** Determine the unstretched length of spring  $AC$  if a force  $P = 80$  lb causes the angle  $\theta = 60^\circ$  for equilibrium. Cord  $AB$  is 2 ft long. Take  $k = 50$  lb/ft.



$$l = \sqrt{4^2 + 2^2 - 2(2)(4)\cos 60^\circ}$$

$$l = \sqrt{12}$$

$$\frac{\sqrt{12}}{\sin 60^\circ} = \frac{2}{\sin \phi}$$

$$\phi = \sin^{-1}\left(\frac{2 \sin 60^\circ}{\sqrt{12}}\right) = 30^\circ$$

$$+\uparrow \Sigma F_y = 0; \quad T \sin 60^\circ + F_s \sin 30^\circ - 80 = 0$$

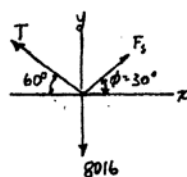
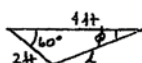
$$+\rightarrow \Sigma F_x = 0; \quad -T \cos 60^\circ + F_s \cos 30^\circ = 0$$

Solving for  $F_s$ ,

$$F_s = 40 \text{ lb}$$

$$F_s = kx$$

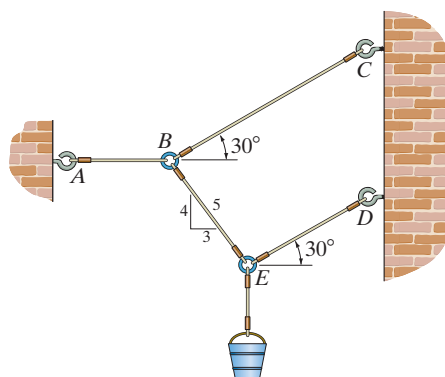
$$40 = 50(\sqrt{12} - l')$$



$$l = \sqrt{12} - \frac{40}{50} = 2.66 \text{ ft} \quad \text{Ans.}$$

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\*3–24. If the bucket weighs 50 lb, determine the tension developed in each of the wires.



**Equations of Equilibrium:** First, we will apply the equation of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $E$  shown in Fig. (a).

$$+\rightarrow \Sigma F_x = 0; \quad F_{ED} \cos 30^\circ - F_{EB} \left( \frac{3}{5} \right) = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{ED} \sin 30^\circ + F_{EB} \left( \frac{4}{5} \right) - 50 = 0 \quad (2)$$

Solving Eqs. (1) and (2), yields

$$F_{ED} = 30.2 \text{ lb}$$

$$F_{EB} = 43.61 \text{ lb} = 43.6 \text{ lb}$$

**Ans.**

Using the result  $F_{EB} = 43.61 \text{ lb}$  and applying the equation of equilibrium to the free-body diagram of joint  $B$  shown in Fig. (b),

$$+\uparrow \Sigma F_y = 0; \quad F_{BC} \sin 30^\circ - 43.61 \left( \frac{4}{5} \right) = 0$$

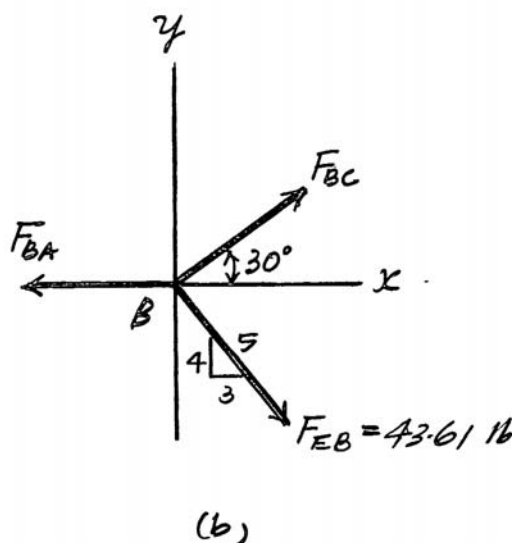
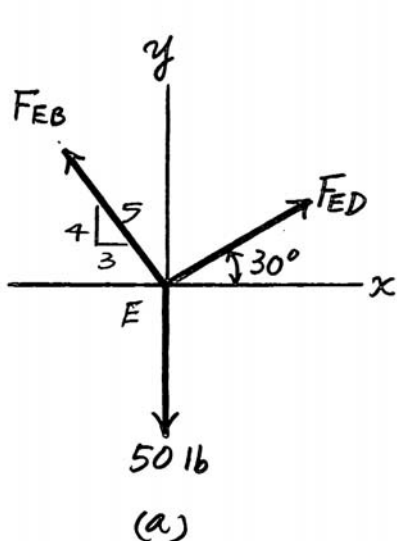
$$F_{BC} = 69.78 \text{ lb} = 69.8 \text{ lb}$$

**Ans.**

$$+\rightarrow \Sigma F_x = 0; \quad 69.78 \cos 30^\circ + 43.61 \left( \frac{3}{5} \right) - F_{BA} = 0$$

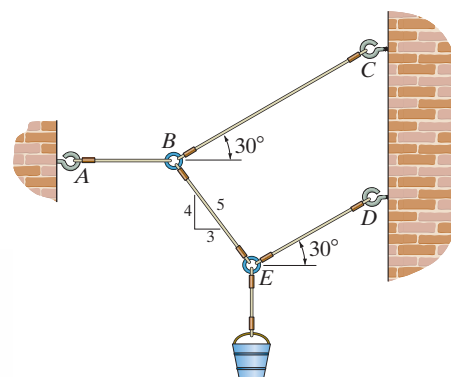
$$F_{BA} = 86.6 \text{ lb}$$

**Ans.**



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•3–25. Determine the maximum weight of the bucket that the wire system can support so that no single wire develops a tension exceeding 100 lb.



**Equations of Equilibrium:** First, we will apply the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $E$  shown in Fig. (a).

$$+\rightarrow \Sigma F_x = 0; \quad F_{ED} \cos 30^\circ - F_{EB} \left( \frac{3}{5} \right) = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{ED} \sin 30^\circ + F_{EB} \left( \frac{4}{5} \right) - W = 0 \quad (2)$$

Solving,

$$F_{EB} = 0.8723W \quad F_{ED} = 0.6043W$$

Using the result  $F_{EB} = 0.8723W$  and applying the equations of equilibrium to the free-body diagram of joint  $B$  shown in Fig. (b),

$$+\uparrow \Sigma F_y = 0; \quad F_{BC} \sin 30^\circ - 0.8723W \left( \frac{4}{5} \right) = 0$$

$$F_{BC} = 1.3957W$$

$$+\rightarrow \Sigma F_x = 0; \quad 1.3957W \cos 30^\circ + 0.8723W \left( \frac{3}{5} \right) - F_{BA} = 0$$

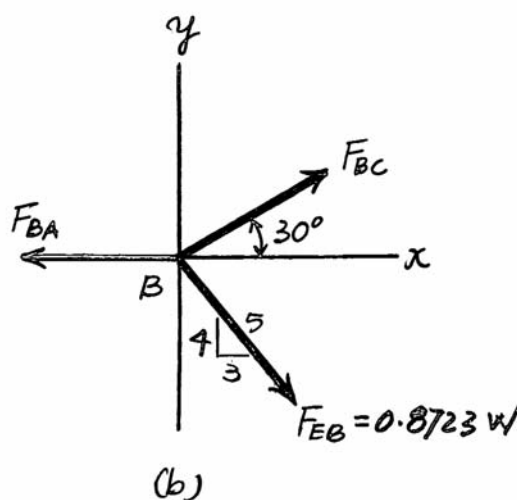
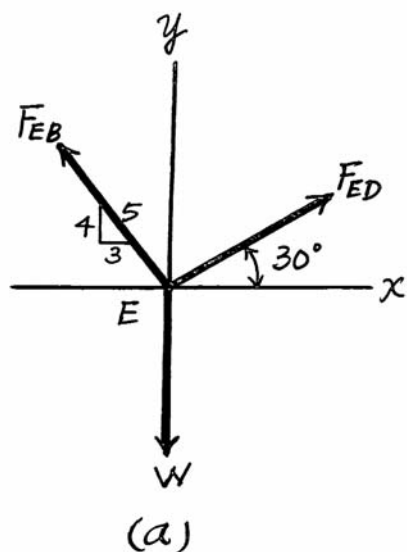
$$F_{BA} = 1.7320W$$

From these results, notice that wire  $BA$  is subjected to the greatest tensile force. Thus, it will achieve the maximum allowable tensile force first.

$$F_{BA} = 100 = 1.7320W$$

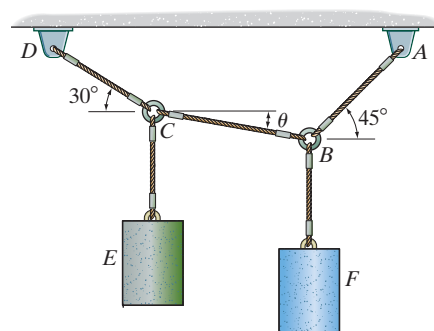
$$W = 57.7 \text{ lb}$$

Ans.



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**3–26.** Determine the tensions developed in wires  $CD$ ,  $CB$ , and  $BA$  and the angle  $\theta$  required for equilibrium of the 30-lb cylinder  $E$  and the 60-lb cylinder  $F$ .



**Equations of Equilibrium:** Applying the equations of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $C$  shown in Fig. (a),

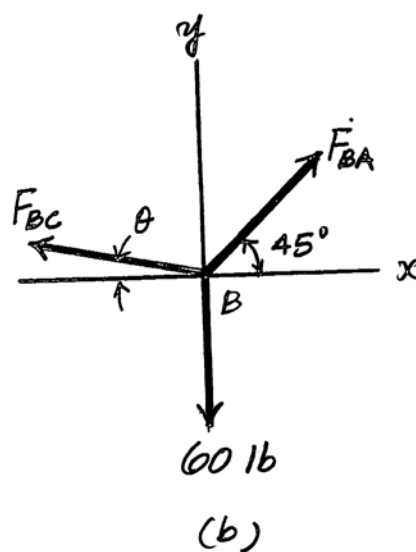
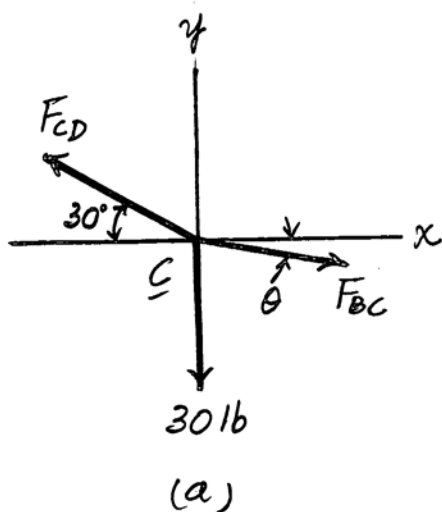
$$\begin{aligned} +\rightarrow \Sigma F_x &= 0; & F_{BC} \cos \theta - F_{CD} \cos 30^\circ &= 0 & (1) \\ +\uparrow \Sigma F_y &= 0; & -F_{BC} \sin \theta + F_{CD} \sin 30^\circ - 30 &= 0 & (2) \end{aligned}$$

By referring to the free-body diagram of joint  $B$  in Fig. (b),

$$\begin{aligned} +\rightarrow \Sigma F_x &= 0; & F_{BA} \cos 45^\circ - F_{BC} \cos \theta &= 0 & (3) \\ +\uparrow \Sigma F_y &= 0; & F_{BA} \sin 45^\circ + F_{BC} \sin \theta - 60 &= 0 & (4) \end{aligned}$$

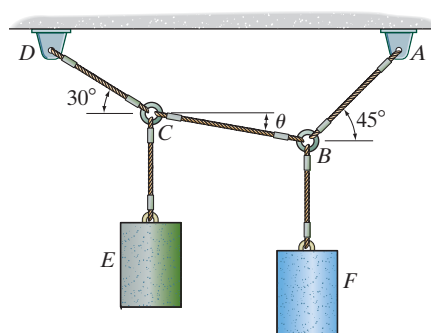
Solving Eqs. (1) through (4), yields

$$\begin{aligned} F_{BA} &= 80.7 \text{ lb} & \text{Ans.} \\ F_{CD} &= 65.9 \text{ lb} & \text{Ans.} \\ F_{BC} &= 57.1 \text{ lb} & \text{Ans.} \\ \theta &= 2.95^\circ & \text{Ans.} \end{aligned}$$



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3-27. If cylinder  $E$  weighs 30 lb and  $\theta = 15^\circ$ , determine the weight of cylinder  $F$ .



**Equations of Equilibrium:** First, we will apply the equation of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $C$  shown in Fig. (a).

$$+\rightarrow \Sigma F_x = 0; \quad F_{BC} \cos 15^\circ - F_{CD} \cos 30^\circ = 0 \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad F_{CD} \sin 30^\circ - F_{BC} \sin 15^\circ - 30 = 0 \quad (2)$$

Solving Eqs. (1) and (2), yields

$$F_{BC} = 100.38 \text{ lb} \quad F_{CD} = 111.96 \text{ lb} \quad \text{Ans.}$$

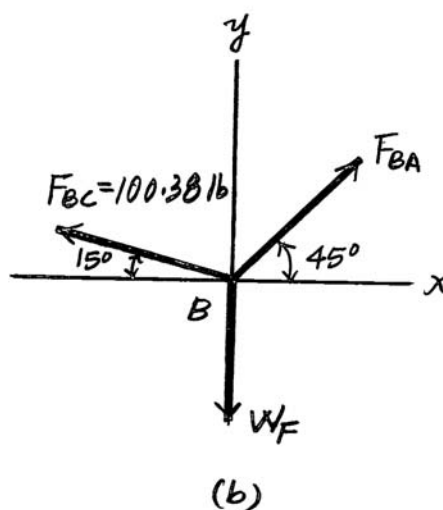
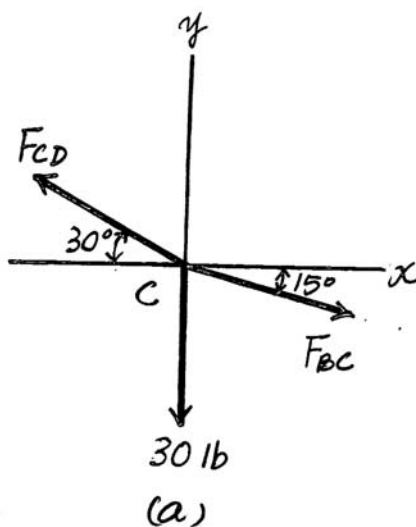
Using the result  $F_{BC} = 100.38 \text{ lb}$  and applying the equation of equilibrium along the  $x$  and  $y$  axes to the free-body diagram of joint  $B$  shown in Fig. (b),

$$+\rightarrow \Sigma F_x = 0; \quad F_{BA} \cos 45^\circ - 100.38 \cos 15^\circ = 0$$

$$F_{BA} = 137.12 \text{ lb} \quad \text{Ans.}$$

$$+\uparrow \Sigma F_y = 0; \quad 137.12 \sin 45^\circ + 100.38 \sin 15^\circ - W_F = 0$$

$$W_F = 123 \text{ lb} \quad \text{Ans.}$$





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**\*3–28.** Two spheres *A* and *B* have an equal mass and are electrostatically charged such that the repulsive force acting between them has a magnitude of 20 mN and is directed along line *AB*. Determine the angle  $\theta$ , the tension in cords *AC* and *BC*, and the mass *m* of each sphere.

For *B* :

$$\rightarrow \Sigma F_x = 0; \quad 0.02 \cos 30^\circ - T_B \sin 30^\circ = 0$$

$$+ \uparrow \Sigma F_y = 0; \quad 0.02 \sin 30^\circ + T_B \cos 30^\circ - W = 0$$

$$T_B = 0.0346 \text{ N} = 34.6 \text{ mN} \quad \text{Ans}$$

$$W = 0.04 \text{ N}$$

For *A* :

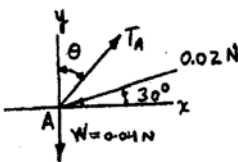
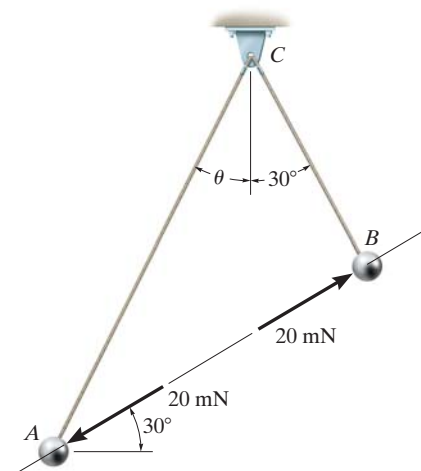
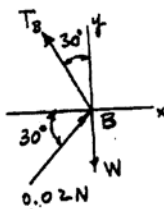
$$\rightarrow \Sigma F_x = 0; \quad T_A \sin \theta - 0.02 \cos 30^\circ = 0$$

$$+ \uparrow \Sigma F_y = 0; \quad T_A \cos \theta - 0.02 \sin 30^\circ - 0.04 = 0$$

$$T_A = 0.0529 \text{ N} = 52.9 \text{ mN} \quad \text{Ans}$$

$$\theta = 19.1^\circ \quad \text{Ans}$$

$$m = \frac{W}{g} = \frac{0.04}{9.81} = 4.08 (10^{-3}) \text{ kg} = 4.08 \text{ g} \quad \text{Ans}$$



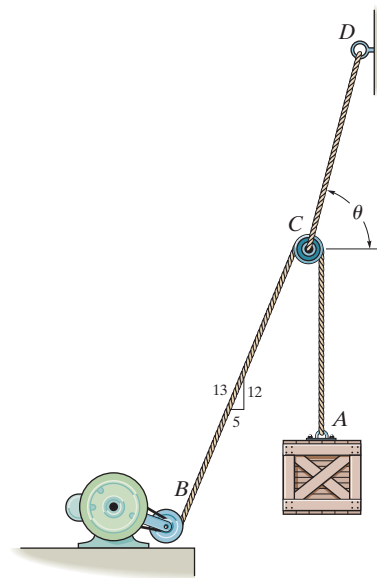
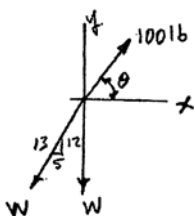
**•3–29.** The cords *BCA* and *CD* can each support a maximum load of 100 lb. Determine the maximum weight of the crate that can be hoisted at constant velocity and the angle  $\theta$  for equilibrium. Neglect the size of the smooth pulley at *C*.

$$\rightarrow \Sigma F_x = 0; \quad 100 \cos \theta = W \left( \frac{5}{13} \right)$$

$$+ \uparrow \Sigma F_y = 0; \quad 100 \sin \theta = W \left( \frac{12}{13} \right) + W$$

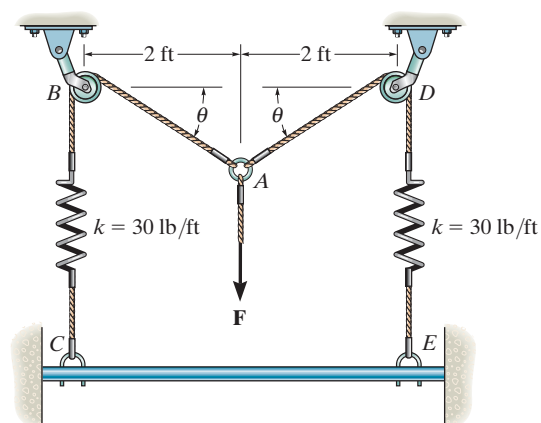
$$\theta = 78.7^\circ \quad \text{Ans}$$

$$W = 51.0 \text{ lb} \quad \text{Ans}$$



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**3–30.** The springs on the rope assembly are originally unstretched when  $\theta = 0^\circ$ . Determine the tension in each rope when  $F = 90$  lb. Neglect the size of the pulleys at  $B$  and  $D$ .



$$l = \frac{2}{\cos \theta}$$

$$T = kx = k(l - l_0) = 30\left(\frac{2}{\cos \theta} - 2\right) = 60\left(\frac{1}{\cos \theta} - 1\right) \quad (1)$$

$$+\uparrow \Sigma F_y = 0; \quad 2T \sin \theta - 90 = 0 \quad (2)$$

Substituting Eq. (1) into (2) yields :

$$120(\tan \theta - \sin \theta) - 90 = 0$$

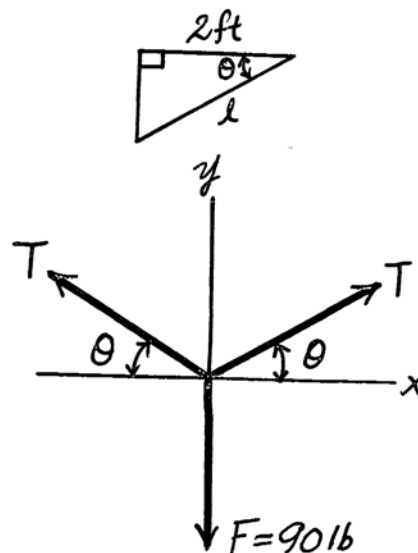
$$\tan \theta - \sin \theta = 0.75$$

By trial and error :

$$\theta = 57.957^\circ$$

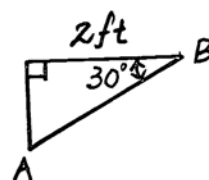
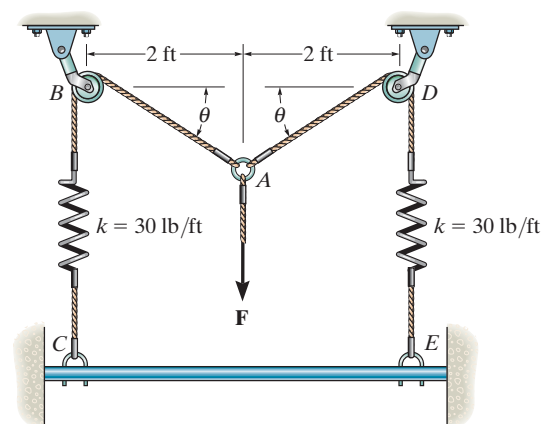
From Eq. (1),

$$T = 60\left(\frac{1}{\cos 57.957^\circ} - 1\right) = 53.1 \text{ lb} \quad \text{Ans}$$



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**3–31.** The springs on the rope assembly are originally stretched 1 ft when  $\theta = 0^\circ$ . Determine the vertical force  $F$  that must be applied so that  $\theta = 30^\circ$ .



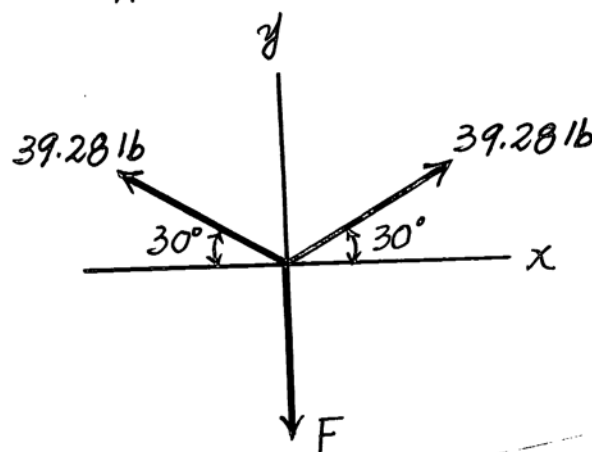
$$BA = \frac{2}{\cos 30^\circ} = 2.3094 \text{ ft}$$

When  $\theta = 30^\circ$ , the springs are stretched  $1 \text{ ft} + (2.3094 - 2) \text{ ft} = 1.3094 \text{ ft}$

$$F_s = kx = 30(1.3094) = 39.28 \text{ lb}$$

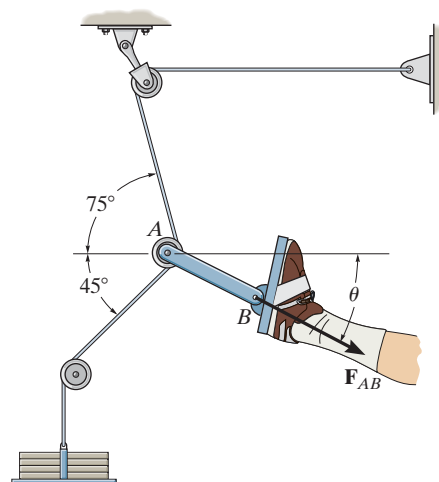
$$+\uparrow \Sigma F_y = 0; \quad 2(39.28) \sin 30^\circ - F = 0$$

$$F = 39.3 \text{ lb} \quad \text{Ans}$$



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**\*3–32.** Determine the magnitude and direction  $\theta$  of the equilibrium force  $F_{AB}$  exerted along link  $AB$  by the tractive apparatus shown. The suspended mass is 10 kg. Neglect the size of the pulley at  $A$ .



**Free Body Diagram :** The tension in the cord is the same throughout the cord, that is  $10(9.81) = 98.1$  N.

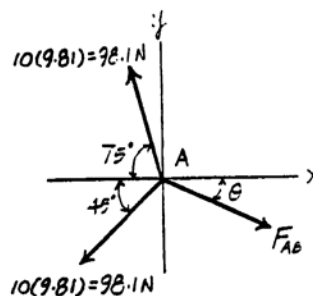
**Equations of Equilibrium :**

$$\begin{aligned} \rightarrow \Sigma F_x = 0; \quad F_{AB} \cos \theta - 98.1 \cos 75^\circ - 98.1 \cos 45^\circ &= 0 \\ F_{AB} \cos \theta &= 94.757 \end{aligned} \quad [1]$$

$$\begin{aligned} + \uparrow \Sigma F_y = 0; \quad 98.1 \sin 75^\circ - 98.1 \sin 45^\circ - F_{AB} \sin \theta &= 0 \\ F_{AB} \sin \theta &= 25.390 \end{aligned} \quad [2]$$

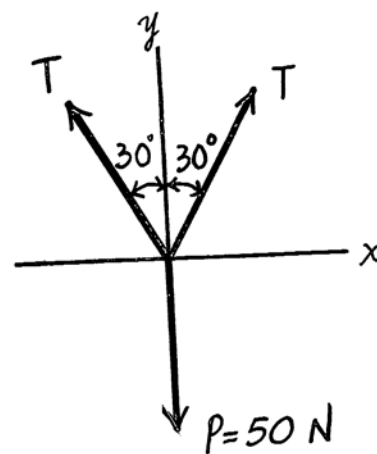
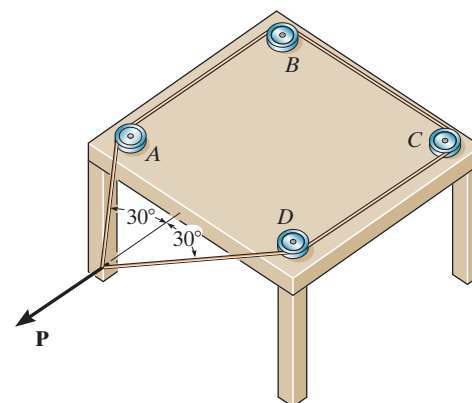
Solving Eqs. [1] and [2] yields

$$\theta = 15.0^\circ \quad F_{AB} = 98.1 \text{ N} \quad \text{Ans}$$



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•3–33. The wire forms a loop and passes over the small pulleys at  $A$ ,  $B$ ,  $C$ , and  $D$ . If its end is subjected to a force of  $P = 50$  N, determine the force in the wire and the magnitude of the resultant force that the wire exerts on each of the pulleys.



$$+\uparrow \Sigma F_y = 0; \quad 2(T \cos 30^\circ) - 50 = 0$$

$$T = 28.868 = 28.9 \text{ N} \quad \text{Ans}$$

For  $A$  and  $D$ :

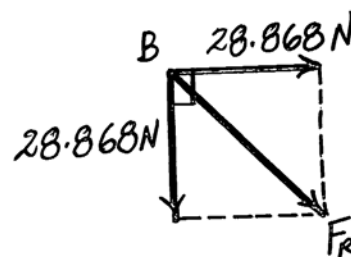
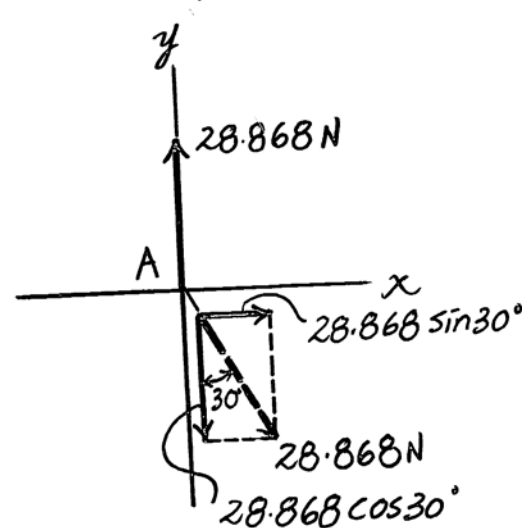
$$F_{Rx} = \Sigma F_x; \quad F_{Rx} = 28.868 \sin 30^\circ = 14.43 \text{ N}$$

$$F_{Ry} = \Sigma F_y; \quad F_{Ry} = 28.868 - 28.868 \cos 30^\circ = 3.868 \text{ N}$$

$$F_R = \sqrt{(14.43)^2 + (3.868)^2} = 14.9 \text{ N} \quad (A \text{ and } D) \quad \text{Ans}$$

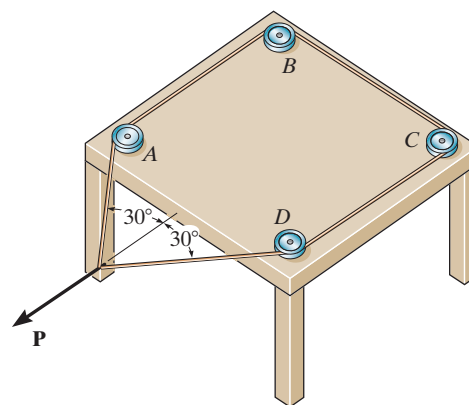
For  $B$  and  $C$ :

$$F_R = \sqrt{(28.868)^2 + (28.868)^2} = 40.8 \text{ N} \quad (B \text{ and } C) \quad \text{Ans}$$



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**3–34.** The wire forms a loop and passes over the small pulleys at  $A$ ,  $B$ ,  $C$ , and  $D$ . If the maximum *resultant* force that the wire can exert on each pulley is 120 N, determine the greatest force  $P$  that can be applied to the wire as shown.



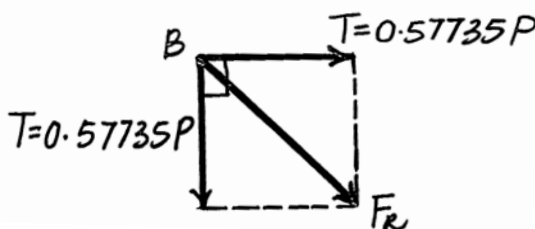
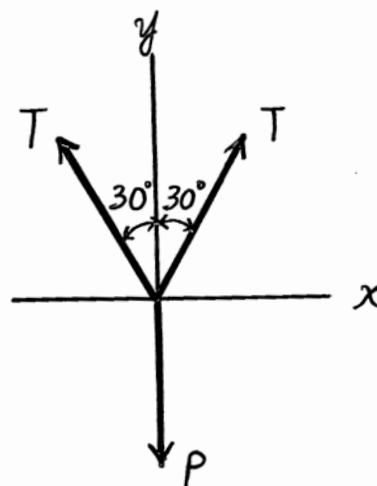
$$+\uparrow \Sigma F_y = 0; \quad 2T \cos 30^\circ - P = 0; \quad T = 0.57735 P$$

Maximum resultant force is resisted by pulleys  $B$  and  $C$ .

$$F_R = \sqrt{(0.57735 P)^2 + (0.57735 P)^2}$$

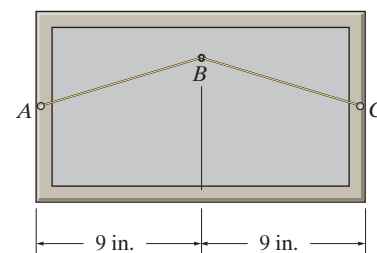
$$F_R = 0.8165 P = 120$$

$$P = 147 \text{ N} \quad \text{Ans}$$



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**3–35.** The picture has a weight of 10 lb and is to be hung over the smooth pin  $B$ . If a string is attached to the frame at points  $A$  and  $C$ , and the maximum force the string can support is 15 lb, determine the shortest string that can be safely used.



**Free Body Diagram :** Since the pin is smooth, the tension force in the cord is the same throughout the cord.

**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad T \cos \theta - T \cos \theta = 0 \quad (\text{Satisfied!})$$

$$+ \uparrow \Sigma F_y = 0; \quad 10 - 2T \sin \theta = 0 \quad T = \frac{5}{\sin \theta}$$

If tension in the cord cannot exceed 15 lb, then

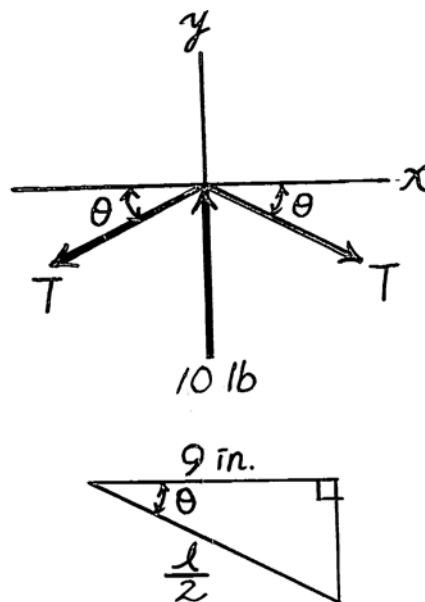
$$\frac{5}{\sin \theta} = 15$$

$$\theta = 19.47^\circ$$

From the geometry,  $\frac{l}{2} = \frac{9}{\cos \theta}$  and  $\theta = 19.47^\circ$ . Therefore

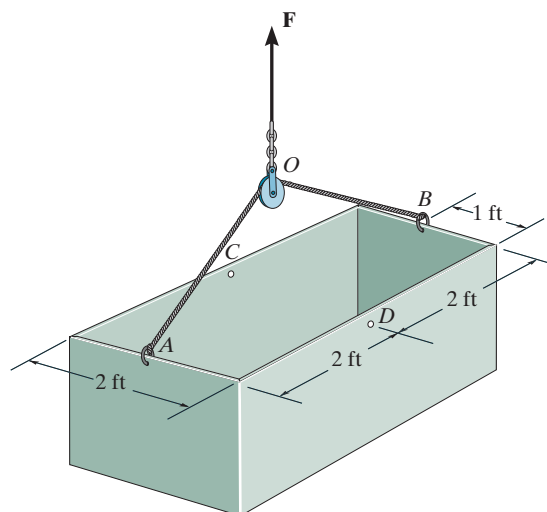
$$l = \frac{18}{\cos 19.47^\circ} = 19.1 \text{ in.}$$

**Ans**



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**\*3–36.** The 200-lb uniform tank is suspended by means of a 6-ft-long cable, which is attached to the sides of the tank and passes over the small pulley located at  $O$ . If the cable can be attached at either points  $A$  and  $B$  or  $C$  and  $D$ , determine which attachment produces the least amount of tension in the cable. What is this tension?



**Free Body Diagram :** By observation, the force  $F$  has to support the entire weight of the tank. Thus,  $F = 200$  lb. The tension in cable is the same throughout the cable.

**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad T \cos \theta - T \cos \theta = 0 \quad (\text{Satisfied!})$$

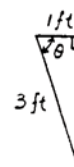
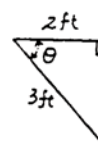
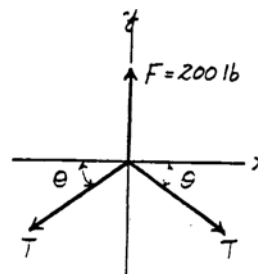
$$+ \uparrow \Sigma F_y = 0; \quad 200 - 2T \sin \theta = 0 \quad T = \frac{100}{\sin \theta} \quad [1]$$

From the function obtained above, one realizes that in order to produce the least amount of tension in the cable,  $\sin \theta$  hence  $\theta$  must be as great as possible. Since the attachment of the cable to point  $C$  and  $D$  produces a greater  $\theta$  ( $\theta = \cos^{-1} \frac{1}{3} = 70.53^\circ$ ) as compared to the attachment of the cable to points  $A$  and  $B$  ( $\theta = \cos^{-1} \frac{2}{3} = 48.19^\circ$ ),

The attachment of the cable to point  $C$  and  $D$  will produce the least amount of tension in the cable. **Ans**

Thus,

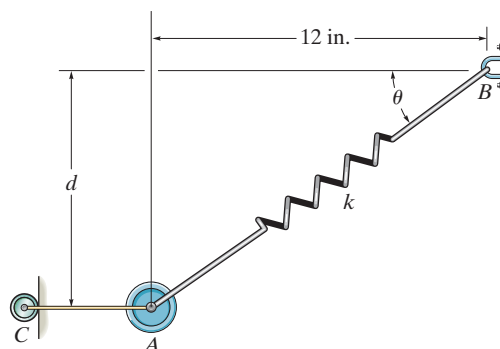
$$T = \frac{100}{\sin 70.53^\circ} = 106 \text{ lb} \quad \text{Ans}$$





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•**3-37.** The 10-lb weight is supported by the cord  $AC$  and roller and by the spring that has a stiffness of  $k = 10 \text{ lb/in.}$  and an unstretched length of 12 in. Determine the distance  $d$  to where the weight is located when it is in equilibrium.



$$\rightarrow \Sigma F_x = 0; \quad -T_{AC} + F_s \cos \theta = 0$$

$$+\uparrow \Sigma F_y = 0; \quad F_s \sin \theta - 10 = 0$$

$$F_s = kx; \quad F_s = 10 \left( \frac{12}{\cos \theta} - 12 \right)$$

$$= 120 (\sec \theta - 1)$$

Thus,

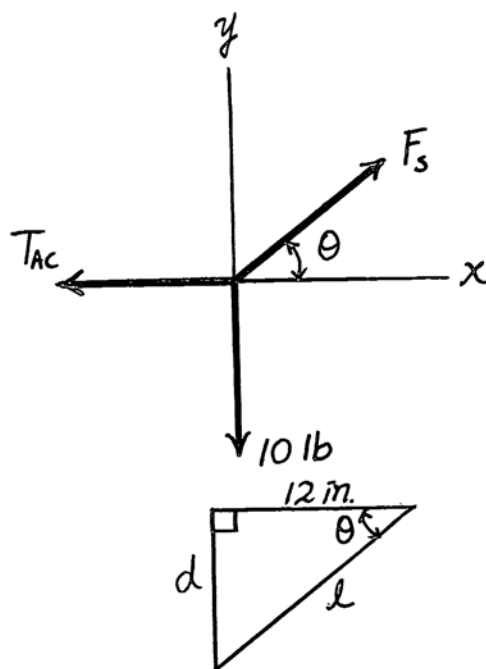
$$120 (\sec \theta - 1) \sin \theta = 10$$

$$(\tan \theta - \sin \theta) = \frac{1}{12}$$

Solving,

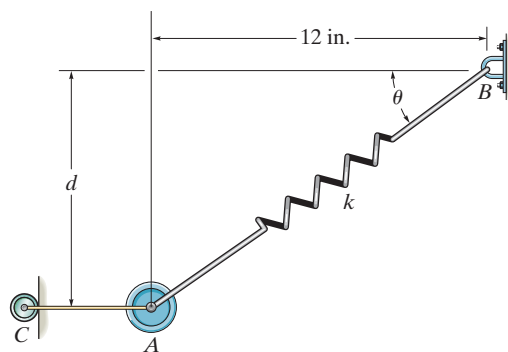
$$\theta = 30.71^\circ$$

$$d = 12 \tan 30.71^\circ = 7.13 \text{ in.} \quad \text{Ans}$$



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**3–38.** The 10-lb weight is supported by the cord  $AC$  and roller and by a spring. If the spring has an unstretched length of 8 in. and the weight is in equilibrium when  $d = 4$  in., determine the stiffness  $k$  of the spring.



$$+\uparrow \Sigma F_y = 0; \quad F_s \sin \theta - 10 = 0$$

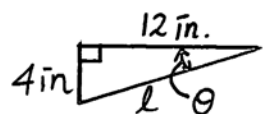
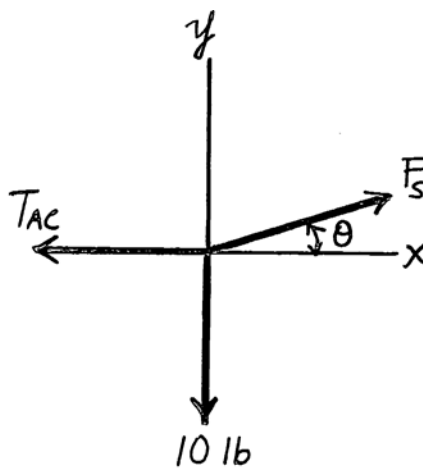
$$F_s = kx; \quad F_s = k\left(\frac{12}{\cos \theta} - 8\right)$$

$$\tan \theta = \frac{4}{12}; \quad \theta = 18.435^\circ$$

Thus,

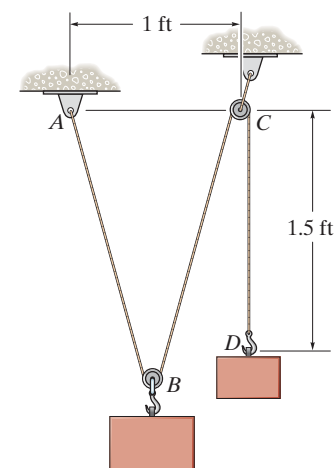
$$k\left(\frac{12}{\cos 18.435^\circ} - 8\right) \sin 18.435^\circ = 10$$

$$k = 6.80 \text{ lb/in.} \quad \text{Ans}$$



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•3–39. A “scale” is constructed with a 4-ft-long cord and the 10-lb block *D*. The cord is fixed to a pin at *A* and passes over two *small* pulleys at *B* and *C*. Determine the weight of the suspended block at *B* if the system is in equilibrium.



**Free Body Diagram :** The tension force in the cord is the same throughout the cord, that is 10 lb. From the geometry,

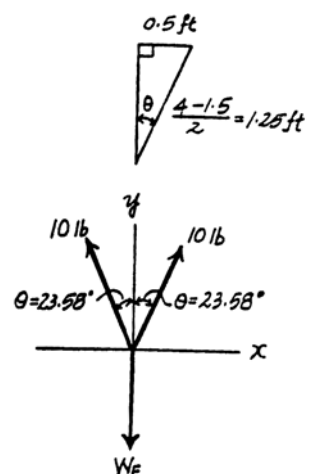
$$\theta = \sin^{-1}\left(\frac{0.5}{1.25}\right) = 23.58^\circ.$$

**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad 10 \sin 23.58^\circ - 10 \sin 23.58^\circ = 0 \quad (\text{Satisfied!})$$

$$+ \uparrow \Sigma F_y = 0; \quad 2(10) \cos 23.58^\circ - W_B = 0$$

$$W_B = 18.3 \text{ lb} \quad \text{Ans}$$



•\*3–40. The spring has a stiffness of  $k = 800 \text{ N/m}$  and an unstretched length of 200 mm. Determine the force in cables *BC* and *BD* when the spring is held in the position shown.

**The Force in The Spring :** The spring stretches  $s = l - l_0 = 0.5 - 0.2 = 0.3 \text{ m}$ . Applying Eq. 3–2, we have

$$F_{sp} = ks = 800(0.3) = 240 \text{ N}$$

**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad F_{BC} \cos 45^\circ + F_{BD} \left(\frac{4}{5}\right) - 240 = 0$$

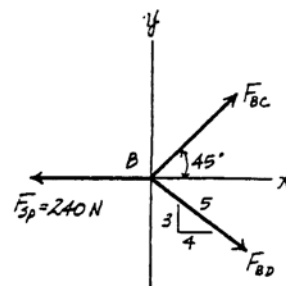
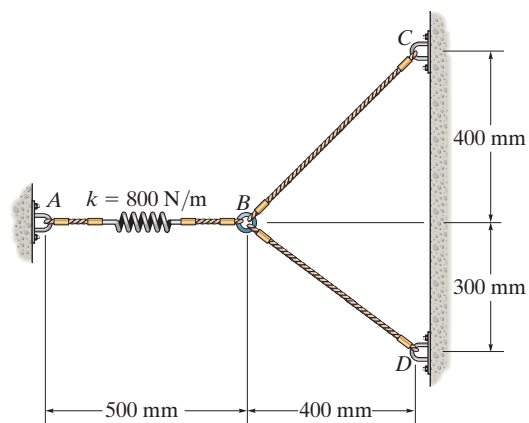
$$0.7071 F_{BC} + 0.8 F_{BD} = 240 \quad [1]$$

$$+ \uparrow \Sigma F_y = 0; \quad F_{BC} \sin 45^\circ - F_{BD} \left(\frac{3}{5}\right) = 0$$

$$F_{BC} = 0.8485 F_{BD} \quad [2]$$

Solving Eqs. [1] and [2] yields,

$$F_{BD} = 171 \text{ N} \quad F_{BC} = 145 \text{ N} \quad \text{Ans}$$



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•3–41. A continuous cable of total length 4 m is wrapped around the *small* pulleys at *A*, *B*, *C*, and *D*. If each spring is stretched 300 mm, determine the mass *m* of each block. Neglect the weight of the pulleys and cords. The springs are unstretched when *d* = 2 m.

$$F_s = kx; \quad F_s = 500(0.3) = 150 \text{ N}$$

At *A*:

$$+\uparrow \Sigma F_y = 0; \quad -150 + 2T \sin \theta = 0$$

$$T = \frac{75}{\sin \theta} \quad (1)$$

Note that when  $\theta = 90^\circ$ , the springs are unstretched and the tension in the cord is zero. When the springs are stretched 300 mm = 0.3 m, then  $d = (2 - 2(0.3)) = 1.4 \text{ m}$

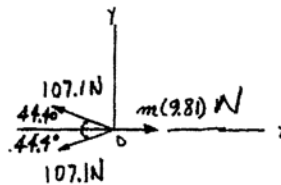
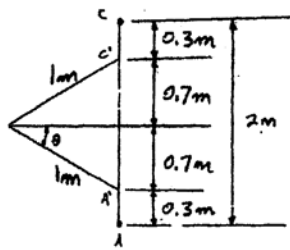
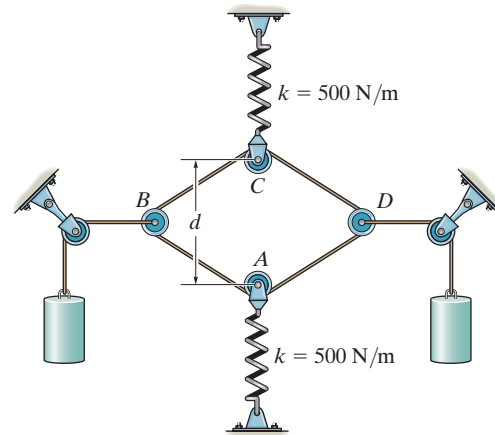
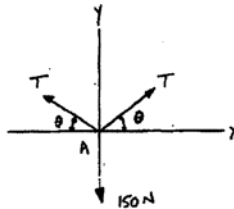
$$\theta = \sin^{-1}\left(\frac{0.7}{1}\right) = 44.4^\circ$$

From Eq. (1),  $T = 107.1 \text{ N}$

At *D*:

$$+\rightarrow \Sigma F_x = 0; \quad -2(107.1) \cos 44.4^\circ + m(9.81) = 0$$

$$m = 15.6 \text{ kg} \quad \text{Ans}$$

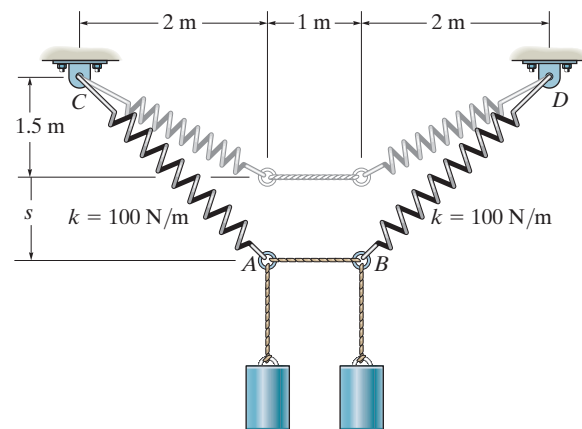
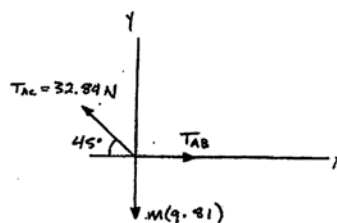
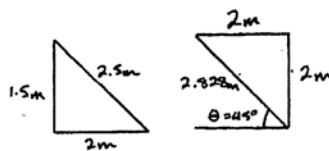


3–42. Determine the mass of each of the two cylinders if they cause a sag of  $s = 0.5 \text{ m}$  when suspended from the rings at *A* and *B*. Note that  $s = 0$  when the cylinders are removed.

$$T_{AC} = 100 \text{ N/m}(2.828 - 2.5) = 32.84 \text{ N}$$

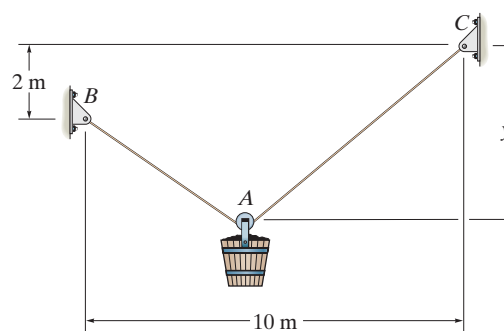
$$+\uparrow \Sigma F_y = 0; \quad 32.84 \sin 45^\circ - m(9.81) = 0$$

$$m = 2.37 \text{ kg} \quad \text{Ans}$$



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•3–43. The pail and its contents have a mass of 60 kg. If the cable  $BAC$  is 15 m long, determine the distance  $y$  to the pulley at  $A$  for equilibrium. Neglect the size of the pulley.



**Free Body Diagram :** Since the pulley is smooth, the tension in the cable is the same throughout the cable.

**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad T \sin \theta - T \sin \phi = 0 \quad \theta = \phi$$

**Geometry :**

$$l_1 = \sqrt{(10-x)^2 + (y-2)^2} \quad l_2 = \sqrt{x^2 + y^2}$$

Since  $\theta = \phi$ , two triangles are similar.

$$\frac{10-x}{x} = \frac{y-2}{y} = \frac{\sqrt{(10-x)^2 + (y-2)^2}}{\sqrt{x^2 + y^2}} \quad [1]$$

Also,

$$\begin{aligned} l_1 + l_2 &= 15 \\ \sqrt{(10-x)^2 + (y-2)^2} + \sqrt{x^2 + y^2} &= 15 \\ \left( \frac{\sqrt{x^2 + y^2}}{\sqrt{x^2 + y^2}} \right) \sqrt{(10-x)^2 + (y-2)^2} + \sqrt{x^2 + y^2} &= 15 \end{aligned} \quad [2]$$

However, from Eq. [1]  $\frac{\sqrt{(10-x)^2 + (y-2)^2}}{\sqrt{x^2 + y^2}} = \frac{10-x}{x}$ , Eq. [2] becomes

$$\sqrt{x^2 + y^2} \left( \frac{10-x}{x} \right) + \sqrt{x^2 + y^2} = 15 \quad [3]$$

Dividing both sides of Eq. [3] by  $\sqrt{x^2 + y^2}$  yields

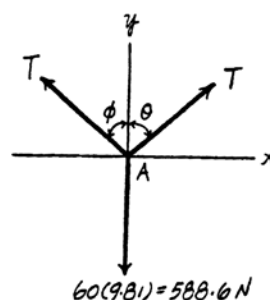
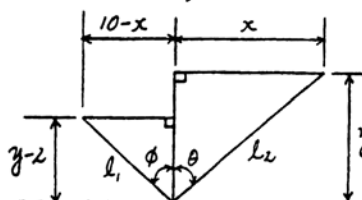
$$\frac{10}{x} = \frac{15}{\sqrt{x^2 + y^2}} \quad x = \sqrt{0.8}y \quad [4]$$

$$\text{From Eq. [1]} \quad \frac{10-x}{x} = \frac{y-2}{y} \quad x = \frac{5y}{y-1} \quad [5]$$

Equating Eq. [4] and [5] yields

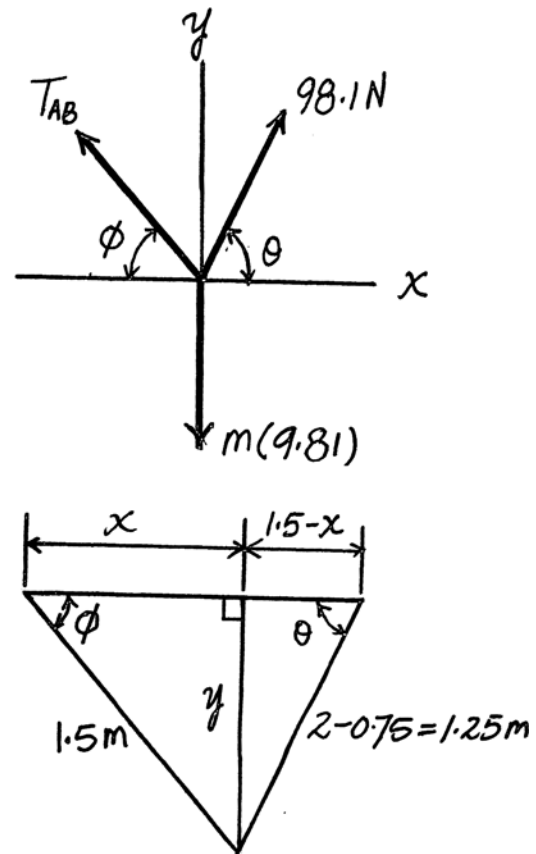
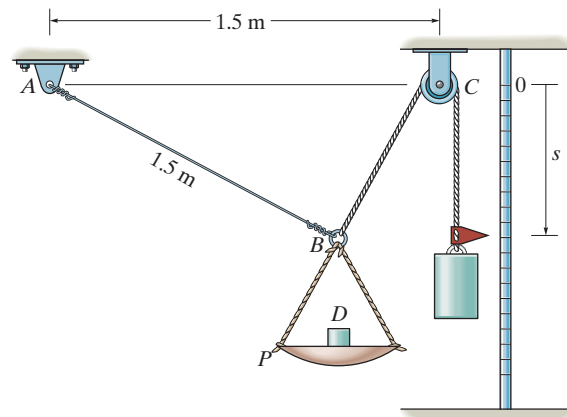
$$\begin{aligned} \sqrt{0.8}y &= \frac{5y}{y-1} \\ y &= 6.59 \text{ m} \end{aligned}$$

Ans



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•\*3-44. A scale is constructed using the 10-kg mass, the 2-kg pan  $P$ , and the pulley and cord arrangement. Cord  $BCA$  is 2 m long. If  $s = 0.75$  m, determine the mass  $D$  in the pan. Neglect the size of the pulley.



$$\rightarrow \Sigma F_x = 0; \quad 98.1 \cos \theta - T_{AB} \cos \phi = 0 \quad (1)$$

$$+ \uparrow \Sigma F_y = 0; \quad T_{AB} \sin \phi + 98.1 \sin \theta - m(9.81) = 0 \quad (2)$$

$$(1.5)^2 = x^2 + y^2$$

$$(1.25)^2 = (1.5 - x)^2 + y^2$$

$$(1.25)^2 = (1.5 - x)^2 + (1.5)^2 - x^2$$

$$-3x + 2.9375 = 0$$

$$x = 0.9792 \text{ m}$$

$$y = 1.1363 \text{ m}$$

Thus,

$$\phi = \sin^{-1} \left( \frac{1.1363}{1.5} \right) = 49.25^\circ$$

$$\theta = \sin^{-1} \left( \frac{1.1363}{1.25} \right) = 65.38^\circ$$

Solving Eq. (1) and (2),

$$T_{AB} = 62.62 \text{ N}$$

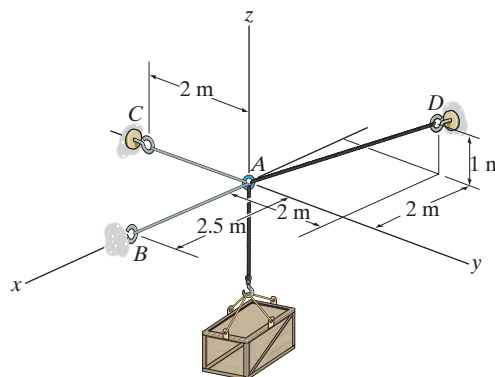
$$m = 13.9 \text{ kg}$$

Therefore,

$$m_D = 13.9 \text{ kg} - 2 \text{ kg} = 11.9 \text{ kg} \quad \text{Ans}$$

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- 3–45. Determine the tension in the cables in order to support the 100-kg crate in the equilibrium position shown.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \mathbf{i}$$

$$\mathbf{F}_{AC} = -F_{AC} \mathbf{j}$$

$$\mathbf{F}_{AD} = F_{AD} \left[ \frac{(-2-0)\mathbf{i} + (2-0)\mathbf{j} + (1-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (2-0)^2 + (1-0)^2}} \right] = -\frac{2}{3}F_{AD}\mathbf{i} + \frac{2}{3}F_{AD}\mathbf{j} + \frac{1}{3}F_{AD}\mathbf{k}$$

$$\mathbf{W} = [-100(9.81)\mathbf{k}] \text{ N} = [-981\mathbf{k}] \text{ N}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$F_{AB}\mathbf{i} + (-F_{AC}\mathbf{j}) + \left( -\frac{2}{3}F_{AD}\mathbf{i} + \frac{2}{3}F_{AD}\mathbf{j} + \frac{1}{3}F_{AD}\mathbf{k} \right) + (-981\mathbf{k}) = \mathbf{0}$$

$$\left( F_{AB} - \frac{2}{3}F_{AD} \right) \mathbf{i} + \left( -F_{AC} + \frac{2}{3}F_{AD} \right) \mathbf{j} + \left( \frac{1}{3}F_{AD} - 981 \right) \mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$F_{AB} - \frac{2}{3}F_{AD} = 0 \quad (1)$$

$$-F_{AC} + \frac{2}{3}F_{AD} = 0 \quad (2)$$

$$\frac{1}{3}F_{AD} - 981 = 0 \quad (3)$$

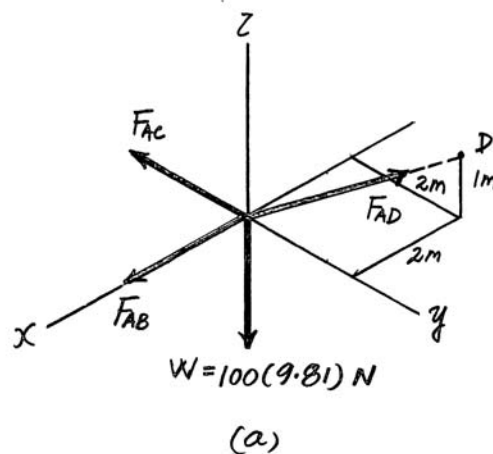
Solving Eqs. (1) through (3) yields

$$F_{AD} = 2943 \text{ N} = 2.94 \text{ kN}$$

Ans.

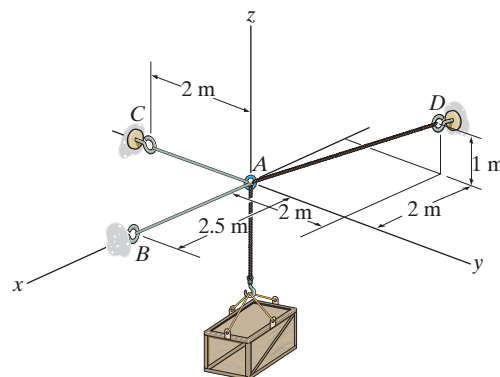
$$F_{AB} = F_{AC} = 1962 \text{ N} = 1.96 \text{ kN}$$

Ans.



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**3–46.** Determine the maximum mass of the crate so that the tension developed in any cable does not exceed 3 kN.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \mathbf{i}$$

$$\mathbf{F}_{AC} = -F_{AC} \mathbf{j}$$

$$\mathbf{F}_{AD} = F_{AD} \left[ \frac{(-2-0)\mathbf{i} + (2-0)\mathbf{j} + (1-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (2-0)^2 + (1-0)^2}} \right] = -\frac{2}{3}F_{AD}\mathbf{i} + \frac{2}{3}F_{AD}\mathbf{j} + \frac{1}{3}F_{AD}\mathbf{k}$$

$$\mathbf{W} = [-m(9.81)\mathbf{k}]$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$F_{AB} \mathbf{i} + (-F_{AC} \mathbf{j}) + \left( -\frac{2}{3}F_{AD} \mathbf{i} + \frac{2}{3}F_{AD} \mathbf{j} + \frac{1}{3}F_{AD} \mathbf{k} \right) + [-m(9.81)\mathbf{k}] = \mathbf{0}$$

$$\left( F_{AB} - \frac{2}{3}F_{AD} \right) \mathbf{i} + \left( -F_{AC} + \frac{2}{3}F_{AD} \right) \mathbf{j} + \left( \frac{1}{3}F_{AD} - 9.81m \right) \mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$F_{AB} - \frac{2}{3}F_{AD} = 0 \quad (1)$$

$$-F_{AC} + \frac{2}{3}F_{AD} = 0 \quad (2)$$

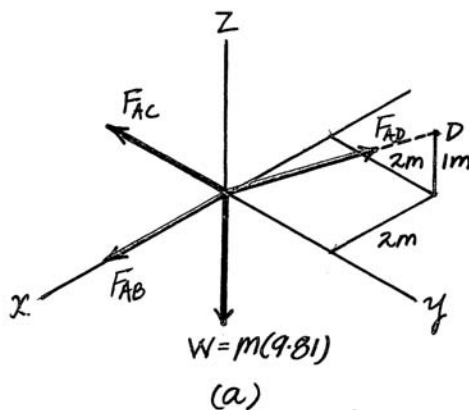
$$\frac{1}{3}F_{AD} - 9.81m = 0 \quad (3)$$

When cable  $AD$  is subjected to maximum tension,  $F_{AD} = 3000 \text{ N}$ . Thus, by substituting this value into Eqs. (1) through (3), we have

$$F_{AB} = F_{AC} = 2000 \text{ N}$$

$$m = 102 \text{ kg}$$

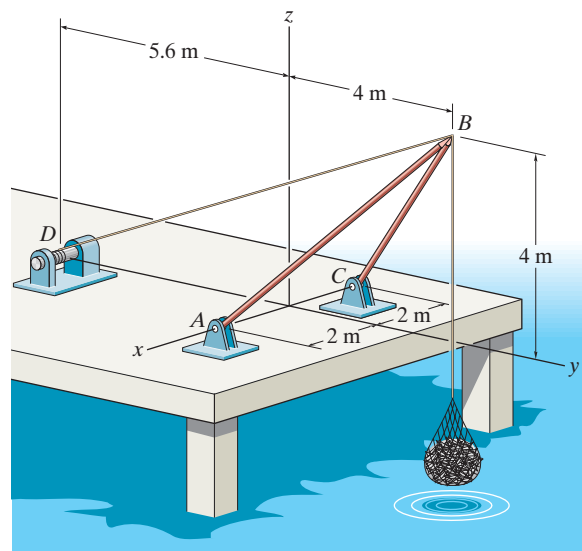
**Ans.**





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**3-47.** The shear leg derrick is used to haul the 200-kg net of fish onto the dock. Determine the compressive force along each of the legs  $AB$  and  $CB$  and the tension in the winch cable  $DB$ . Assume the force in each leg acts along its axis.



$$\begin{aligned} \mathbf{F}_{AB} &= F_{AB} \left( -\frac{2}{6}\mathbf{i} + \frac{4}{6}\mathbf{j} + \frac{4}{6}\mathbf{k} \right) \\ &= -0.3333 F_{AB} \mathbf{i} + 0.6667 F_{AB} \mathbf{j} + 0.6667 F_{AB} \mathbf{k} \end{aligned}$$

$$\begin{aligned} \mathbf{F}_{CB} &= F_{CB} \left( \frac{2}{6}\mathbf{i} + \frac{4}{6}\mathbf{j} + \frac{4}{6}\mathbf{k} \right) \\ &= 0.3333 F_{CB} \mathbf{i} + 0.6667 F_{CB} \mathbf{j} + 0.6667 F_{CB} \mathbf{k} \end{aligned}$$

$$\begin{aligned} \mathbf{F}_{BD} &= F_{BD} \left( -\frac{9.6}{10.4}\mathbf{j} - \frac{4}{10.4}\mathbf{k} \right) \\ &= -0.9231 F_{BD} \mathbf{j} - 0.3846 F_{BD} \mathbf{k} \end{aligned}$$

$$\mathbf{W} = -1962 \mathbf{k}$$

$$\Sigma F_x = 0; \quad -0.3333 F_{AB} + 0.3333 F_{CB} = 0$$

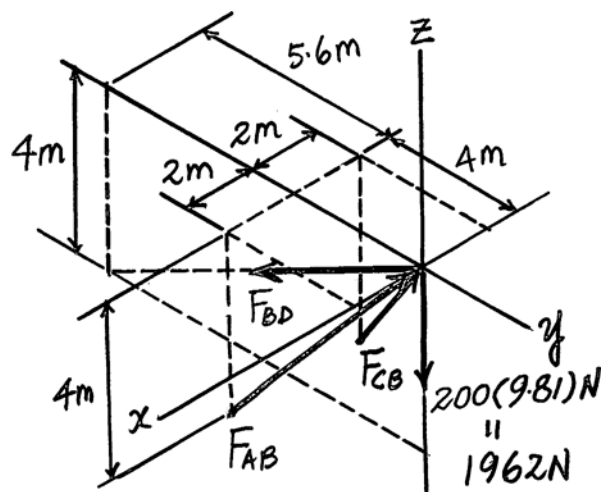
$$\Sigma F_y = 0; \quad 0.6667 F_{AB} + 0.6667 F_{CB} - 0.9231 F_{BD} = 0$$

$$\Sigma F_z = 0; \quad 0.6667 F_{AB} + 0.6667 F_{CB} - 0.3846 F_{BD} - 1962 = 0$$

$$F_{AB} = 2.52 \text{ kN} \quad \text{Ans}$$

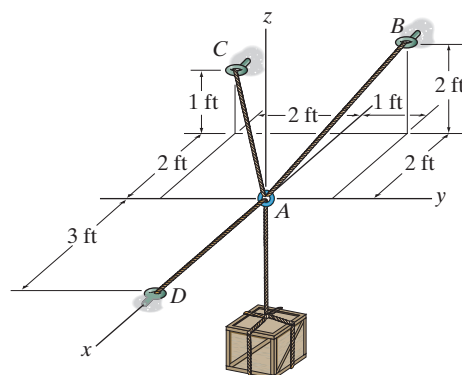
$$F_{CB} = 2.52 \text{ kN} \quad \text{Ans}$$

$$F_{BD} = 3.64 \text{ kN} \quad \text{Ans}$$



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\*3-48. Determine the tension developed in cables  $AB$ ,  $AC$ , and  $AD$  required for equilibrium of the 300-lb crate.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\begin{aligned} \mathbf{F}_{AB} &= F_{AB} \left[ \frac{(-2-0)\mathbf{i} + (1-0)\mathbf{j} + (2-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (1-0)^2 + (2-0)^2}} \right] = -\frac{2}{3}F_{AB}\mathbf{i} + \frac{1}{3}F_{AB}\mathbf{j} + \frac{2}{3}F_{AB}\mathbf{k} \\ \mathbf{F}_{AC} &= F_{AC} \left[ \frac{(-2-0)\mathbf{i} + (-2-0)\mathbf{j} + (1-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (-2-0)^2 + (1-0)^2}} \right] = -\frac{2}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{1}{3}F_{AC}\mathbf{k} \\ \mathbf{F}_{AD} &= F_{AD}\mathbf{i} \\ \mathbf{W} &= [-300\mathbf{k}] \text{ lb} \end{aligned}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$\left( -\frac{2}{3}F_{AB}\mathbf{i} + \frac{1}{3}F_{AB}\mathbf{j} + \frac{2}{3}F_{AB}\mathbf{k} \right) + \left( -\frac{2}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{1}{3}F_{AC}\mathbf{k} \right) + F_{AD}\mathbf{i} + (-300\mathbf{k}) = \mathbf{0}$$

$$\left( -\frac{2}{3}F_{AB} - \frac{2}{3}F_{AC} + F_{AD} \right)\mathbf{i} + \left( \frac{1}{3}F_{AB} - \frac{2}{3}F_{AC} \right)\mathbf{j} + \left( \frac{2}{3}F_{AB} + \frac{1}{3}F_{AC} - 300 \right)\mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$-\frac{2}{3}F_{AB} - \frac{2}{3}F_{AC} + F_{AD} = 0 \quad (1)$$

$$\frac{1}{3}F_{AB} - \frac{2}{3}F_{AC} = 0 \quad (2)$$

$$\frac{2}{3}F_{AB} + \frac{1}{3}F_{AC} - 300 = 0 \quad (3)$$

Solving Eqs. (1) through (3) yields

$$F_{AB} = 360 \text{ lb}$$

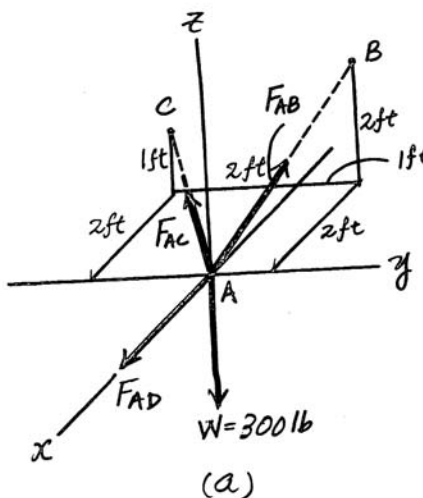
$$F_{AC} = 180 \text{ lb}$$

$$F_{AD} = 360 \text{ lb}$$

Ans.

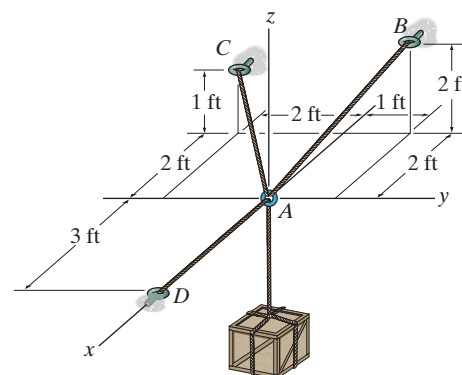
Ans.

Ans.



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- 3–49. Determine the maximum weight of the crate so that the tension developed in any cable does not exceed 450 lb.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \left[ \frac{(-2-0)\mathbf{i} + (1-0)\mathbf{j} + (2-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (1-0)^2 + (2-0)^2}} \right] = -\frac{2}{3}F_{AB}\mathbf{i} + \frac{1}{3}F_{AB}\mathbf{j} + \frac{2}{3}F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left[ \frac{(-2-0)\mathbf{i} + (-2-0)\mathbf{j} + (1-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (-2-0)^2 + (1-0)^2}} \right] = -\frac{2}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{1}{3}F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD}\mathbf{i}$$

$$\mathbf{W} = -W\mathbf{k}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$\left( -\frac{2}{3}F_{AB}\mathbf{i} + \frac{1}{3}F_{AB}\mathbf{j} + \frac{2}{3}F_{AB}\mathbf{k} \right) + \left( -\frac{2}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{1}{3}F_{AC}\mathbf{k} \right) + F_{AD}\mathbf{i} + (-W\mathbf{k}) = \mathbf{0}$$

$$\left( -\frac{2}{3}F_{AB} - \frac{2}{3}F_{AC} + F_{AD} \right)\mathbf{i} + \left( \frac{1}{3}F_{AB} - \frac{2}{3}F_{AC} \right)\mathbf{j} + \left( \frac{2}{3}F_{AB} + \frac{1}{3}F_{AC} - W \right)\mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$-\frac{2}{3}F_{AB} - \frac{2}{3}F_{AC} + F_{AD} = 0 \quad (1)$$

$$\frac{1}{3}F_{AB} - \frac{2}{3}F_{AC} = 0 \quad (2)$$

$$\frac{2}{3}F_{AB} + \frac{1}{3}F_{AC} - W = 0 \quad (3)$$

Let us assume that cable  $AB$  achieves maximum tension first. Substituting  $F_{AB} = 450$  lb into Eqs. (1) through (3) and solving, yields

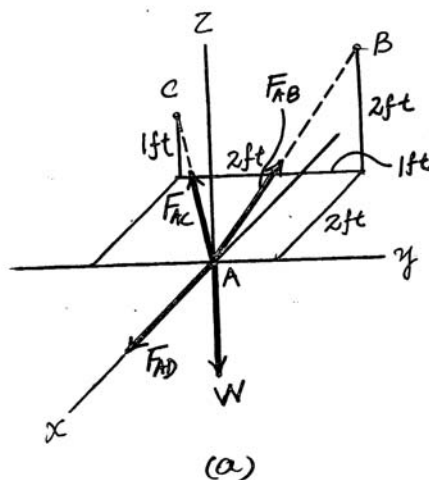
$$F_{AC} = 225 \text{ lb}$$

$$F_{AD} = 450 \text{ lb}$$

$$W = 375 \text{ lb}$$

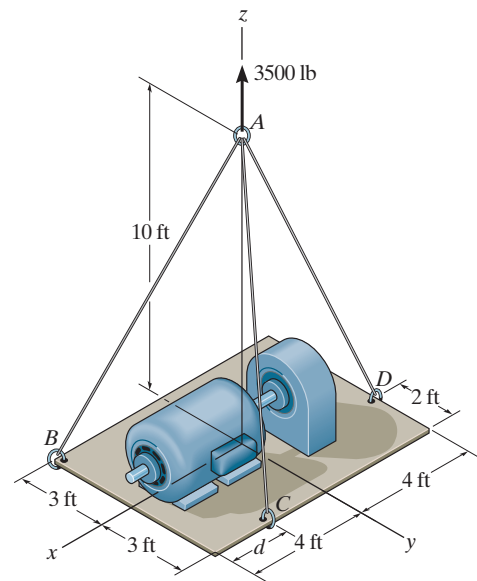
**Ans.**

Since  $F_{AC} = 225 \text{ lb} < 450 \text{ lb}$ , our assumption is correct.



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**3–50.** Determine the force in each cable needed to support the 3500-lb platform. Set  $d = 2$  ft.



**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{4\mathbf{i} - 3\mathbf{j} - 10\mathbf{k}}{\sqrt{4^2 + (-3)^2 + (-10)^2}} \right) = 0.3578F_{AB}\mathbf{i} - 0.2683F_{AB}\mathbf{j} - 0.8944F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{2\mathbf{i} + 3\mathbf{j} - 10\mathbf{k}}{\sqrt{2^2 + 3^2 + (-10)^2}} \right) = 0.1881F_{AC}\mathbf{i} + 0.2822F_{AC}\mathbf{j} - 0.9407F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left( \frac{-4\mathbf{i} + 1\mathbf{j} - 10\mathbf{k}}{\sqrt{(-4)^2 + 1^2 + (-10)^2}} \right) = -0.3698F_{AD}\mathbf{i} + 0.09245F_{AD}\mathbf{j} - 0.9245F_{AD}\mathbf{k}$$

$$\mathbf{F} = \{3500\mathbf{k}\} \text{ lb}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = \mathbf{0}$$

$$(0.3578F_{AB} + 0.1881F_{AC} - 0.3698F_{AD})\mathbf{i} + (-0.2683F_{AB} + 0.2822F_{AC} + 0.09245F_{AD})\mathbf{j} + (-0.8944F_{AB} - 0.9407F_{AC} - 0.9245F_{AD} + 3500)\mathbf{k} = \mathbf{0}$$

Equating  $\mathbf{i}$ ,  $\mathbf{j}$  and  $\mathbf{k}$  components, we have

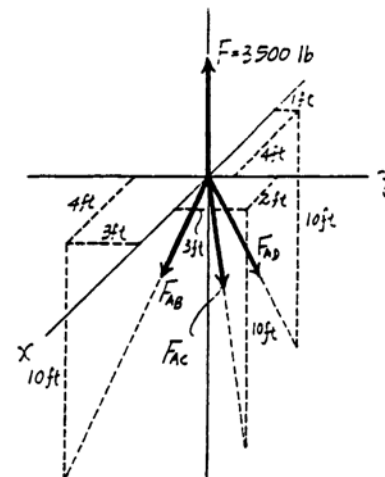
$$0.3578F_{AB} + 0.1881F_{AC} - 0.3698F_{AD} = 0 \quad [1]$$

$$-0.2683F_{AB} + 0.2822F_{AC} + 0.09245F_{AD} = 0 \quad [2]$$

$$-0.8944F_{AB} - 0.9407F_{AC} - 0.9245F_{AD} + 3500 = 0 \quad [3]$$

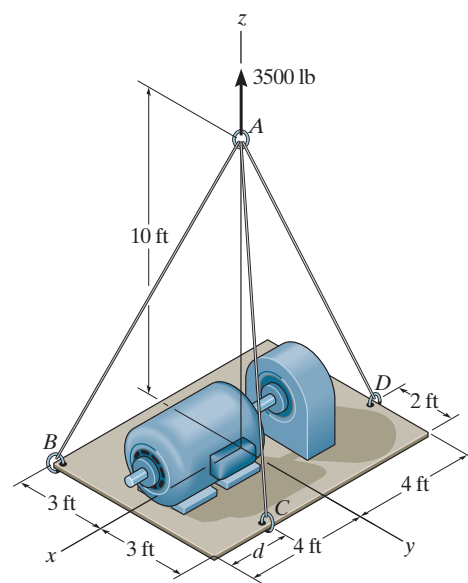
Solving Eqs. [1], [2] and [3] yields

$$\begin{aligned} F_{AB} &= 1369.59 \text{ lb} = 1.37 \text{ kip} & F_{AC} &= 744.11 \text{ lb} = 0.744 \text{ kip} & \text{Ans} \\ F_{AD} &= 1703.62 \text{ lb} = 1.70 \text{ kip} & & & \text{Ans} \end{aligned}$$



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**3–51.** Determine the force in each cable needed to support the 3500-lb platform. Set  $d = 4$  ft.



**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{4\mathbf{i} - 3\mathbf{j} - 10\mathbf{k}}{\sqrt{4^2 + (-3)^2 + (-10)^2}} \right) = 0.3578F_{AB}\mathbf{i} - 0.2683F_{AB}\mathbf{j} - 0.8944F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{3\mathbf{j} - 10\mathbf{k}}{\sqrt{3^2 + (-10)^2}} \right) = 0.2873F_{AC}\mathbf{j} - 0.9578F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left( \frac{-4\mathbf{i} + 1\mathbf{j} - 10\mathbf{k}}{\sqrt{(-4)^2 + 1^2 + (-10)^2}} \right) = -0.3698F_{AD}\mathbf{i} + 0.09245F_{AD}\mathbf{j} - 0.9245F_{AD}\mathbf{k}$$

$$\mathbf{F} = \{3500\mathbf{k}\} \text{ lb}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = 0; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = 0$$

$$(0.3578F_{AB} - 0.3698F_{AD})\mathbf{i} + (-0.2683F_{AB} + 0.2873F_{AC} + 0.09245F_{AD})\mathbf{j} + (-0.8944F_{AB} - 0.9578F_{AC} - 0.9245F_{AD} + 3500)\mathbf{k} = 0$$

Equating i, j and k components, we have

$$0.3578F_{AB} - 0.3698F_{AD} = 0 \quad [1]$$

$$-0.2683F_{AB} + 0.2873F_{AC} + 0.09245F_{AD} = 0 \quad [2]$$

$$-0.8944F_{AB} - 0.9578F_{AC} - 0.9245F_{AD} + 3500 = 0 \quad [3]$$

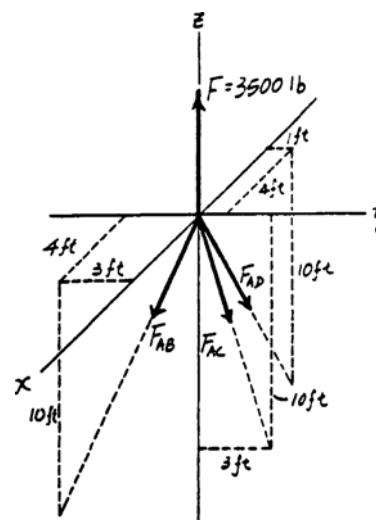
Solving Eqs. [1], [2] and [3] yields

$$F_{AB} = 1467.42 \text{ lb} = 1.47 \text{ kip}$$

$$F_{AD} = 1419.69 \text{ lb} = 1.42 \text{ kip}$$

$$F_{AC} = 913.53 \text{ lb} = 0.914 \text{ kip}$$

Ans  
Ans



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\*3-52. Determine the force in each of the three cables needed to lift the tractor which has a mass of 8 Mg.

**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{2\mathbf{i} - 1.25\mathbf{j} - 3\mathbf{k}}{\sqrt{2^2 + (-1.25)^2 + (-3)^2}} \right) = 0.5241F_{AB}\mathbf{i} - 0.3276F_{AB}\mathbf{j} - 0.7861F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{2\mathbf{i} + 1.25\mathbf{j} - 3\mathbf{k}}{\sqrt{2^2 + 1.25^2 + (-3)^2}} \right) = 0.5241F_{AC}\mathbf{i} + 0.3276F_{AC}\mathbf{j} - 0.7861F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left( \frac{-1\mathbf{i} - 3\mathbf{k}}{\sqrt{(-1)^2 + (-3)^2}} \right) = -0.3162F_{AD}\mathbf{i} - 0.9487F_{AD}\mathbf{k}$$

$$\mathbf{F} = \{78.48\mathbf{k}\} \text{ kN}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = \mathbf{0}$$

$$(0.5241F_{AB} + 0.5241F_{AC} - 0.3162F_{AD})\mathbf{i} + (-0.3276F_{AB} + 0.3276F_{AC})\mathbf{j} + (-0.7861F_{AB} - 0.7861F_{AC} - 0.9487F_{AD} + 78.48)\mathbf{k} = \mathbf{0}$$

Equating i, j and k components, we have

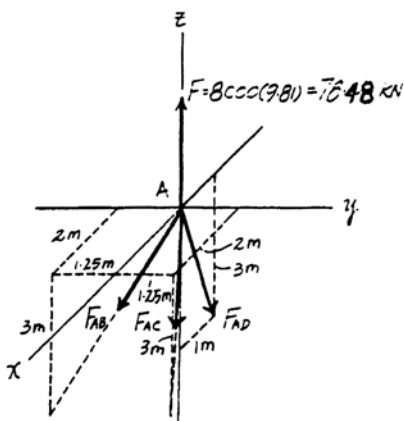
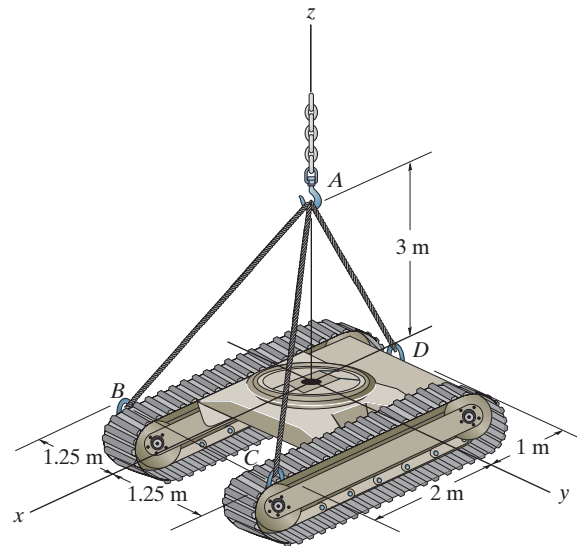
$$0.5241F_{AB} + 0.5241F_{AC} - 0.3162F_{AD} = 0 \quad [1]$$

$$-0.3276F_{AB} + 0.3276F_{AC} = 0 \quad [2]$$

$$-0.7861F_{AB} - 0.7861F_{AC} - 0.9487F_{AD} + 78.48 = 0 \quad [3]$$

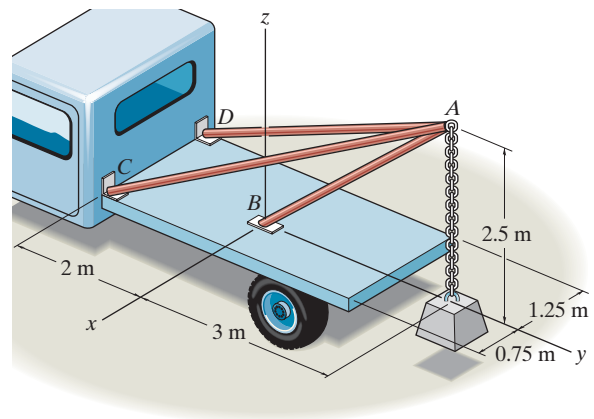
Solving Eqs. [1], [2] and [3] yields

$$F_{AB} = F_{AC} = 16.6 \text{ kN} \quad F_{AD} = 55.2 \text{ kN} \quad \text{Ans}$$



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•3–53. Determine the force acting along the axis of each of the three struts needed to support the 500-kg block.



$$\mathbf{F}_B = F_B \left( \frac{3\mathbf{j} + 2.5\mathbf{k}}{3.905} \right)$$

$$= 0.7682 F_B \mathbf{j} + 0.6402 F_B \mathbf{k}$$

$$\mathbf{F}_C = F_C \left( \frac{0.75\mathbf{i} - 5\mathbf{j} - 2.5\mathbf{k}}{5.640} \right)$$

$$= 0.1330 F_C \mathbf{i} - 0.8865 F_C \mathbf{j} - 0.4432 F_C \mathbf{k}$$

$$\mathbf{F}_D = F_D \left( \frac{-1.25\mathbf{i} - 5\mathbf{j} - 2.5\mathbf{k}}{5.728} \right)$$

$$= -0.2182 F_D \mathbf{i} - 0.8729 F_D \mathbf{j} - 0.4364 F_D \mathbf{k}$$

$$\mathbf{W} = -500(9.81) \mathbf{k} = -4905 \mathbf{k}$$

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_B + \mathbf{F}_C + \mathbf{F}_D + \mathbf{W} = \mathbf{0}$$

$$\Sigma F_x = 0; \quad 0.1330 F_C - 0.2182 F_D = 0$$

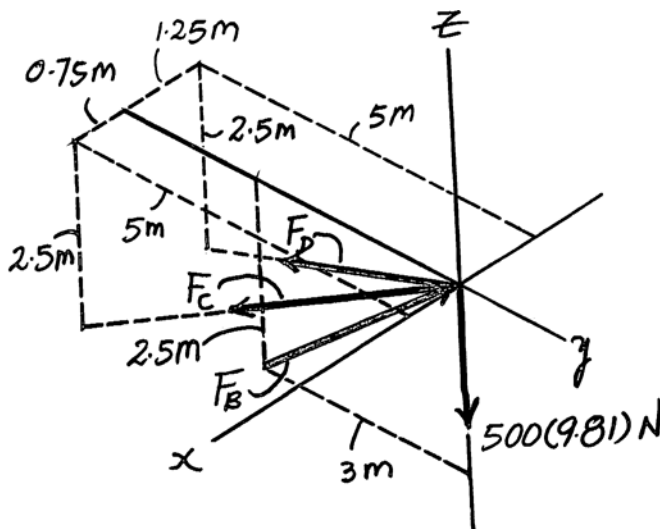
$$\Sigma F_y = 0; \quad 0.7682 F_B - 0.8865 F_C - 0.8729 F_D = 0$$

$$\Sigma F_z = 0; \quad 0.6402 F_B - 0.4432 F_C - 0.4364 F_D - 4905 = 0$$

$$F_B = 19.2 \text{ kN} \quad \text{Ans}$$

$$F_C = 10.4 \text{ kN} \quad \text{Ans}$$

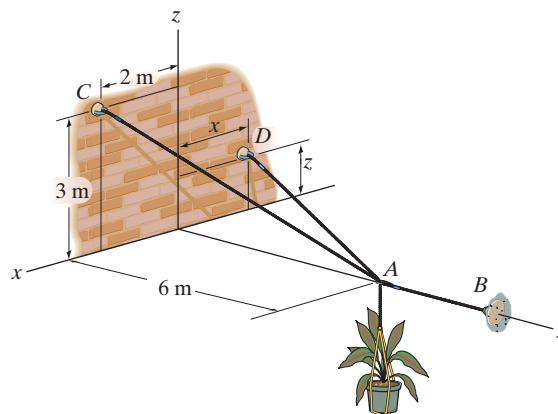
$$F_D = 6.32 \text{ kN} \quad \text{Ans}$$





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**3–54.** If the mass of the flowerpot is 50 kg, determine the tension developed in each wire for equilibrium. Set  $x = 1.5$  m and  $z = 2$  m.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \mathbf{j}$$

$$\mathbf{F}_{AC} = F_{AC} \left[ \frac{(2-0)\mathbf{i} + (-6-0)\mathbf{j} + (3-0)\mathbf{k}}{\sqrt{(2-0)^2 + (-6-0)^2 + (3-0)^2}} \right] = \frac{2}{7} F_{AC} \mathbf{i} - \frac{6}{7} F_{AC} \mathbf{j} + \frac{3}{7} F_{AC} \mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left[ \frac{(-1.5-0)\mathbf{i} + (-6-0)\mathbf{j} + (2-0)\mathbf{k}}{\sqrt{(-1.5-0)^2 + (-6-0)^2 + (2-0)^2}} \right] = -\frac{3}{13} F_{AD} \mathbf{i} - \frac{12}{13} F_{AD} \mathbf{j} + \frac{4}{13} F_{AD} \mathbf{k}$$

$$\mathbf{W} = [-50(9.81)\mathbf{k}] \text{ N} = [-490.5\mathbf{k}] \text{ N}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$F_{AB} \mathbf{j} + \left( \frac{2}{7} F_{AC} \mathbf{i} - \frac{6}{7} F_{AC} \mathbf{j} + \frac{3}{7} F_{AC} \mathbf{k} \right) + \left( -\frac{3}{13} F_{AD} \mathbf{i} - \frac{12}{13} F_{AD} \mathbf{j} + \frac{4}{13} F_{AD} \mathbf{k} \right) + (-490.5\mathbf{k}) = \mathbf{0}$$

$$\left( \frac{2}{7} F_{AC} - \frac{3}{13} F_{AD} \right) \mathbf{i} + \left( F_{AB} - \frac{6}{7} F_{AC} - \frac{12}{13} F_{AD} \right) \mathbf{j} + \left( \frac{3}{7} F_{AC} + \frac{4}{13} F_{AD} - 490.5 \right) \mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$\frac{2}{7} F_{AC} - \frac{3}{13} F_{AD} = 0 \quad (1)$$

$$F_{AB} - \frac{6}{7} F_{AC} - \frac{12}{13} F_{AD} = 0 \quad (2)$$

$$\frac{3}{7} F_{AC} + \frac{4}{13} F_{AD} - 490.5 = 0 \quad (3)$$

Solving Eqs. (1) through (3) yields

$$F_{AB} = 1211.82 \text{ N} = 1.21 \text{ kN}$$

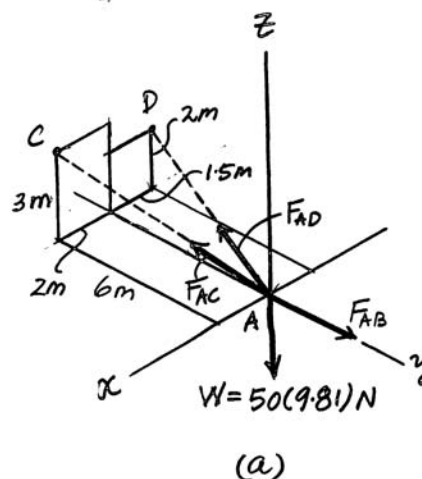
$$F_{AC} = 606 \text{ N}$$

$$F_{AD} = 750 \text{ N}$$

Ans.

Ans.

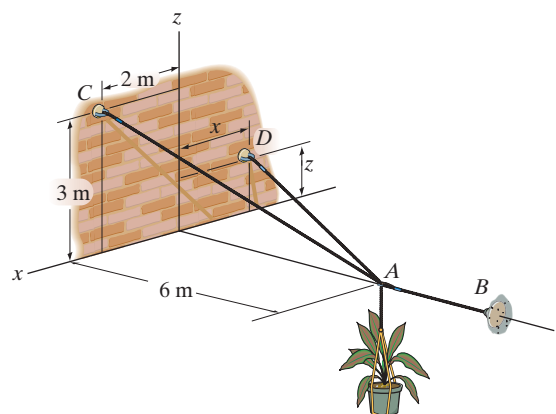
Ans.





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**3–55.** If the mass of the flowerpot is 50 kg, determine the tension developed in each wire for equilibrium. Set  $x = 2$  m and  $z = 1.5$  m.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \mathbf{j}$$

$$\mathbf{F}_{AC} = F_{AC} \left[ \frac{(2-0)\mathbf{i} + (-6-0)\mathbf{j} + (3-0)\mathbf{k}}{\sqrt{(2-0)^2 + (-6-0)^2 + (3-0)^2}} \right] = \frac{2}{7}F_{AC}\mathbf{i} - \frac{6}{7}F_{AC}\mathbf{j} + \frac{3}{7}F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left[ \frac{(-2-0)\mathbf{i} + (-6-0)\mathbf{j} + (1.5-0)\mathbf{k}}{\sqrt{(-2-0)^2 + (-6-0)^2 + (1.5-0)^2}} \right] = -\frac{4}{13}F_{AD}\mathbf{i} - \frac{12}{13}F_{AD}\mathbf{j} + \frac{3}{13}F_{AD}\mathbf{k}$$

$$\mathbf{W} = [-50(9.81)\mathbf{k}] \text{ N} = [-490.5\mathbf{k}] \text{ N}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$F_{AB} \mathbf{j} + \left( \frac{2}{7}F_{AC}\mathbf{i} - \frac{6}{7}F_{AC}\mathbf{j} + \frac{3}{7}F_{AC}\mathbf{k} \right) + \left( -\frac{4}{13}F_{AD}\mathbf{i} - \frac{12}{13}F_{AD}\mathbf{j} + \frac{3}{13}F_{AD}\mathbf{k} \right) + (-490.5\mathbf{k}) = \mathbf{0}$$

$$\left( \frac{2}{7}F_{AC} - \frac{4}{13}F_{AD} \right) \mathbf{i} + \left( F_{AB} - \frac{6}{7}F_{AC} - \frac{12}{13}F_{AD} \right) \mathbf{j} + \left( \frac{3}{7}F_{AC} + \frac{3}{13}F_{AD} - 490.5 \right) \mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$\frac{2}{7}F_{AC} - \frac{4}{13}F_{AD} = 0 \quad (1)$$

$$F_{AB} - \frac{6}{7}F_{AC} - \frac{12}{13}F_{AD} = 0 \quad (2)$$

$$\frac{3}{7}F_{AC} + \frac{3}{13}F_{AD} - 490.5 = 0 \quad (3)$$

Solving Eqs. (1) through (3) yields

$$F_{AB} = 1308 \text{ N} = 1.31 \text{ kN}$$

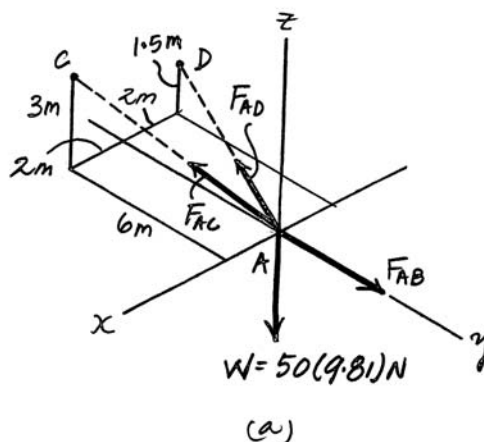
$$F_{AC} = 763 \text{ N}$$

$$F_{AD} = 708.5 \text{ N}$$

Ans.

Ans.

Ans.



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\*3-56. The ends of the three cables are attached to a ring at  $A$  and to the edge of a uniform 150-kg plate. Determine the tension in each of the cables for equilibrium.

$$\mathbf{P} = 150(9.81)\mathbf{k} = 1471.5\mathbf{k}$$

$$\mathbf{F}_B = \frac{4}{14}F_B\mathbf{i} - \frac{6}{14}F_B\mathbf{j} - \frac{12}{14}F_B\mathbf{k}$$

$$\mathbf{F}_C = -\frac{6}{14}F_C\mathbf{i} - \frac{4}{14}F_C\mathbf{j} - \frac{12}{14}F_C\mathbf{k}$$

$$\mathbf{F}_D = -\frac{4}{14}F_D\mathbf{i} + \frac{6}{14}F_D\mathbf{j} - \frac{12}{14}F_D\mathbf{k}$$

$$\Sigma F_x = 0; \quad \frac{4}{14}F_B - \frac{6}{14}F_C - \frac{4}{14}F_D = 0$$

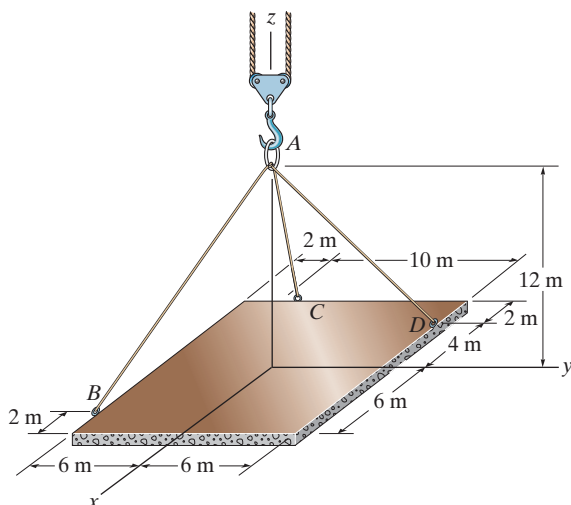
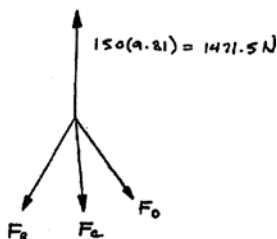
$$\Sigma F_y = 0; \quad -\frac{6}{14}F_B - \frac{4}{14}F_C + \frac{6}{14}F_D = 0$$

$$\Sigma F_z = 0; \quad -\frac{12}{14}F_B - \frac{12}{14}F_C - \frac{12}{14}F_D + 1471.5 = 0$$

$$F_B = 858\text{ N} \quad \text{Ans}$$

$$F_C = 0 \quad \text{Ans}$$

$$F_D = 858\text{ N} \quad \text{Ans}$$



•3-57. The ends of the three cables are attached to a ring at  $A$  and to the edge of the uniform plate. Determine the largest mass the plate can have if each cable can support a maximum tension of 15 kN.

$$\mathbf{W} = W\mathbf{k}$$

$$\mathbf{F}_B = F_B \left( \frac{4}{14}\mathbf{i} - \frac{6}{14}\mathbf{j} - \frac{12}{14}\mathbf{k} \right)$$

$$\mathbf{F}_C = F_C \left( -\frac{6}{14}\mathbf{i} - \frac{4}{14}\mathbf{j} - \frac{12}{14}\mathbf{k} \right)$$

$$\mathbf{F}_D = F_D \left( -\frac{4}{14}\mathbf{i} + \frac{6}{14}\mathbf{j} - \frac{12}{14}\mathbf{k} \right)$$

$$\Sigma F_x = 0; \quad \frac{4}{14}F_B - \frac{6}{14}F_C - \frac{4}{14}F_D = 0$$

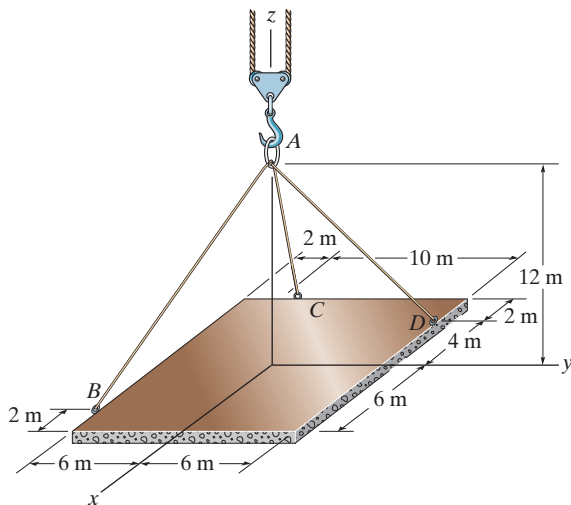
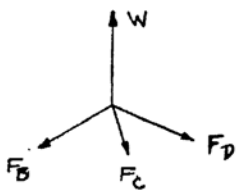
$$\Sigma F_y = 0; \quad -\frac{6}{14}F_B - \frac{4}{14}F_C + \frac{6}{14}F_D = 0$$

$$\Sigma F_z = 0; \quad -\frac{12}{14}F_B - \frac{12}{14}F_C - \frac{12}{14}F_D + W = 0$$

Assume  $F_B = 15\text{ kN}$ . Solving,

$$F_C = 0 < 15\text{ kN} \quad (\text{OK})$$

$$F_D = 15\text{ kN} \quad (\text{OK})$$



Thus,

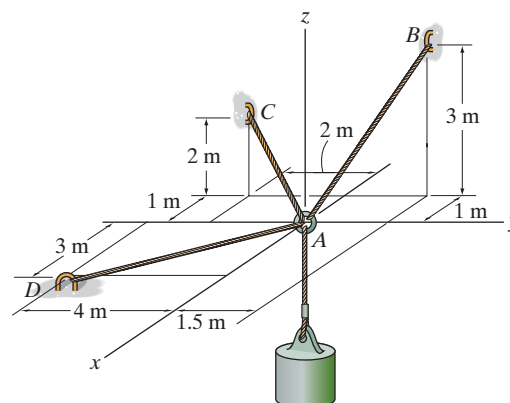
$$-\frac{12}{14}(15) - 0 - \frac{12}{14}(15) + W = 0$$

$$W = 25.714\text{ kN}$$

$$m = \frac{W}{g} = \frac{25.714}{9.81} = 2.62\text{ Mg} \quad \text{Ans}$$

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**3–58.** Determine the tension developed in cables  $AB$ ,  $AC$ , and  $AD$  required for equilibrium of the 75-kg cylinder.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \frac{(-1-0)\mathbf{i} + (1.5-0)\mathbf{j} + (3-0)\mathbf{k}}{\sqrt{(-1-0)^2 + (1.5-0)^2 + (3-0)^2}} = -\frac{2}{7}F_{AB}\mathbf{i} + \frac{3}{7}F_{AB}\mathbf{j} + \frac{6}{7}F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \frac{(-1-0)\mathbf{i} + (-2-0)\mathbf{j} + (2-0)\mathbf{k}}{\sqrt{(-1-0)^2 + (-2-0)^2 + (2-0)^2}} = -\frac{1}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{2}{3}F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \frac{(3-0)\mathbf{i} + (-4-0)\mathbf{j} + (0-0)\mathbf{k}}{\sqrt{(3-0)^2 + (-4-0)^2 + (0-0)^2}} = \frac{3}{5}F_{AD}\mathbf{i} - \frac{4}{5}F_{AD}\mathbf{j}$$

$$\mathbf{W} = [-75(9.81)\mathbf{k}]\text{N} = [-735.75\mathbf{k}]\text{N}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$\left(-\frac{2}{7}F_{AB}\mathbf{i} + \frac{3}{7}F_{AB}\mathbf{j} + \frac{6}{7}F_{AB}\mathbf{k}\right) + \left(-\frac{1}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{2}{3}F_{AC}\mathbf{k}\right) + \left(\frac{3}{5}F_{AD}\mathbf{i} - \frac{4}{5}F_{AD}\mathbf{j}\right) + (-735.75\mathbf{k}) = \mathbf{0}$$

$$\left(-\frac{2}{7}F_{AB} - \frac{1}{3}F_{AC} + \frac{3}{5}F_{AD}\right)\mathbf{i} + \left(\frac{3}{7}F_{AB} - \frac{2}{3}F_{AC} - \frac{4}{5}F_{AD}\right)\mathbf{j} + \left(\frac{6}{7}F_{AB} + \frac{2}{3}F_{AC} - 735.75\right)\mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$-\frac{2}{7}F_{AB} - \frac{1}{3}F_{AC} + \frac{3}{5}F_{AD} = 0 \quad (1)$$

$$\frac{3}{7}F_{AB} - \frac{2}{3}F_{AC} - \frac{4}{5}F_{AD} = 0 \quad (2)$$

$$\frac{6}{7}F_{AB} + \frac{2}{3}F_{AC} - 735.75 = 0 \quad (3)$$

Solving Eqs. (1) through (3) yields

$$F_{AB} = 831\text{ N}$$

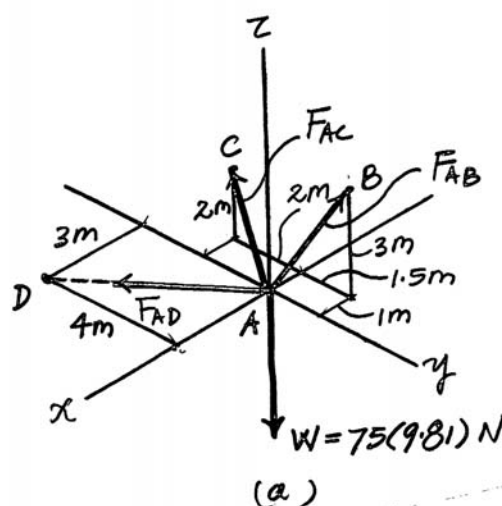
Ans.

$$F_{AC} = 35.6\text{ N}$$

Ans.

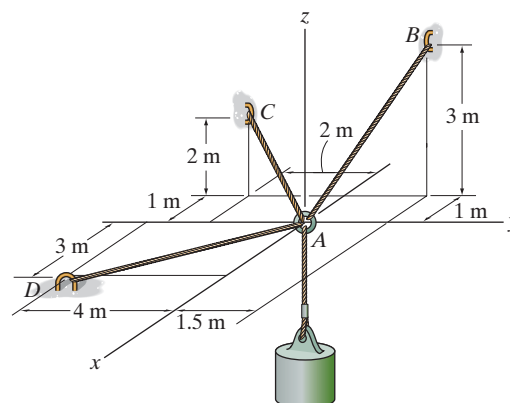
$$F_{AD} = 415\text{ N}$$

Ans.



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**3–59.** If each cable can withstand a maximum tension of 1000 N, determine the largest mass of the cylinder for equilibrium.



**Force Vectors:** We can express each of the forces on the free-body diagram shown in Fig. (a) in Cartesian vector form as

$$\mathbf{F}_{AB} = F_{AB} \left[ \frac{(-1-0)\mathbf{i} + (1.5-0)\mathbf{j} + (3-0)\mathbf{k}}{\sqrt{(-1-0)^2 + (1.5-0)^2 + (3-0)^2}} \right] = -\frac{2}{7}F_{AB}\mathbf{i} + \frac{3}{7}F_{AB}\mathbf{j} + \frac{6}{7}F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left[ \frac{(-1-0)\mathbf{i} + (-2-0)\mathbf{j} + (2-0)\mathbf{k}}{\sqrt{(-1-0)^2 + (-2-0)^2 + (2-0)^2}} \right] = -\frac{1}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{2}{3}F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left[ \frac{(3-0)\mathbf{i} + (-4-0)\mathbf{j} + (0-0)\mathbf{k}}{\sqrt{(3-0)^2 + (-4-0)^2 + (0-0)^2}} \right] = \frac{3}{5}F_{AD}\mathbf{i} - \frac{4}{5}F_{AD}\mathbf{j}$$

$$\mathbf{W} = -m(9.81)\mathbf{k}$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{W} = \mathbf{0}$$

$$\left( -\frac{2}{7}F_{AB}\mathbf{i} + \frac{3}{7}F_{AB}\mathbf{j} + \frac{6}{7}F_{AB}\mathbf{k} \right) + \left( -\frac{1}{3}F_{AC}\mathbf{i} - \frac{2}{3}F_{AC}\mathbf{j} + \frac{2}{3}F_{AC}\mathbf{k} \right) + \left( \frac{3}{5}F_{AD}\mathbf{i} - \frac{4}{5}F_{AD}\mathbf{j} \right) + [-m(9.81)\mathbf{k}] = \mathbf{0}$$

$$\left( -\frac{2}{7}F_{AB} - \frac{1}{3}F_{AC} + \frac{3}{5}F_{AD} \right)\mathbf{i} + \left( \frac{3}{7}F_{AB} - \frac{2}{3}F_{AC} - \frac{4}{5}F_{AD} \right)\mathbf{j} + \left( \frac{6}{7}F_{AB} + \frac{2}{3}F_{AC} - m(9.81) \right)\mathbf{k} = \mathbf{0}$$

Equating the  $\mathbf{i}$ ,  $\mathbf{j}$ , and  $\mathbf{k}$  components yields

$$-\frac{2}{7}F_{AB} - \frac{1}{3}F_{AC} + \frac{3}{5}F_{AD} = 0 \quad (1)$$

$$\frac{3}{7}F_{AB} - \frac{2}{3}F_{AC} - \frac{4}{5}F_{AD} = 0 \quad (2)$$

$$\frac{6}{7}F_{AB} + \frac{2}{3}F_{AC} - m(9.81) = 0 \quad (3)$$

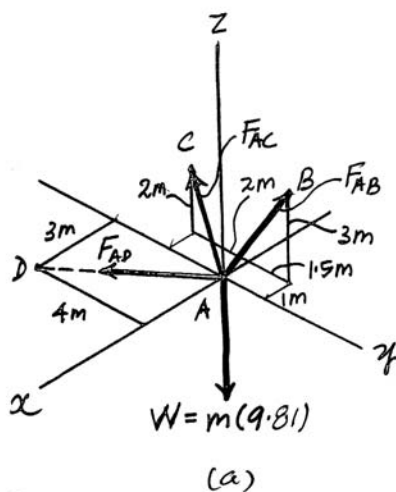
Let us assume that cable  $AB$  achieves maximum tension first. Substituting  $F_{AB} = 1000$  N into Eqs. (1) through (3) and solving, yields

$$F_{AD} = 500 \text{ N}$$

$$F_{AC} = 42.86 \text{ N}$$

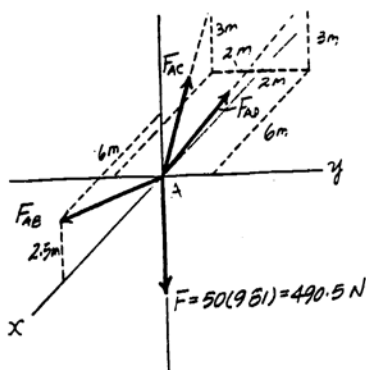
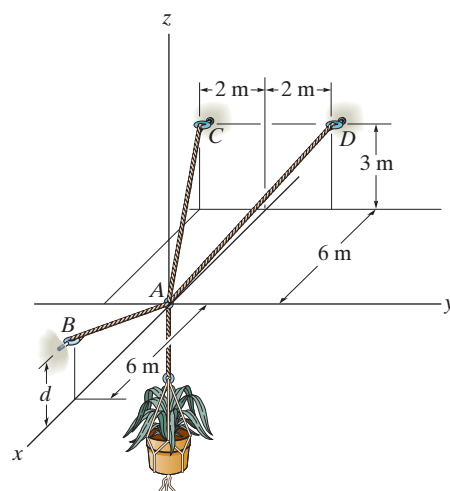
$$m = 90.3 \text{ kg}$$

**Ans.**



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\*3–60. The 50-kg pot is supported from  $A$  by the three cables. Determine the force acting in each cable for equilibrium. Take  $d = 2.5$  m.



**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{6\mathbf{i} + 2.5\mathbf{k}}{\sqrt{6^2 + 2.5^2}} \right) = \frac{12}{13} F_{AB} \mathbf{i} + \frac{5}{13} F_{AB} \mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{-6\mathbf{i} - 2\mathbf{j} + 3\mathbf{k}}{\sqrt{(-6)^2 + (-2)^2 + 3^2}} \right) = -\frac{6}{7} F_{AC} \mathbf{i} - \frac{2}{7} F_{AC} \mathbf{j} + \frac{3}{7} F_{AC} \mathbf{k}$$

$$\mathbf{F}_{AD} = F_{AD} \left( \frac{-6\mathbf{i} + 2\mathbf{j} + 3\mathbf{k}}{\sqrt{(-6)^2 + 2^2 + 3^2}} \right) = -\frac{6}{7} F_{AD} \mathbf{i} + \frac{2}{7} F_{AD} \mathbf{j} + \frac{3}{7} F_{AD} \mathbf{k}$$

$$\mathbf{F} = \{-490.5\mathbf{k}\} \text{ N}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = \mathbf{0}$$

$$\left( \frac{12}{13} F_{AB} - \frac{6}{7} F_{AC} - \frac{6}{7} F_{AD} \right) \mathbf{i} + \left( -\frac{2}{7} F_{AC} + \frac{2}{7} F_{AD} \right) \mathbf{j} + \left( \frac{5}{13} F_{AB} + \frac{3}{7} F_{AC} + \frac{3}{7} F_{AD} - 490.5 \right) \mathbf{k} = \mathbf{0}$$

Equating  $\mathbf{i}$ ,  $\mathbf{j}$  and  $\mathbf{k}$  components, we have

$$\frac{12}{13} F_{AB} - \frac{6}{7} F_{AC} - \frac{6}{7} F_{AD} = 0 \quad [1]$$

$$-\frac{2}{7} F_{AC} + \frac{2}{7} F_{AD} = 0 \quad [2]$$

$$\frac{5}{13} F_{AB} + \frac{3}{7} F_{AC} + \frac{3}{7} F_{AD} - 490.5 = 0 \quad [3]$$

Solving Eqs. [1], [2] and [3] yields

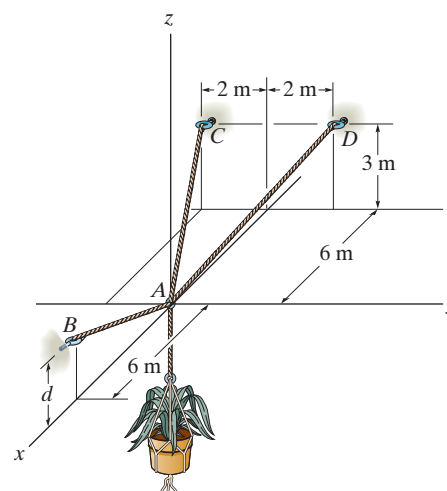
$$F_{AC} = F_{AD} = 312 \text{ N}$$

$$F_{AB} = 580 \text{ N}$$

**Ans**

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•3–61. Determine the height  $d$  of cable  $AB$  so that the force in cables  $AD$  and  $AC$  is one-half as great as the force in cable  $AB$ . What is the force in each cable for this case? The flower pot has a mass of 50 kg.



**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = (F_{AB})_x \mathbf{i} + (F_{AB})_z \mathbf{k}$$

$$\mathbf{F}_{AC} = \frac{F_{AB}}{2} \left( \frac{-6\mathbf{i} - 2\mathbf{j} + 3\mathbf{k}}{\sqrt{(-6)^2 + (-2)^2 + 3^2}} \right) = -\frac{3}{7}F_{AB}\mathbf{i} - \frac{1}{7}F_{AB}\mathbf{j} + \frac{3}{14}F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AD} = \frac{F_{AB}}{2} \left( \frac{-6\mathbf{i} + 2\mathbf{j} + 3\mathbf{k}}{\sqrt{(-6)^2 + 2^2 + 3^2}} \right) = -\frac{3}{7}F_{AB}\mathbf{i} + \frac{1}{7}F_{AB}\mathbf{j} + \frac{3}{14}F_{AB}\mathbf{k}$$

$$\mathbf{F} = (-490.5\mathbf{k}) \text{ N}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = 0; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = 0$$

$$\left( (F_{AB})_x - \frac{3}{7}F_{AB} - \frac{3}{7}F_{AB} \right) \mathbf{i} + \left( -\frac{1}{7}F_{AB} + \frac{1}{7}F_{AB} \right) \mathbf{j} + \left( (F_{AB})_z + \frac{3}{14}F_{AB} + \frac{3}{14}F_{AB} - 490.5 \right) \mathbf{k} = 0$$

Equating i, j and k components, we have

$$(F_{AB})_x - \frac{3}{7}F_{AB} - \frac{3}{7}F_{AB} = 0 \quad (F_{AB})_x = \frac{6}{7}F_{AB} \quad [1]$$

$$-\frac{1}{7}F_{AB} + \frac{1}{7}F_{AB} = 0 \quad (\text{Satisfied!})$$

$$(F_{AB})_z + \frac{3}{14}F_{AB} + \frac{3}{14}F_{AB} - 490.5 = 0 \quad (F_{AB})_z = 490.5 - \frac{3}{7}F_{AB} \quad [2]$$

However,  $F_{AB}^2 = (F_{AB})_x^2 + (F_{AB})_z^2$ , then substitute Eqs. [1] and [2] into this expression yields

$$F_{AB}^2 = \left( \frac{6}{7}F_{AB} \right)^2 + \left( 490.5 - \frac{3}{7}F_{AB} \right)^2$$

Solving for positive root, we have

$$F_{AB} = 519.79 \text{ N} = 520 \text{ N} \quad \text{Ans}$$

$$\text{Thus,} \quad F_{AC} = F_{AD} = \frac{1}{2}(519.79) = 260 \text{ N} \quad \text{Ans}$$

Also,

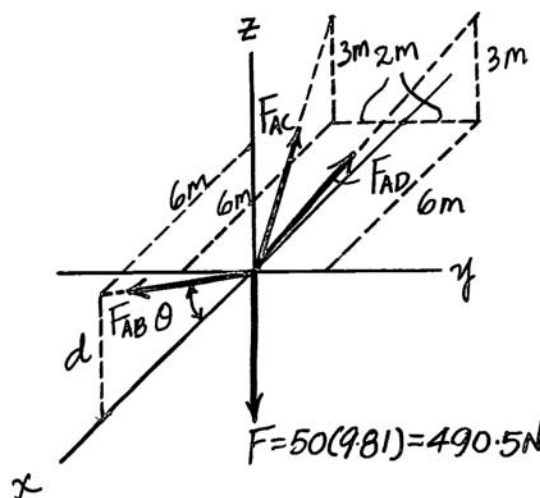
$$(F_{AB})_x = \frac{6}{7}(519.79) = 445.53 \text{ N}$$

$$(F_{AB})_z = 490.5 - \frac{3}{7}(519.79) = 267.73 \text{ N}$$

then,

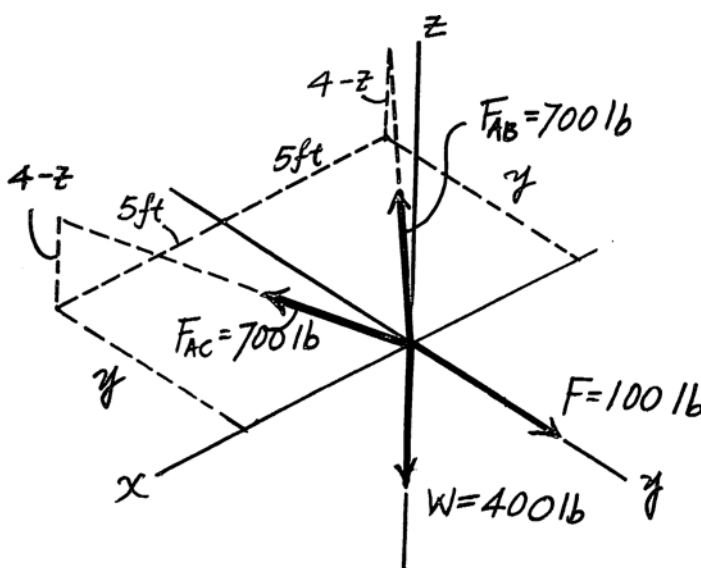
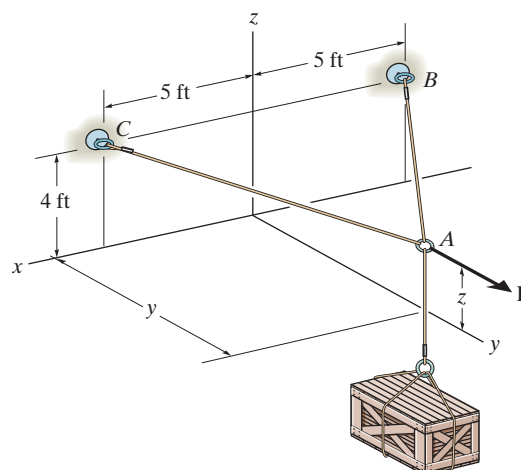
$$\theta = \tan^{-1} \left( \frac{(F_{AB})_z}{(F_{AB})_x} \right) = \tan^{-1} \left( \frac{267.73}{445.53} \right) = 31.00^\circ$$

$$d = 6 \tan \theta = 6 \tan 31.00^\circ = 3.61 \text{ m} \quad \text{Ans}$$



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3-62. A force of  $F = 100$  lb holds the 400-lb crate in equilibrium. Determine the coordinates  $(0, y, z)$  of point A if the tension in cords AC and AB is 700 lb each.



$$\mathbf{F}_{AC} = 700 \left( \frac{5\mathbf{i} - y\mathbf{j} + (4-z)\mathbf{k}}{\sqrt{5^2 + (-y)^2 + (4-z)^2}} \right)$$

$$\mathbf{F}_{AB} = 700 \left( \frac{-5\mathbf{i} - y\mathbf{j} + (4-z)\mathbf{k}}{\sqrt{(-5)^2 + (-y)^2 + (4-z)^2}} \right)$$

$$\mathbf{F} = \{100\mathbf{j}\} \text{ lb} \quad \mathbf{W} = \{-400\mathbf{k}\} \text{ lb}$$

$$\Sigma F_x = 0; \quad \frac{3500}{\sqrt{25 + y^2 + (4-z)^2}} + \frac{-3500}{\sqrt{25 + y^2 + (4-z)^2}} = 0$$

$$\Sigma F_y = 0; \quad \frac{-700y}{\sqrt{25 + y^2 + (4-z)^2}} + \frac{-700y}{\sqrt{25 + y^2 + (4-z)^2}} + 100 = 0 \quad (1)$$

$$\Sigma F_z = 0; \quad \frac{700(4-z)}{\sqrt{25 + y^2 + (4-z)^2}} + \frac{700(4-z)}{\sqrt{25 + y^2 + (4-z)^2}} - 400 = 0 \quad (2)$$

$$1400y = 100\sqrt{25 + y^2 + (4-z)^2}$$

$$1400(4-z) = 400\sqrt{25 + y^2 + (4-z)^2}$$

$$\frac{y}{4-z} = \frac{1}{4} \quad 4y = 4-z$$

Thus,

$$1400y = 100\sqrt{25 + y^2 + 16y^2}$$

$$196y^2 = 25 + 17y^2$$

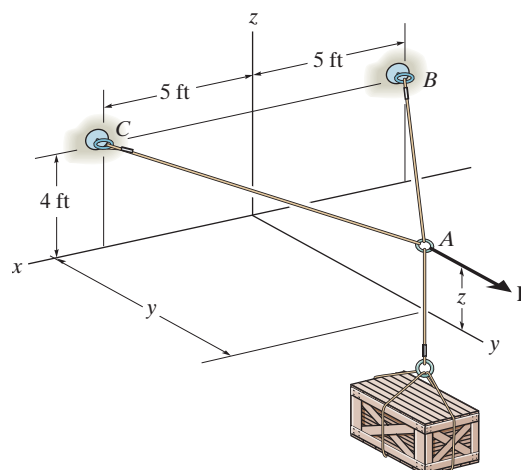
$$y = 0.3737 \text{ ft} = 0.374 \text{ ft} \quad \text{Ans}$$

$$4(0.3737) = 4-z; \quad z = 2.51 \text{ ft} \quad \text{Ans}$$



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**3–63.** If the maximum allowable tension in cables  $AB$  and  $AC$  is 500 lb, determine the maximum height  $z$  to which the 200-lb crate can be lifted. What horizontal force  $F$  must be applied? Take  $y = 8$  ft.



$$\Sigma F_y = 0; \quad -2 \left[ 500 \left( \frac{8}{\sqrt{5^2 + 8^2 + (4-z)^2}} \right) \right] + F = 0 \quad (1)$$

$$\Sigma F_z = 0; \quad 2 \left[ 500 \left( \frac{4-z}{\sqrt{5^2 + 8^2 + (4-z)^2}} \right) \right] - 200 = 0 \quad (2)$$

Dividing Eq. (2) by Eq. (1),

$$\frac{4-z}{8} = \frac{200}{F}$$

$$(4-z) = \frac{1600}{F}$$

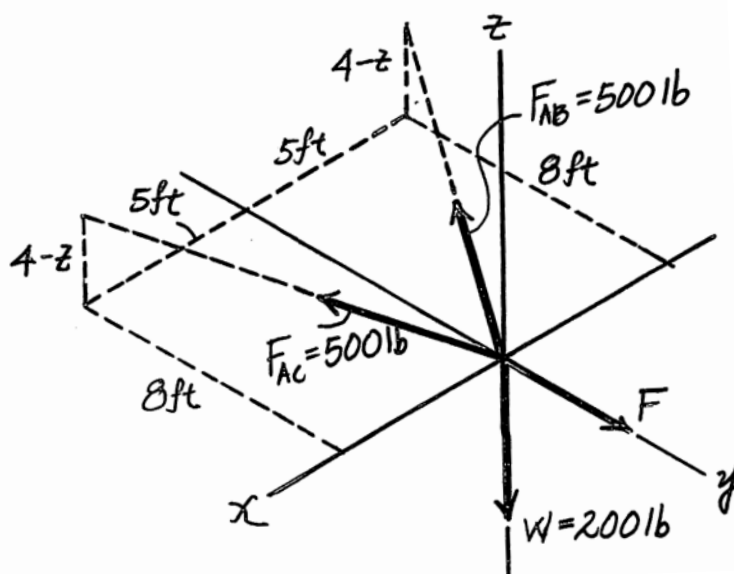
From Eq. (1):

$$\frac{8000}{F} = \sqrt{89 + \left( \frac{1600}{F} \right)^2}$$

$$\left( \frac{8000}{F} \right)^2 = 89 + \left( \frac{1600}{F} \right)^2$$

$$F = 831 \text{ lb} \quad \text{Ans}$$

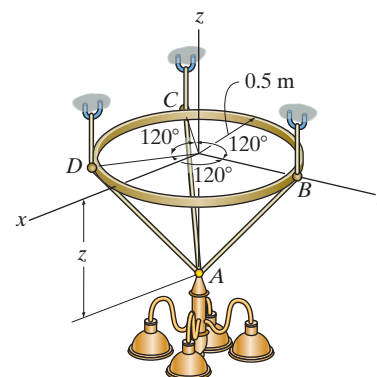
$$z = 2.07 \text{ ft} \quad \text{Ans}$$





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\*3-64. The thin ring can be adjusted vertically between three equally long cables from which the 100-kg chandelier is suspended. If the ring remains in the horizontal plane and  $z = 600$  mm, determine the tension in each cable.



**Geometry:** Referring to the geometry of the free-body diagram shown in Fig. (a), the lengths of cables  $AB$ ,  $AC$ , and  $AD$  are

$$\text{all } l = \sqrt{0.5^2 + 0.6^2} = \sqrt{0.61} \text{ m}$$

**Equations of Equilibrium:** Equilibrium requires

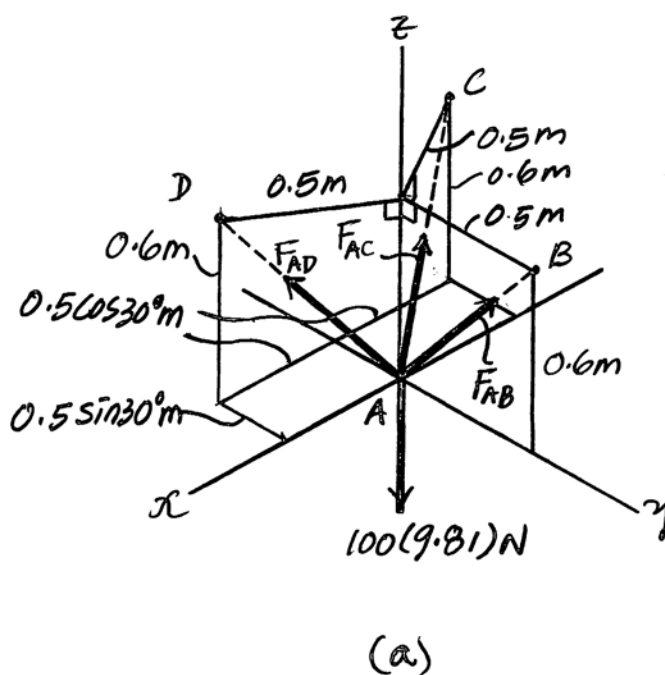
$$\Sigma F_x = 0; \quad F_{AD} \left( \frac{0.5 \cos 30^\circ}{\sqrt{0.61}} \right) - F_{AC} \left( \frac{0.5 \cos 30^\circ}{\sqrt{0.61}} \right) = 0 \quad F_{AD} = F_{AC} = F$$

$$\Sigma F_y = 0; \quad F_{AB} \left( \frac{0.5}{\sqrt{0.61}} \right) - 2 \left[ F \left( \frac{0.5 \sin 30^\circ}{\sqrt{0.61}} \right) \right] = 0 \quad F_{AB} = F$$

Thus, cables  $AB$ ,  $AC$ , and  $AD$  all develop the same tension.

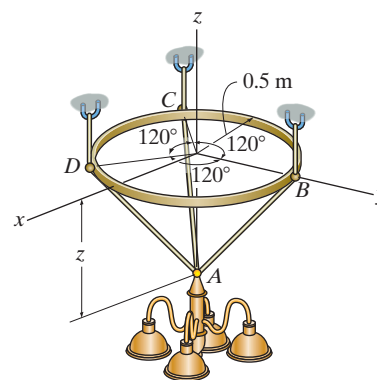
$$\Sigma F_z = 0; \quad 3F \left( \frac{0.6}{\sqrt{0.61}} \right) - 100(9.81) = 0$$

$$F_{AB} = F_{AC} = F_{AD} = 426 \text{ N} \quad \text{Ans.}$$



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•3–65. The thin ring can be adjusted vertically between three equally long cables from which the 100-kg chandelier is suspended. If the ring remains in the horizontal plane and the tension in each cable is not allowed to exceed 1 kN, determine the smallest allowable distance  $z$  required for equilibrium.



**Geometry:** Referring to the geometry of the free-body diagram shown in Fig. (a), the lengths of cables  $AB$ ,  $AC$ , and  $AD$  are

$$\text{all } l = \sqrt{0.5^2 + z^2}.$$

**Equations of Equilibrium:** Equilibrium requires

$$\Sigma F_x = 0; \quad F_{AD} \left( \frac{0.5 \cos 30^\circ}{\sqrt{0.5^2 + z^2}} \right) - F_{AC} \left( \frac{0.5 \cos 30^\circ}{\sqrt{0.5^2 + z^2}} \right) = 0 \quad F_{AD} = F_{AC} = F$$

$$\Sigma F_y = 0; \quad F_{AB} \left( \frac{0.5}{\sqrt{0.5^2 + z^2}} \right) - 2 \left[ F \left( \frac{0.5 \sin 30^\circ}{\sqrt{0.5^2 + z^2}} \right) \right] = 0 \quad F_{AB} = F$$

Thus, cables  $AB$ ,  $AC$ , and  $AD$  all develop the same tension.

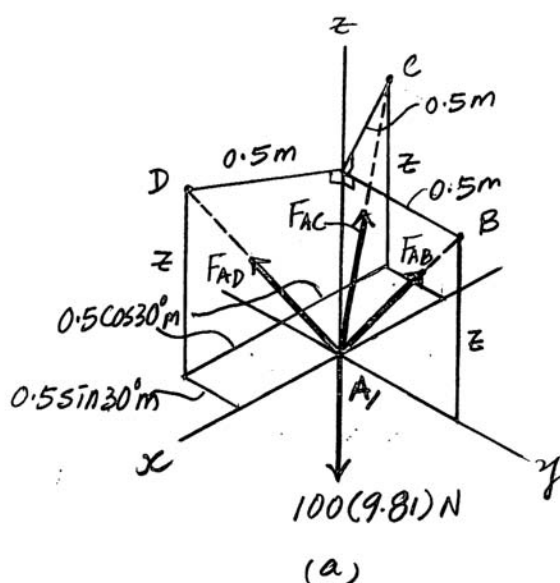
$$\Sigma F_z = 0; \quad 3F \left( \frac{z}{\sqrt{0.5^2 + z^2}} \right) - 100(9.81) = 0$$

Cables  $AB$ ,  $AC$ , and  $AD$  will also achieve maximum tension simultaneously. Substituting  $F = 1000$  N, we obtain

$$3(1000) \left( \frac{z}{\sqrt{0.5^2 + z^2}} \right) - 100(9.81) = 0$$

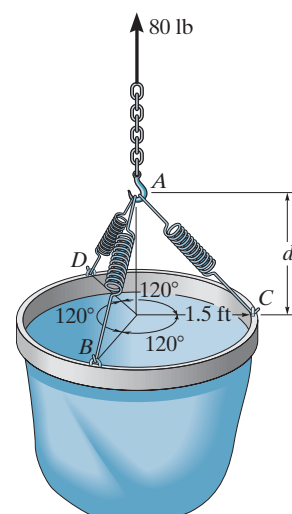
$$z = 0.1730 \text{ m} = 173 \text{ mm}$$

**Ans.**



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**3–66.** The bucket has a weight of 80 lb and is being hoisted using three springs, each having an unstretched length of  $l_0 = 1.5$  ft and stiffness of  $k = 50$  lb/ft. Determine the vertical distance  $d$  from the rim to point A for equilibrium.



$$\Sigma F_z = 0;$$

$$80 - \left( \frac{3d}{\sqrt{d^2 + (1.5)^2}} \right) F = 0$$

$$80 - \frac{3d}{\sqrt{d^2 + (1.5)^2}} [50 (\sqrt{d^2 + (1.5)^2} - 1.5)] = 0$$

$$\frac{d}{\sqrt{d^2 + (1.5)^2}} (\sqrt{d^2 + (1.5)^2} - 1.5) = 0.5333$$

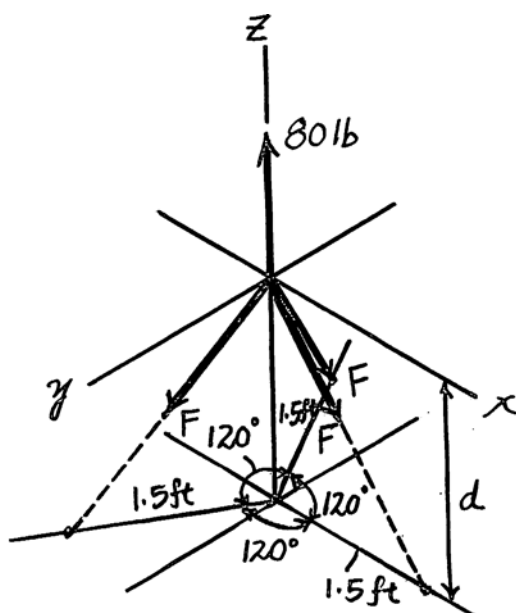
$$d \sqrt{d^2 + (1.5)^2} - 1.5d = 0.5333 \sqrt{d^2 + (1.5)^2}$$

$$\sqrt{d^2 + (1.5)^2} (d - 0.5333) = 1.5d$$

$$[d^2 + (1.5)^2] [d^2 - 2d(0.5333) + (0.5333)^2] = (1.5)^2 d^2$$

$$d^4 - 1.067d^3 + 0.284d^2 - 2.4d + 0.64 = 0$$

$$d = 1.64 \text{ ft} \quad \text{Ans}$$



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3-67. Three cables are used to support a 900-lb ring. Determine the tension in each cable for equilibrium.

**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{3\mathbf{j} - 4\mathbf{k}}{\sqrt{3^2 + (-4)^2}} \right) = 0.6F_{AB}\mathbf{j} - 0.8F_{AB}\mathbf{k}$$

$$\begin{aligned} \mathbf{F}_{AC} &= F_{AC} \left( \frac{3\cos 30^\circ\mathbf{i} - 3\sin 30^\circ\mathbf{j} - 4\mathbf{k}}{\sqrt{(3\cos 30^\circ)^2 + (-3\sin 30^\circ)^2 + (-4)^2}} \right) \\ &= 0.5196F_{AC}\mathbf{i} - 0.3F_{AC}\mathbf{j} - 0.8F_{AC}\mathbf{k} \end{aligned}$$

$$\begin{aligned} \mathbf{F}_{AD} &= F_{AD} \left( \frac{-3\cos 30^\circ\mathbf{i} - 3\sin 30^\circ\mathbf{j} - 4\mathbf{k}}{\sqrt{(-3\cos 30^\circ)^2 + (-3\sin 30^\circ)^2 + (-4)^2}} \right) \\ &= -0.5196F_{AD}\mathbf{i} - 0.3F_{AD}\mathbf{j} - 0.8F_{AD}\mathbf{k} \end{aligned}$$

$$\mathbf{F} = \{900\mathbf{k}\} \text{ lb}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AD} + \mathbf{F} = \mathbf{0}$$

$$(0.5196F_{AC} - 0.5196F_{AD})\mathbf{i} + (0.6F_{AB} - 0.3F_{AC} - 0.3F_{AD})\mathbf{j} + (-0.8F_{AB} - 0.8F_{AC} - 0.8F_{AD} + 900)\mathbf{k} = \mathbf{0}$$

Equating i, j and k components, we have

$$0.5196F_{AC} - 0.5196F_{AD} = 0 \quad [1]$$

$$0.6F_{AB} - 0.3F_{AC} - 0.3F_{AD} = 0 \quad [2]$$

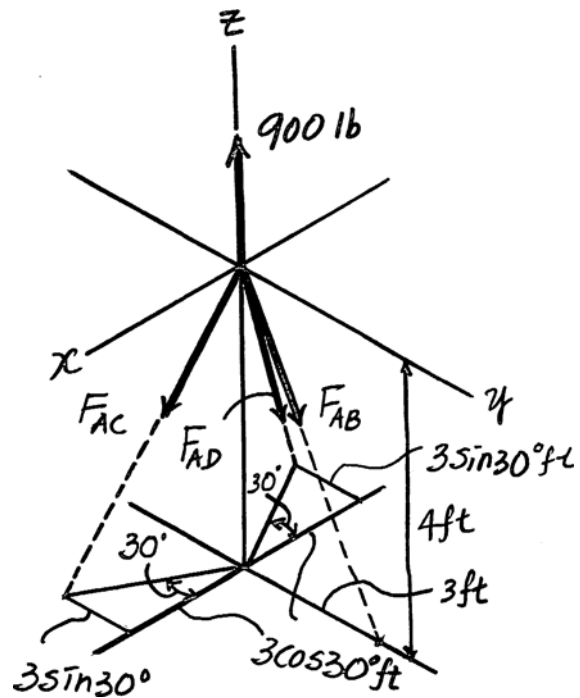
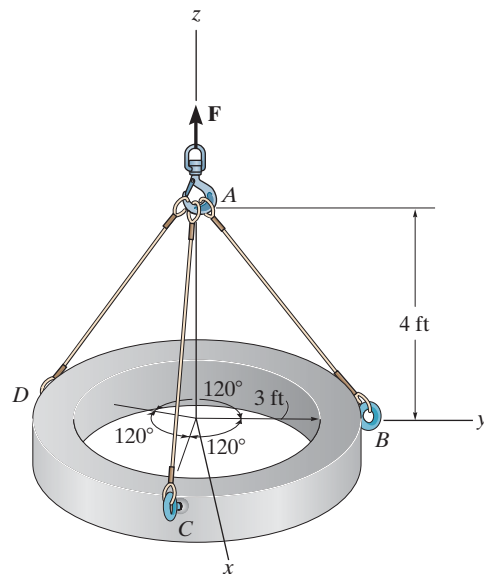
$$-0.8F_{AB} - 0.8F_{AC} - 0.8F_{AD} + 900 = 0 \quad [3]$$

Solving Eqs. [1], [2] and [3] yields

$$F_{AB} = F_{AC} = F_{AD} = 375 \text{ lb} \quad \text{Ans}$$

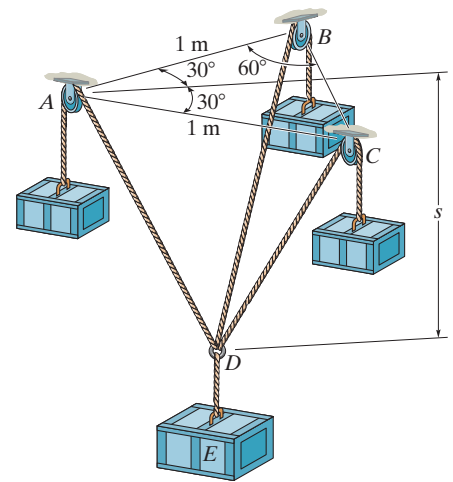
This problem also can be easily solved if one realizes that due to symmetry all cables are subjected to a same tensile force, that is  $F_{AB} = F_{AC} = F_{AD} = F$ . Summing forces along z axis yields

$$\Sigma F_z = 0; \quad 900 - 3F\left(\frac{4}{5}\right) = 0 \quad F = 375 \text{ lb}$$



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**\*3–68.** The three outer blocks each have a mass of 2 kg, and the central block *E* has a mass of 3 kg. Determine the sag *s* for equilibrium of the system.



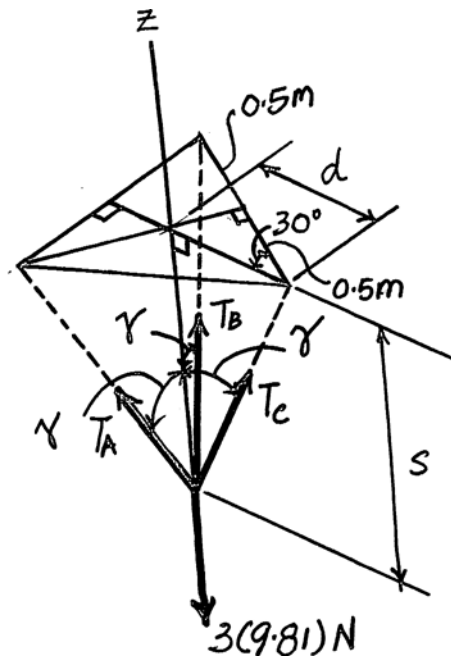
$$T_A = T_B = T_C = 2(9.81)$$

$$\Sigma F_z = 0; \quad 3(2(9.81)) \cos \gamma - 3(9.81) = 0$$

$$\cos \gamma = 0.5; \quad \gamma = 60^\circ$$

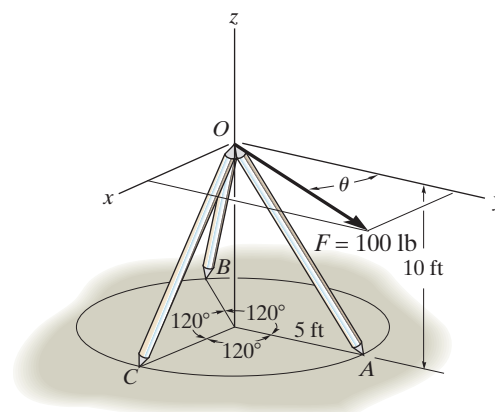
$$d = \frac{0.5}{\cos 30^\circ} = 0.577 \text{ m}$$

$$s = \frac{0.577}{\tan 60^\circ} = 0.333 \text{ m} = 333 \text{ mm} \quad \text{Ans}$$



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•3–69. Determine the angle  $\theta$  such that an equal force is developed in legs  $OB$  and  $OC$ . What is the force in each leg if the force is directed along the axis of each leg? The force  $\mathbf{F}$  lies in the  $x$ – $y$  plane. The supports at  $A$ ,  $B$ ,  $C$  can exert forces in either direction along the attached legs.



$$\begin{aligned} F_{OA} &= F_{OA} \left( -\frac{5}{11.180} \mathbf{j} + \frac{10}{11.180} \mathbf{k} \right) \\ &= F_{OA} (-0.4472 \mathbf{j} + 0.89443 \mathbf{k}) \end{aligned}$$

$$\begin{aligned} F_{OB} &= F_{OB} \left( -\frac{5 \sin 60^\circ}{11.18} \mathbf{i} - \frac{5 \cos 60^\circ}{11.18} \mathbf{j} - \frac{10}{11.18} \mathbf{k} \right) \\ &= F_{OB} (-0.3873 \mathbf{i} - 0.2236 \mathbf{j} - 0.8944 \mathbf{k}) \end{aligned}$$

$$\begin{aligned} F_{OC} &= F_{OC} \left( \frac{5 \sin 60^\circ}{11.18} \mathbf{i} - \frac{5 \cos 60^\circ}{11.18} \mathbf{j} - \frac{10}{11.18} \mathbf{k} \right) \\ &= F_{OC} (0.3873 \mathbf{i} - 0.2236 \mathbf{j} - 0.8944 \mathbf{k}) \end{aligned}$$

$$\mathbf{F} = 100 (\sin \theta \mathbf{i} + \cos \theta \mathbf{j})$$

$$\Sigma F_x = 0; \quad -0.3873 F_{OB} + 0.3873 F_{OC} + 100 \sin \theta = 0$$

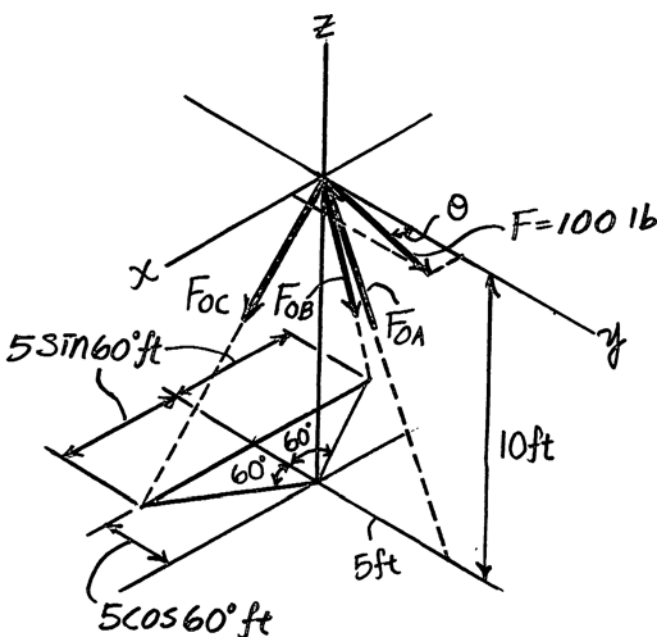
$$\text{If } F_{OC} = F_{OB}, \text{ then } 100 \sin \theta = 0; \quad \theta = 0^\circ \quad \text{Ans}$$

$$\Sigma F_y = 0; \quad -0.4472 F_{OA} - 0.2236 F_{OB} - 0.2236 F_{OC} + 100 = 0$$

$$\Sigma F_z = 0; \quad 0.8944 F_{OA} - 0.8944 F_{OB} - 0.8944 F_{OC} = 0$$

$$F_{OA} = 149 \text{ lb} \quad \text{Ans}$$

$$F_{OB} = F_{OC} = 74.5 \text{ lb} \quad \text{Ans}$$



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**3–70.** The 500-lb crate is hoisted using the ropes  $AB$  and  $AC$ . Each rope can withstand a maximum tension of 2500 lb before it breaks. If  $AB$  always remains horizontal, determine the smallest angle  $\theta$  to which the crate can be hoisted.

Case 1: Assume  $T_{AB} = 2500$  lb

$$+\rightarrow \Sigma F_x = 0; \quad 2500 - T_{AC} \cos \theta = 0$$

$$+\uparrow \Sigma F_y = 0; \quad T_{AC} \sin \theta - 500 = 0$$

Solving,

$$\theta = 11.31^\circ$$

$$T_{AC} = 2549.5 \text{ lb} > 2500 \text{ lb} \quad (\text{N.G.})$$

Case 2: Assume  $T_{AC} = 2500$  lb

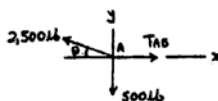
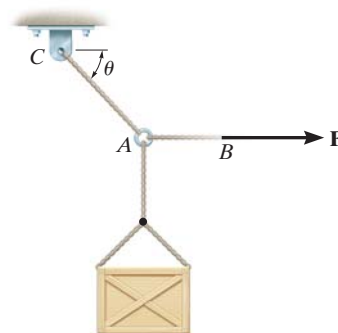
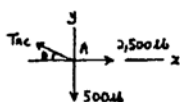
$$+\uparrow \Sigma F_y = 0; \quad 2500 \sin \theta - 500 = 0$$

$$\theta = 11.54^\circ$$

$$+\rightarrow \Sigma F_x = 0; \quad T_{AB} - 2500 \cos 11.54^\circ = 0$$

$$T_{AB} = 2449.49 \text{ lb} < 2500 \text{ lb}$$

Thus, the smallest angle is  $\theta = 11.5^\circ$  **Ans**



**3–71.** The members of a truss are pin connected at joint  $O$ . Determine the magnitude of  $F_1$  and its angle  $\theta$  for equilibrium. Set  $F_2 = 6$  kN.

$$+\rightarrow \Sigma F_x = 0; \quad 6 \sin 70^\circ + F_1 \cos \theta - 5 \cos 30^\circ - \frac{4}{5}(7) = 0$$

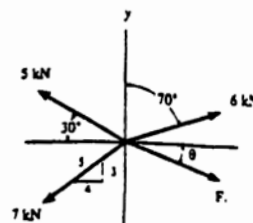
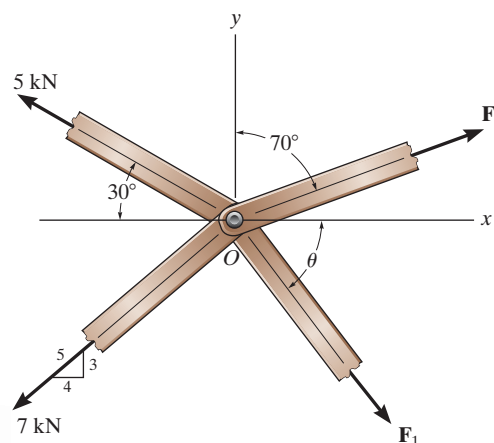
$$F_1 \cos \theta = 4.2920$$

$$+\uparrow \Sigma F_y = 0; \quad 6 \cos 70^\circ + 5 \sin 30^\circ - F_1 \sin \theta - \frac{3}{5}(7) = 0$$

$$F_1 \sin \theta = 0.3521$$

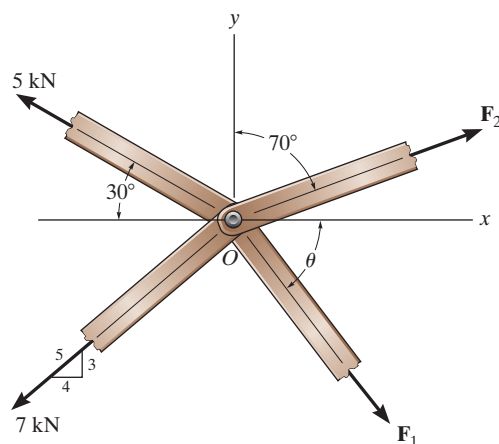
Solving:

$$\theta = 4.69^\circ \quad \text{Ans} \quad F_1 = 4.31 \text{ kN} \quad \text{Ans}$$



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**\*3–72.** The members of a truss are pin connected at joint  $O$ . Determine the magnitudes of  $F_1$  and  $F_2$  for equilibrium. Set  $\theta = 60^\circ$ .



$$\rightarrow \Sigma F_x = 0; \quad F_2 \sin 70^\circ + F_1 \cos 60^\circ - 5 \cos 30^\circ - \frac{4}{5}(7) = 0$$

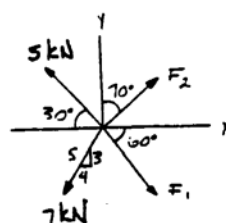
$$0.9397F_2 + 0.5F_1 = 9.930$$

$$+ \uparrow \Sigma F_y = 0; \quad F_2 \cos 70^\circ + 5 \sin 30^\circ - F_1 \sin 60^\circ - \frac{3}{5}(7) = 0$$

$$0.3420F_2 - 0.8660F_1 = 1.7$$

Solving:

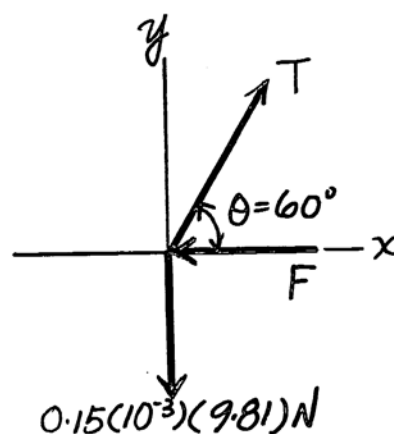
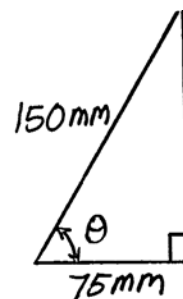
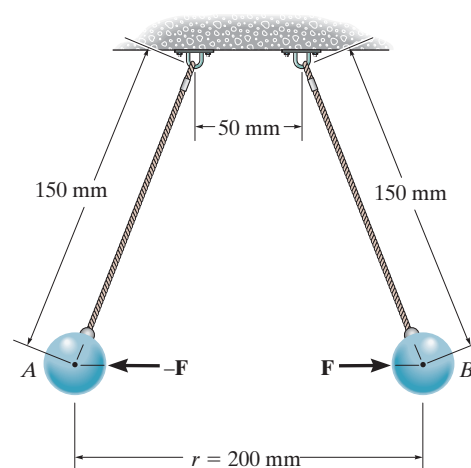
$$F_2 = 9.60 \text{ kN} \quad \text{Ans} \quad F_1 = 1.83 \text{ kN} \quad \text{Ans}$$





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•3–73. Two electrically charged pith balls, each having a mass of 0.15 g, are suspended from light threads of equal length. Determine the magnitude of the horizontal repulsive force,  $F$ , acting on each ball if the measured distance between them is  $r = 200$  mm.



$$\cos \theta = \frac{75}{150} \quad \theta = 60^\circ$$

$$\uparrow \Sigma F_y = 0; \quad T \sin 60^\circ - 0.15(10)^{-3}(9.81) = 0$$

$$T = 1.699(10)^{-3} \text{ N}$$

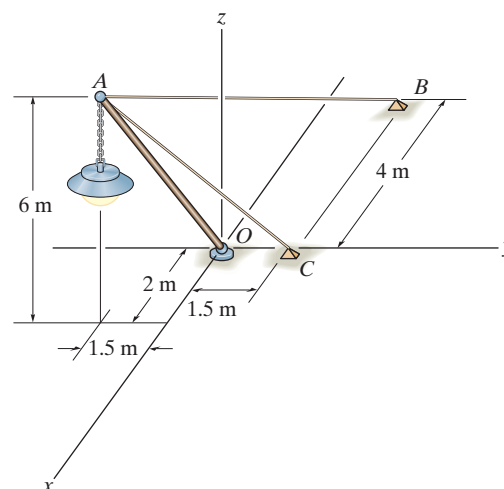
$$\rightarrow \Sigma F_x = 0; \quad 1.699(10)^{-3} \cos 60^\circ - F = 0$$

$$F = 0.850(10)^{-3} \text{ N}$$

$$= 0.850 \text{ mN} \quad \text{Ans}$$

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**3–74.** The lamp has a mass of 15 kg and is supported by a pole  $AO$  and cables  $AB$  and  $AC$ . If the force in the pole acts along its axis, determine the forces in  $AO$ ,  $AB$ , and  $AC$  for equilibrium.



**Cartesian Vector Notation :**

$$\mathbf{F}_{AB} = F_{AB} \left( \frac{-6\mathbf{i} + 3\mathbf{j} - 6\mathbf{k}}{\sqrt{(-6)^2 + 3^2 + (-6)^2}} \right) = -\frac{2}{3}F_{AB}\mathbf{i} + \frac{1}{3}F_{AB}\mathbf{j} - \frac{2}{3}F_{AB}\mathbf{k}$$

$$\mathbf{F}_{AC} = F_{AC} \left( \frac{-2\mathbf{i} + 3\mathbf{j} - 6\mathbf{k}}{\sqrt{(-2)^2 + 3^2 + (-6)^2}} \right) = -\frac{2}{7}F_{AC}\mathbf{i} + \frac{3}{7}F_{AC}\mathbf{j} - \frac{6}{7}F_{AC}\mathbf{k}$$

$$\mathbf{F}_{AO} = F_{AO} \left( \frac{2\mathbf{i} - 1.5\mathbf{j} + 6\mathbf{k}}{\sqrt{2^2 + (-1.5)^2 + 6^2}} \right) = \frac{4}{13}F_{AO}\mathbf{i} - \frac{3}{13}F_{AO}\mathbf{j} + \frac{12}{13}F_{AO}\mathbf{k}$$

$$\mathbf{F} = \{-147.15\mathbf{k}\} \text{ N}$$

**Equations of Equilibrium :**

$$\Sigma \mathbf{F} = \mathbf{0}; \quad \mathbf{F}_{AB} + \mathbf{F}_{AC} + \mathbf{F}_{AO} + \mathbf{F} = \mathbf{0}$$

$$\left( -\frac{2}{3}F_{AB} - \frac{2}{7}F_{AC} + \frac{4}{13}F_{AO} \right)\mathbf{i} + \left( \frac{1}{3}F_{AB} + \frac{3}{7}F_{AC} - \frac{3}{13}F_{AO} \right)\mathbf{j} + \left( -\frac{2}{3}F_{AB} - \frac{6}{7}F_{AC} + \frac{12}{13}F_{AO} - 147.15 \right)\mathbf{k} = \mathbf{0}$$

Equating i, j and k components, we have

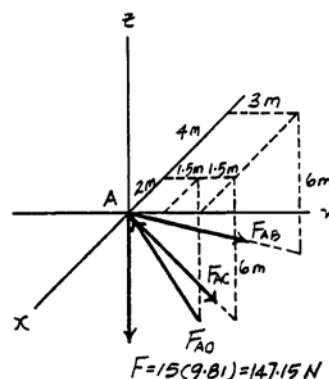
$$-\frac{2}{3}F_{AB} - \frac{2}{7}F_{AC} + \frac{4}{13}F_{AO} = 0 \quad [1]$$

$$\frac{1}{3}F_{AB} + \frac{3}{7}F_{AC} - \frac{3}{13}F_{AO} = 0 \quad [2]$$

$$-\frac{2}{3}F_{AB} - \frac{6}{7}F_{AC} + \frac{12}{13}F_{AO} - 147.15 = 0 \quad [3]$$

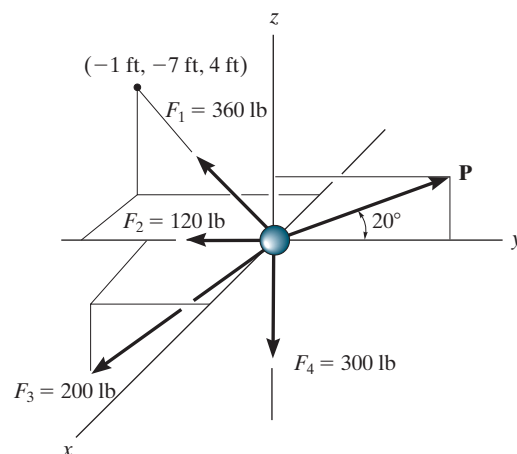
Solving Eqs. [1], [2] and [3] yields

$$F_{AB} = 110 \text{ N} \quad F_{AC} = 85.8 \text{ N} \quad F_{AO} = 319 \text{ N} \quad \text{Ans}$$



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**3-75.** Determine the magnitude of  $\mathbf{P}$  and the coordinate direction angles of  $\mathbf{F}_3$  required for equilibrium of the particle. Note that  $\mathbf{F}_3$  acts in the octant shown.



$$\mathbf{F}_1 = 360 \left( -\frac{1}{\sqrt{66}} \mathbf{i} - \frac{7}{\sqrt{66}} \mathbf{j} + \frac{4}{\sqrt{66}} \mathbf{k} \right)$$

$$= -44.313 \mathbf{i} - 310.191 \mathbf{j} + 177.252 \mathbf{k}$$

$$\mathbf{F}_2 = -120 \mathbf{j}$$

$$\mathbf{F}_4 = -300 \mathbf{k}$$

$$\mathbf{F}_3 = F_x \mathbf{i} + F_y \mathbf{j} + F_z \mathbf{k} \quad (1)$$

$$\mathbf{P} = P \cos 20^\circ \mathbf{j} + P \sin 20^\circ \mathbf{k}$$

$$\Sigma F_x = 0; \quad -44.313 + F_x = 0$$

$$F_x = 44.313 \text{ lb}$$

$$\Sigma F_y = 0; \quad -310.191 - 120 + F_y + 0.9397 P = 0$$

$$\Sigma F_z = 0; \quad 177.252 - 300 + F_z + 0.3420 P = 0$$

From Eq. (1), require

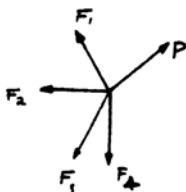
$$200 = \sqrt{F_x^2 + F_y^2 + F_z^2}$$

$$(200)^2 = (44.313)^2 + (430.191 - 0.9397 P)^2 + (122.748 - 0.3420 P)^2$$

$$P^2 - 892.459 P + 162.095 = 0$$

Solving,

$$P = 638.65 \text{ lb} \quad \text{and} \quad P = 253.81 \text{ lb}$$



Thus, with  $P = 638.65 \text{ lb}$ ,  $F_y = -169.95 \text{ lb}$ .

With  $P = 253.81 \text{ lb}$ ,  $F_y = 191.69 \text{ lb}$ .

In order for  $\mathbf{F}_3$  to be within the octant shown, choose

$$P = 639 \text{ lb} \quad \text{Ans}$$

so that

$$F_z = -95.672$$

Thus, the direction of  $\mathbf{F}_3$  is :

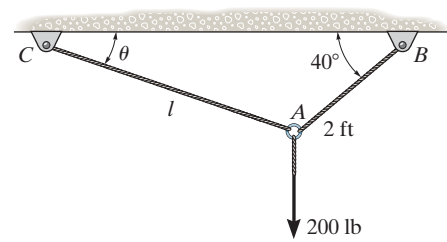
$$\alpha_3 = \cos^{-1} \left( \frac{44.313}{200} \right) = 77.2^\circ \quad \text{Ans}$$

$$\beta_3 = \cos^{-1} \left( \frac{-169.95}{200} \right) = 148^\circ \quad \text{Ans}$$

$$\gamma_3 = \cos^{-1} \left( \frac{-95.672}{200} \right) = 119^\circ \quad \text{Ans}$$

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**\*3–76.** The ring of negligible size is subjected to a vertical force of 200 lb. Determine the longest length  $l$  of cord  $AC$  such that the tension acting in  $AC$  is 160 lb. Also, what is the force acting in cord  $AB$ ? *Hint:* Use the equilibrium condition to determine the required angle  $\theta$  for attachment, then determine  $l$  using trigonometry applied to  $\triangle ABC$ .



**Equations of Equilibrium :**

$$\rightarrow \Sigma F_x = 0; \quad F_{AB} \cos 40^\circ - 160 \cos \theta = 0 \quad [1]$$

$$+ \uparrow \Sigma F_y = 0; \quad F_{AB} \sin 40^\circ + 160 \sin \theta - 200 = 0 \quad [2]$$

Solving Eqs. [1] and [2] yields

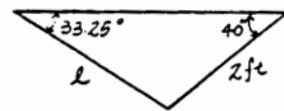
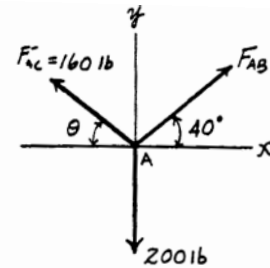
$$\theta = 33.25^\circ$$

$$F_{AB} = 175 \text{ lb} \quad \text{Ans}$$

**Geometry :** Applying law of sines, we have

$$\frac{l}{\sin 40^\circ} = \frac{2}{\sin 33.25^\circ}$$

$$l = 2.34 \text{ ft} \quad \text{Ans}$$



**•3–77.** Determine the magnitudes of  $F_1$ ,  $F_2$ , and  $F_3$  for equilibrium of the particle.

$$\Sigma F_x = 0; \quad F_2 + F_1 \cos 60^\circ - 800 \left( \frac{3}{5} \right) = 0$$

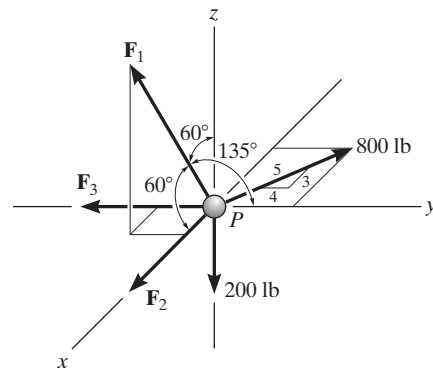
$$\Sigma F_y = 0; \quad 800 \left( \frac{4}{5} \right) + F_1 \cos 135^\circ - F_3 = 0$$

$$\Sigma F_z = 0; \quad F_1 \cos 60^\circ - 200 = 0$$

$$F_1 = 400 \text{ lb} \quad \text{Ans}$$

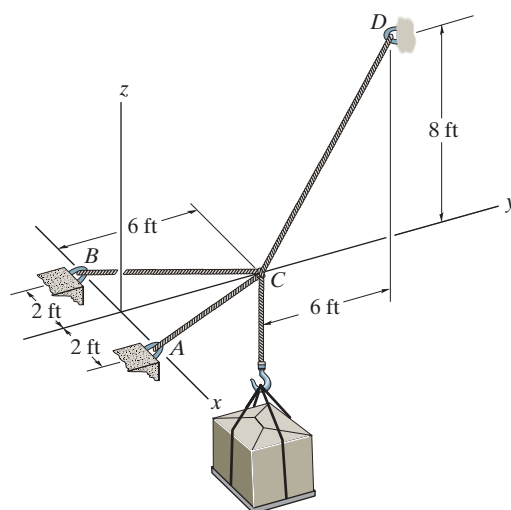
$$F_2 = 280 \text{ lb} \quad \text{Ans}$$

$$F_3 = 357 \text{ lb} \quad \text{Ans}$$



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**3–78.** Determine the force in each cable needed to support the 500-lb load.



**Equation of Equilibrium :**

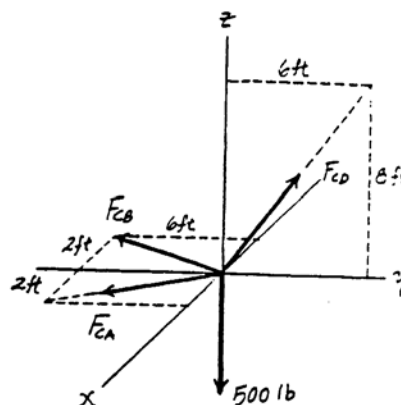
$$\Sigma F_z = 0; \quad F_{CD} \left( \frac{4}{5} \right) - 500 = 0 \quad F_{CD} = 625 \text{ lb} \quad \text{Ans}$$

Using the results  $F_{CD} = 625 \text{ lb}$  and then summing forces along  $x$  and  $y$  axes we have

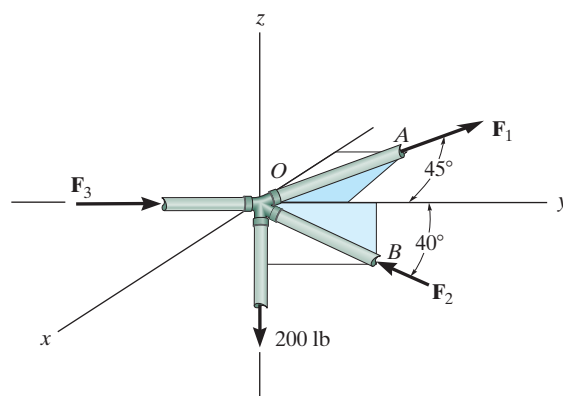
$$\Sigma F_x = 0; \quad F_{CA} \left( \frac{2}{\sqrt{40}} \right) - F_{CB} \left( \frac{2}{\sqrt{40}} \right) = 0 \quad F_{CA} = F_{CB} = F$$

$$\Sigma F_y = 0; \quad 2F \left( \frac{6}{\sqrt{40}} \right) - 625 \left( \frac{3}{5} \right) = 0$$

$$F_{CA} = F_{CB} = F = 198 \text{ lb} \quad \text{Ans}$$



**3–79.** The joint of a space frame is subjected to four member forces. Member  $OA$  lies in the  $x$ – $y$  plane and member  $OB$  lies in the  $y$ – $z$  plane. Determine the forces acting in each of the members required for equilibrium of the joint.



**Equation of Equilibrium :**

$$\Sigma F_x = 0; \quad F_1 \sin 45^\circ = 0 \quad F_1 = 0 \quad \text{Ans}$$

$$\Sigma F_z = 0; \quad F_2 \sin 40^\circ - 200 = 0 \quad F_2 = 311.14 \text{ lb} = 311 \text{ lb} \quad \text{Ans}$$

Using the results  $F_1 = 0$  and  $F_2 = 311.14 \text{ lb}$  and then summing forces along the  $y$  axis, we have

$$\Sigma F_y = 0; \quad F_3 - 311.14 \cos 40^\circ = 0 \quad F_3 = 238 \text{ lb} \quad \text{Ans}$$

